

# **Equipment Design and Cost Estimation for Small Modular Biomass Systems, Synthesis** Gas Cleanup, and Oxygen **Separation Equipment**

**Task 2: Gas Cleanup Design and Cost** Estimates – Wood Feedstock

Nexant Inc. San Francisco. California Subcontract Report NREL/SR-510-39945 May 2006

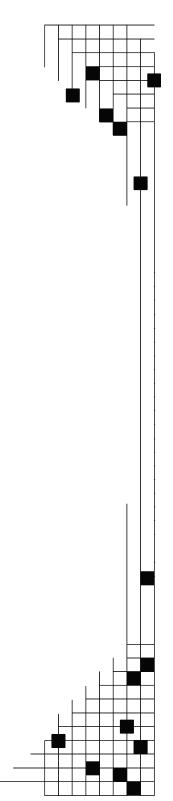


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Nexant Inc. San Francisco, California

NREL Technical Monitor: Kelly Ibsen Prepared under Subcontract No. ACO-5-44027 Subcontract Report NREL/SR-510-39945 May 2006



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## **Executive Summary**

As part of Task 2, Gas Cleanup and Cost Estimates, the team investigated the appropriate process scheme for treatment of wood derived syngas for use in the synthesis of liquid fuels. Two different 2,000 metric tonne per day gasification schemes, a low-pressure, indirect system using the BCL gasifier, and a high-pressure, direct system using GTI gasification technology, were evaluated. Initial syngas conditions from each of the gasifiers was provided to the team by NREL. Nexant was the prime contractor and principal investigator during this task; technical assistance was provided by both GTI and Emery Energy.

The first task explored the different process options available for the removal of the main process impurities, including particulates, sulfur, carbon dioxide, tar, ammonia, and metals. From this list, selection of commercial technologies appropriate for syngas clean-up was made based on the criteria of cost and the ability to meet the final specifications. Preliminary flow schemes were established and presented to NREL; after discussion and modification, final designs, including unit sizes, energy use, capital and operating costs, and labor requirements, were developed. Finally, Nexant performed an analysis to determine how changes in syngas flowrates and compositions would impact the designs, for future reference as the plant size changes.

The technologies chosen for both cases did not differ considerably. Each case possesses the following pieces of equipment:

- Cyclones for particulate removal
- Tar cracking for the removal of heavy and light hydrocarbons. Steam is injected in varying amounts into the tar cracker to set the appropriate hydrogen to carbon monoxide ratio.
- Syngas cooling, necessary for downstream sulfur treatment, and a water quench/venturi scrubber for ammonia and trace contaminant removal
- Amine treatment for sulfur and carbon dioxide removal
- Zinc oxide beds for additional sulfur removal down to the low levels required for fuels synthesis
- Liquid phase oxidation of acid gas for sulfur recovery

The low-pressure gasifier case required the use of a process gas compressor to raise the gas pressure to the level appropriate for downstream treatment and product synthesis. Information was also provided for the level of clean syngas compression necessary to prepare both cases for methanol synthesis.

The results of the analysis for both cases can be seen in Table A below, with information on the capital and operating costs:

	Low-Pressure BCL Gasifier	High-Pressure GTI Gasifier
Wood Feedrate (MTPD)	2,000	2,000
Syngas Rate (lb/hr)	316,369	418,416
Total Installed Cost (\$MM)	109.4	76.5
Power Required (MW)	18.5	(5.2)
Net Steam Required (lb/hr)	44,000	114,000
Water Required (GPM)	37,806	25,454
Natural Gas (MMSCFD)	7	8
Catalysts and Chemicals (\$/day)	1,931	1,457

## TABLE A SYNGAS CLEAN-UP CASE SUMMARY

The bulk of the cost difference between the two cases is due to the process gas compressor required in the low-pressure case. The two cases use similar equipment for all other steps of the process; although the cases had different gas flowrates and compositions, the equipment impact is small relative to that of the process gas compressor. While these results imply that direct gasification is preferred, this study did not take into account other differences in the two process schemes, such as the potential need for an oxygen plant in the high-pressure to chemicals case.

The team also compared the clean-up system design and costs versus the design developed by NREL for a recent biomass to hydrogen study. The cost for the clean-up section of the biomass to chemicals designs is more expensive due to three main reasons: more equipment necessary in the chemical production designs, the increase in steel prices from 2002 to 2005, and different engineering assumptions made in the chemicals production case. The main engineering difference is the cost assumed for the process gas compressor in the low pressure case; a larger compressor and selection of a different design type increases the installed cost by \$25MM versus the NREL design. In addition, gas clean-up cost assumptions made by NREL from previous studies likely underestimated the cost of the tar cracker and heat exchange equipment.

This study updates previous NREL investigations by providing the most up-to-date information for appropriate technologies and their respective costs. Future studies should focus on the following areas to further define suitable technologies and confirm costs:

- *Alternatives for Tar Removal:* Further study and analysis should be performed to validate the methods used by the team. In addition, alternative tar removal technology should be considered, including cracking within the gasifier.
- Process Integration, Gasification Systems and Biorefinery: Integration of the cleanup section with the other parts of the gasification plant will provide a better picture of the overall plant costs.
- Alternate CO<sub>2</sub>/Sulfur Removal Steps: A cost comparison of amine versus physical solvents would provide additional data to confirm the appropriate use of amine in this design Advanced technologies for acid gas removal, such as warm gas clean-up, should also be considered.

• *Other Impurities in the Syngas:* If it is deemed that the level of items such as metals and halides entering the scrubber will not adversely impact the FT or methanol catalysts, this step could be removed.

This study provides designs and costs for cleaning wood derived syngas in preparation for feed to liquid fuel synthesis units. Two different starting conditions, one with syngas derived from a low-pressure, indirect gasifier, and one from a high-pressure, direct gasifier, were evaluated. The goal was to provide NREL with a complete design package, including process flow diagrams, equipment specification sheets, mass and energy balances, capital and operating costs, and labor requirements, that can be used to evaluate the feasibility of biomass to chemicals technologies. The study also addressed how the designs would be impacted by changing flowrates and syngas compositions, so that the designs could be adapted to other process conditions.

The work was divided into three main task areas. The first Subtask (2.1) presented a list of possible gas clean-up technologies, with recommendations provided for the most suitable ones for additional analysis. The results of this study can be seen in Appendix D. Next, preliminary process flow diagrams were developed, along with an initial material balance (Subtasks 2.2.1 and 2.2.2). This was reviewed with NREL, and modifications made before the final design work began. The final phase consisted of performing equipment sizing, development of costs, and scaling analysis (Subtasks 2.2.3 through 2.2.7).

A variety of resources were used throughout the project to produce the final designs. In gathering the initial technology data, previous team studies, literature reviews, vendor information, and NREL input were all used to establish the items for consideration. Vendors and R&D facilities were especially helpful in providing data for novel technologies, such as tar cracking and liquid phase sulfur oxidation. Team members involved in biomass gasification, GTI and Emery Energy, provided valuable insight on reliability and feasibility issues.

HYSYS was used for modeling the overall process, with vendor input for specialty equipment. Design and performance of the amine system, LO-CAT<sup>TM</sup> unit, tar cracker, and process gas compressor were provided by vendors and estimated through other modeling work. All other process equipment was sized by the HYSYS program. Since the basis for the tar cracker, the NREL TCPDU, is not commercial, data from NREL was used, along with assumptions for bed fluidization needs and heat transfer requirements to produce a size estimate. Greater detail for the assumptions made can be found in Section 2.

Costing was performed in a similar fashion as design, with commercially available software, ICARUS, used for much of the equipment sized using HYSYS. All cost estimates use a second quarter 2005 basis. Quotes were obtained from vendors for unique and capitally intensive items, such as the process gas compressor, cyclones, ZnO beds, and LO-CAT<sup>TM</sup> unit. Industry derived cost curves were used for the amine system and as a check on other process items. Operating costs were developed from vendor supplied information and the energy balance. Finally, labor requirements are derived from a scale-up of a detailed study by Emery Energy specific to biomass gasification. For all results, comparisons were made throughout the study to results from previously developed NREL reports.

## Section 1

## 1.1 INTRODUCTION

The initial task for the Nexant team was to identify and evaluate all commercially available technology for clean-up of wood derived syngas. The technology list, with information on operating size ranges and conditions, materials of construction, and cleanup parameters, can be seen in Appendix D. After a review of technology options with NREL, flow schemes were developed for both the high and low pressure cases. The result of this analysis and justification for the technologies chosen is detailed in this section.

The compositions of the syngas from the gasifiers and the cleanup requirements are listed in Tables 1-1 and 1-2 below<sup>1</sup>. Each case being evaluated assumed a wood feedrate of 2,000 metric tonnes per day (MTPD).

	Syngas from BCL Gasifier	Syngas from GTI Gasifier
Temperature, °F	1,598°F (870°C)	1,598°F (870°C)
Pressure	33 psia (1.6 bar)	460 psia (32 bar)
Steam/bone dry feed	0.4 lb/lb	0.76 kg/kg
Compositions	Mol% (wet)	Mol% (wet)
H <sub>2</sub>	12.91	13.10
CO <sub>2</sub>	6.93	19.40
CO	22.84	8.10
H <sub>2</sub> O	45.87	50.70
CH <sub>4</sub>	8.32	7.80
C <sub>2</sub> H <sub>2</sub>	0.22	
C <sub>2</sub> H <sub>4</sub>	2.35	0.10
C <sub>2</sub> H <sub>6</sub>	0.16	0.20
C <sub>6</sub> H <sub>6</sub>	0.07	0.30
Tar (C <sub>10</sub> H <sub>8</sub> )	0.13	0.10
NH <sub>3</sub>	0.18	0.10
H <sub>2</sub> S	0.04	0.04
Gas Yield	0.04 lbmol of dry gas/lb bone dry feed	0.05 lbmol of dry gas/lb bone dry feed
Char Yield	0.22 lb/lb bone dry feed	0.0514 lb/lb bone dry feed
H <sub>2</sub> :CO molar ratio	0.57	1.62

#### TABLE 1-1 SYNGAS COMPOSITIONS AND OPERATING PARAMETERS

<sup>&</sup>lt;sup>1</sup> Information provided by Pamela Spath, NREL.

The gas pressure assumed from the BCL gasifier, 33 psia, is higher than initially evaluated during this project. Preliminary investigations were performed using a syngas pressure of 23 psia. Raising the pressure by 10 psia allows for a simpler and more reliable design, by allowing a water wash upstream of the compression stage.

Process	Contaminants	Level	Source/Comment
	Sulfur	0.2 ppm	Dry, 1981
		1 ppmv	Boerrigter, et al, 2002
		60 ppb	Turk, et al, 2001
Fischer-Tropsch Synthesis	Halides	10 ppb	Boerrigter, et al, 2002
	Nitrogen	10 ppmv NH3	Turk, et al, 2001
		0.2 ppmv NOx	
		10 ppb HCN	
	Sulfur (not COS)	<0.5 ppmv	Kung, 1992
Mathemal Curthenia		(<0.1 ppmv preferred)	
Methanol Synthesis	Halides	0.001 ppmv	Twigg and Spencer 2001
	Fe and Ni	0.005 ppmv	Kung, 1992

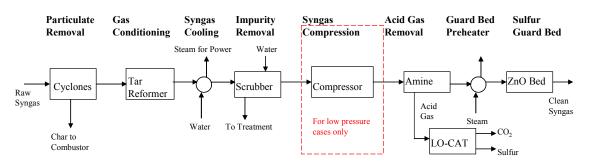
## TABLE 1-2 GAS CLEANUP REQUIREMENTS

The main impurities in the syngas exiting the gasifier that must be removed are char, tars, hydrocarbons, sulfur, and CO<sub>2</sub>. In addition, trace contaminants such as ammonia, metals, halides, and alkali species were of sufficient concern that equipment was added to remove them as well. Finally, the syngas must also be adjusted to obtain the appropriate  $H_2$ /CO ratio.

## 1.2 PROCESS DESCRIPTION AND RATIONALE

A schematic for the process design developed for both cases can be seen in Figure 1-1. Both the low and high pressure cases used very similar processes for syngas clean-up: particulate removal with cyclones, tar reforming, cooling and water scrubbing, acid gas removal with amine, and sulfur polishing. The main difference between the cases is the inclusion of a compression step in the low-pressure case. A detailed description of each design is addressed in this section.

## FIGURE 1-1 GENERAL SYNGAS CLEAN-UP PROCESS FLOW



## 1.2.1 Low-Pressure Syngas Process Description

#### Particulate Removal

The syngas exiting the gasifier contains impurities that must be removed in order to meet the specifications required for methanol or FT synthesis. Cyclones are used as the initial step in the gas cleanup process to remove the bulk of the char entrained in the syngas stream. This technology is standard in industry due to its low cost and high level of performance for removing particulates. Syngas from the low-pressure gasifier is sent through four parallel cyclones operating at 1598°F and 33 psia.

#### Tar Reforming

Syngas is fed to a tar reformer to remove tars, light hydrocarbons, and ammonia before any additional gas treating or cooling. Reforming must occur prior to cooling the syngas to prevent tar condensation and deposition on downstream equipment. The tar reformer was modeled using NREL's "goal design" reactor conversion for the Thermochemical Pilot Development Unit (TCPDU). Table 1-3 shows the assumed reactor conversion rate as provided by NREL. In the tar reformer, tars (mono and polyaromatic compounds) and light hydrocarbons such as methane, ethylene, and ethane are converted to H<sub>2</sub> and CO. Ammonia is converted to N<sub>2</sub> and H<sub>2</sub>. Since the reactor effluent contains about 1.3 mol% CH<sub>4</sub>, and 0.2 mol% of other hydrocarbons, additional downstream steam reforming was deemed not necessary. This conclusion was confirmed by NREL<sup>2</sup>.

Compound	% Conversion
Methane (CH <sub>4</sub> )	80
Ethane (C <sub>2</sub> H <sub>6</sub> )	99
Ethylene (C <sub>2</sub> H <sub>4</sub> )	90
Tars (C10+)	99.9
Benzene (C <sub>6</sub> H <sub>6</sub> )	99
Ammonia (NH <sub>3</sub> )	90

## TABLE 1-3 TAR REFORMER PERFORMANCE

Syngas exiting the tar reformer enters another cyclone to separate both entrained reforming catalyst and any residual char. The solids are then sent to a catalyst regenerator. The catalyst is sent to a regenerator vessel, where char and residual carbon is combusted. The hot, regenerated catalyst is then recycled back to the reactor vessel, acting as the heat source for the reforming reactions.

## Syngas Cooling

The remaining gas treatment steps require the syngas to be at a much lower temperature. Therefore, the gas is cooled in three stages from 1598°F to 225°F prior to scrubbing. The heat

<sup>&</sup>lt;sup>2</sup> Nexant team discussion with Pamela Spath, April 2005.

recovered from the process is used for steam generation throughout the system. The process design has been optimized as much as possible to use this steam, reducing the plant utility load. Integration was limited to the needs of the clean-up section; broader heat integration with the overall thermochemical platform or biomass refinery may lead to additional efficiency gains.

#### Scrubbing and Quench

The syngas is sent to the Syngas Venturi Scrubber, C-200, to remove any remaining ammonia, particulates, metals, halides, or alkali remaining in the system. The water circulation rate to the scrubber is adjusted such that the exiting syngas is quenched to the appropriate temperature for feed to the first stage of the compressor.

#### Compression

Any residual condensate in the syngas exiting the scrubber is removed in the Syngas Compressor KO Drum, V-300. The cooled syngas stream is compressed to 445 psia using a 4-stage centrifugal compressor with interstage cooling. The compressor is modeled assuming a horizontally split centrifugal design, with a polytropic efficiency of 78% and 110°F intercoolers. After discussion with compressor vendors<sup>3</sup> and internal analysis by Nexant, it was determined that this type of compressor is appropriate for this gas flowrate, pressure ratio, and reliability requirements. While an integrally geared compressor was considered due to its lower cost, this type of compressor was not recommended due to the high flowrate and reliability required. The discharge pressure is designed such that the compressed gas is at the operating pressure range for FT synthesis.

## Sulfur Removal

Originally, the scheme developed was use of LO-CAT<sup>TM</sup> and ZnO polishing for H<sub>2</sub>S removal, followed by amine for CO<sub>2</sub> removal. After discussions with NREL, this was modified so that amine was used for both H<sub>2</sub>S and CO<sub>2</sub> removal. The ZnO beds remained in the design as a guard/polishing step after the amine unit, while the LO-CAT<sup>TM</sup> unit is now used to remove H<sub>2</sub>S from the acid gas stream. The benefit of this design is reduced load on both the LO-CAT<sup>TM</sup> and ZnO units; the flow going to the LO-CAT<sup>TM</sup> unit in this case is now only the acid gas stream instead of the entire syngas stream, and the inlet H<sub>2</sub>S concentration at the ZnO bed is expected to be lower. This should increase the lifespan of the ZnO catalyst.

The syngas exiting the gasifier contains ~400 ppmv of H<sub>2</sub>S. An amine unit with a high circulation rate can reduce the syngas sulfur concentration to below 10 ppmv, with a target of 2-3 ppmv. Due to the high amount of CO<sub>2</sub> removal required, it is this component that drives the circulation rate and unit size, not H<sub>2</sub>S. The ZnO beds are used as a polishing step to reduce the sulfur concentration to the < 0.1 ppmv level required for methanol and FT synthesis. The gas exiting the amine absorber is heated to the operating temperature of the ZnO beds, 750°F.

For the low-pressure case, DEA was selected, while MDEA is used for the high-pressure case. This selection is based on design simulation runs by matching the desired  $CO_2$  and  $H_2S$  removal

<sup>&</sup>lt;sup>3</sup> Consultation made with both Elliott Compressor and GE.

requirements to the selectivity of the amine solvents. Attempts were also made to choose solvents that minimized net energy requirements.

## Water-Gas Shift and CO2 Removal

FT synthesis requires a  $H_2/CO$  ratio of 2:1, and methanol synthesis requires the following stoichiometric ratio of  $H_2$ , CO, and CO<sub>2</sub>:

$$(H_2 - CO_2) / (CO + CO_2) = 2$$

The syngas stream exiting the ZnO beds has a  $H_2/CO$  ratio of 1.7 and a stoichiometric ratio of 0.89, which are inadequate for FT or methanol synthesis. A combination of water injection into the tar cracker, followed by CO<sub>2</sub> removal in the amine unit, has been selected to adjust these ratios. In methanol synthesis,  $H_2$  will react preferentially with CO<sub>2</sub> over CO to form methanol. This results in a significantly lowered methanol yield, greatly impacting the process efficiency. In FT synthesis, CO<sub>2</sub> acts as a diluent; however, for a design in which the off-gas from the FT reactor is recycled back to the reactor to improve conversion, removal of CO<sub>2</sub> is necessary to prevent CO<sub>2</sub> buildup in the reactor.

The initial designs for the low pressure system incorporated a shift reactor instead of water injection to assist in obtaining the necessary composition ratios. Further analysis and review with NREL led to the determination that a shift reactor was unnecessary, and that steam injection into the tar cracker is sufficient to perform the required shift. Elimination of this unit operation helps to reduce the overall system cost.

 $CO_2$  removal can be achieved through different processes such as chemical (amine) or physical (Selexol or Rectisol) absorption, as outlined in Appendix D. The syngas stream entering the  $CO_2$  removal unit is at about 420 psia and 110°F. Since physical absorption process is best suited for high pressure (>700 psia) and low temperature systems, an amine system was selected to remove  $CO_2$  from the syngas. In addition to the syngas already possessing the appropriate operating conditions for chemical absorption, an amine system is also likely to be less expensive than the Selexol or Rectisol system. A side-by-side cost analysis from vendors would be necessary to confirm the optimal design. Approximately 98% of the  $CO_2$  in the syngas stream must be removed in order to meet the stoichiometric ratio requirement for methanol synthesis.

The treated syngas exits the amine absorber at approximately 110°F and 440 psia. The treated syngas is sent to either the methanol or FT reactor. For methanol synthesis, the treated gas is compressed and heated to the operating conditions of the methanol reactor, about 1160 psia and 460°F. For FT synthesis, the treated gas is heated to 350°F.

## 1.2.2 High-Pressure Syngas Process Description

The cleanup process scheme for the syngas from the high-pressure gasifier is similar to that of the syngas from the low-pressure gasifier with the exception of the syngas compression step, differences in the heat balances, and process unit size variations due to different syngas compositions and conditions. Information about these differences is presented below.

Similar to the low-pressure case, high-pressure syngas is sent through a series of cyclones to remove the bulk of the char entrained in the syngas stream. The syngas is then sent to the tar reformer for removal of tars, methane, other light hydrocarbons, and ammonia. Steam is added to the syngas entering the tar reformer so that the shift reaction that occurs in the reformer can yield the required  $H_2/CO$  ratio for methanol or FT synthesis. Due to a more appropriate synthesis ratio in the raw syngas stream, less steam is required relative to the low-pressure case. The reformer effluent is then sent to the water scrubbing unit for removal of residual char, alkali, metals, halides, and ammonia.

Following the water scrubbing unit, the syngas is sent to an amine unit where MDEA is used for the removal of both H<sub>2</sub>S and CO<sub>2</sub>. As in the low-pressure case, a LO-CAT<sup>TM</sup> unit is used for sulfur recovery, while ZnO beds are used for reducing the syngas sulfur content to below < 0.1 ppmv H<sub>2</sub>S. Rationale for process selection of the sulfur and CO<sub>2</sub> removal units is similar to that of the low-pressure syngas case, although MDEA was used instead of DEA in the amine system. The treated syngas is sent to either the methanol or FT reactor. For methanol synthesis, the treated gas requires compression and pre-heating to 1160 psia and 460°F prior to entering the methanol reactor. For FT synthesis, the treated gas requires pre-heating to 350°F.

## 1.3 DISCUSSION

## 1.3.1 Technologies Not Chosen

As presented in Appendix D, a list of technologies was provided for performing the various gas cleanup tasks required. From this list, specific technologies have been selected for each of the designs presented here. Below is a list of the technologies that were not chosen, and the rationale behind those decisions.

## Particulate Removal

*Ceramic and Metal Candle Filters:* Candle filters could be used in place of cyclones for char and catalyst separation from the syngas stream. Little commercial experience exists in operating these types of filters at the temperatures  $(1500^{\circ}F^+)$  that the cyclones operate under. At this temperature, only ceramic filters could be considered. A recent study performed by Nexant for the DOE's National Energy Technology Laboratory<sup>4</sup> examined replacing a third stage cyclone with a ceramic candle filter. The cost of this high temperature filter, even assuming an "nth plant design", did not justify the change. Because of the limited commercial experience and high cost, these options were eliminated.

*Baghouse Filters:* As with candle filters, baghouse filters are not appropriate for high temperature applications. Therefore, they cannot replace the cyclones as an effective solids removal option.

*Electrostatic Precipitators:* Since dry ESPs can only operate up to  $\sim$ 750°F and wet ESPs up to  $\sim$ 200°F, this option cannot replace cyclones for solids removal. In addition, the high cost and waste streams produced make them unattractive relative to other filtration options.

<sup>&</sup>lt;sup>4</sup> "Gasification Alternatives for Industrial Applications: Subtask 3.3—Alternate Design for the Eastern Coal Case, DOE Contract DE-AC26-99FT40342, April 2005.

#### Tar and Hydrocarbon Removal

*Wet Scrubbing:* Due to the relatively low content of tar in the syngas stream and the non-power application being considered, wet scrubbing could be considered a viable option for tar removal. However, inclusion of a wet scrubber may make a steam reformer necessary to remove hydrocarbons from the system. In addition, wet scrubbing for tar removal creates considerable waste removal and treatment issues and lowers process efficiencies. A detailed analysis comparing the current configuration with a wet scrubber/steam reformer would be of interest to confirm these assumptions.

*Hydrocarbon Reforming (SMR/POx/ATR):* Due to the low content of hydrocarbons exiting the tar cracker, it was determined that this step was unnecessary. Both FT and methanol synthesis reactors should be able to handle the quantity of hydrocarbons without severely impacting performance.

**Other Technologies:** During the course of the design work for the current configuration, other alternatives, such as injection of cracking catalyst directly into the gasifier and changes in gasifier operation, were identified. Limited empirical data for these technology options make them impractical for design use at this time.

#### Sulfur Removal

**LO-CAT**<sup>TM</sup>: The initial designs for sulfur removal from the syngas stream used the LO-CAT<sup>TM</sup> technology due to the low net syngas sulfur content. Redesigns of the combined sulfur and  $CO_2$  removal system demonstrated that using LO-CAT<sup>TM</sup> for sulfur recovery and amine for sulfur and  $CO_2$  removal was more economic.

*Physical Solvents:* As can be seen in Appendix D, physical solvents (Rectisol/Selexol processes, for example) typically operate at low temperatures and high pressures. Changes in the stream pressure leaving the scrubber/quench may be required prior to entering a physical solvent unit for optimum performance, whereas the current process conditions are more appropriate for feed to an amine system. In addition, previous Nexant studies have determined little to no cost benefit in implementing a physical solvent system over other treatment methods for systems of this nature. A more in-depth analysis would be required to confirm the cost difference between physical absorbents and an amine/ZnO treatment system.

*COS Hydrolysis:* Due to the limited COS expected to be produced from a biomass gasification system, this removal step was omitted.

## 2.1 INTRODUCTION AND METHODOLOGY

Design and cost estimates were obtained using three major sources:

- HYSYS and ICARUS were used to obtain design and cost estimates for generic equipment such as vessels, pumps, compressors, and heat exchangers. The design basis was agreed upon after the submission of the design information outlined in Section 1.
- Vendor quotes were obtained for unique and specialized equipment such as cyclones, ZnO catalyst/reactors, LO-CAT<sup>TM</sup> sulfur absorption, and compressors. Some items, such as compressors and blowers, were estimated both by HYSYS/ICARUS and through vendor quotes in order to validate the results.
- The amine unit performance and energy requirements were estimated using commercially available software that is specific for amine unit modeling. Once performance requirements were obtained, an industry developed cost curve was used for estimating installed cost.

An updated set of PFDs can be seen in Appendices A and B. The design and cost estimates for the high-pressure and low-pressure cases are presented in the Equipment List and Data Sheets, which can be seen in Appendix C. The Equipment List groups process equipment by the following categories: reactors, cyclones, vessels, heat exchangers, compressors, pumps, turbines, and packaged units (the amine and LO-CAT<sup>TM</sup> units). Shown in the Equipment List are the following items:

- Unit size and weight
- Design duty (exchangers)
- Design temperature and pressure
- Power usage
- Materials of construction
- Price (uninstalled) on both a Q2 2004 and Q2 2005 basis
- Source for cost estimate
- Comments and notes

An installation factor of 2.57 was applied to all base equipment costs, with the exception of the process gas compressor, to arrive at the total installed cost. The installation factor was derived based upon previous experience and vendor estimates. An installation factor of 2.47 was used for the compressor based on previous detailed compressor cost analysis. The total installed cost for the low-pressure case is \$109MM, while the installed cost for the high-pressure case is \$76MM. The difference is largely due to the process gas compressor used in the low-pressure case.

## 2.2 KEY DESIGN ASSUMPTIONS

A complete description of the process and rationale for choosing the technologies in this deliverable can be seen in Section 1. Each case assumed a feedrate of 2,000 MTPD. Issues encountered when performing the unit designs are outlined below.

#### 2.2.1 Sulfur and CO<sub>2</sub> Removal

As mentioned in Section 1, DEA was selected for the low-pressure case, while MDEA is used for the high-pressure case. This selection is based on design simulation runs by matching the desired  $CO_2$  and  $H_2S$  removal requirements to the selectivity of the amine solvents. The level of  $CO_2$  removal is the major driving force in determining the amine system size and cost; without the need for  $CO_2$  removal, the unit cost decreases significantly.

#### 2.2.2 Tar Reforming

Design and cost estimation of the tar reformer/regenerator presented a challenge to the team. Because no commercial data exists on design or cost for the performance outlined by the "goal" TCPDU case, a number of assumptions have been made:

- Reaction temperatures equal to the inlet gas temperature (1598 and 1576°F). These temperatures are derived from conversations with NREL. Recent experimental studies at Iowa State University on catalytic tar destruction have demonstrated successful operation at ~1350 to 1550°F<sup>5</sup>. Sensitivity cases were run at 1472 and 1200°F; the results show that heat duty is strongly impacted by the reaction temperature. Since the catalyst is the heat carrier in the reaction, the reaction temperature will greatly impact natural gas use and catalyst circulation rates. Minimizing these factors will trade-off with catalyst activity as the reaction temperature is lowered. This may be an area for future optimization and testing at the TCPDU.
- Low pressure operation for the regenerator to cut down on combustion air blower costs. This design is assuming the use of a pressurized rotary lock to increase recycle catalyst pressure. There is the risk that a rotary lock may be inadequate for this service due to the high catalyst circulation rates leading to premature erosion. If this is the case, either a lockhopper system or pressurized regenerator vessel would need to be included, significantly adding to the cost.
- Catalyst recycle rate based entirely off of thermodynamic requirements. Because of the endothermic reforming reactions, the regenerated catalyst must carry the heat necessary to maintain reactor temperature.
- Catalyst heat capacity of 0.25 Btu/lb/°F
- Plug flow within the reactor, with a Gas Hour Space Velocity (GHSV) of 2000/hr, to establish the basis for the bed volume and catalyst inventory. The calculated cracker

<sup>&</sup>lt;sup>5</sup> Zhang, R., Brown, R., Suby, A., Cummer, K., "Catalytic Destruction of Tar in Biomass Derived Producer Gas", Energy Conversion and Management, Vol. 45, pp. 995-1014, 2004.

bed length was multiplied by a factor of four to account for deviations from ideal plug flow.

 Bed diameter calculated by first estimating the minimum and maximum bed fluidization velocities, then an average of these estimates taken. Fluidization velocities calculated from catalyst and syngas properties.

Both ASPEN and HYSYS were used to model these systems, with all necessary thermodynamic and kinetic assumptions included. The results from both simulations came out very close to one another with a very high heat duty (~150 to 170 MMBTU/hr) and catalyst circulation rate (~24,000 to 29,000 MTPD) in each case. While the cost of the actual vessels are not very high (\$1.3MM to \$1.5MM), the catalyst load is substantial, and costs could be high based on what assumptions are made for catalyst losses and system maintenance requirements. Since the catalyst is regenerated in the process, minimizing losses is key to reducing operating costs.

## 2.2.3 Cyclones

A number of assumptions were made for the particle size distribution, efficiency, and outlet particle loading. Since no explicit direction was given by NREL, assumptions using experimental data from small-scale gasifiers was assumed and given to vendors for sizing (99%+ particulate removal and an average particle size of 50 µm).

## 2.2.4 Heat Integration

The process heating and cooling needs were evaluated and heat integration performed to maximize heat recovery. The process design includes a steam cycle that recovers the majority of the process heat by generating steam. For hot process streams that could not be integrated in the steam cycle, cooling water was used to provide cooling duty. A steam turbine is included in the design to generate power from the excess process steam.

## 2.2.5 Methanol Compressor

It was assumed that a clean syngas pressure of 1160 psia was required for methanol synthesis. Therefore, a compression system with interstage cooling has been included in the design.

## 2.3 OPERATING COSTS AND UTILITY REQUIREMENTS

Catalyst and chemical needs, along with utility requirements, can be seen in Tables 2-1 through 2-3. The units with the highest operating cost are the amine system and the tar cracker. Steam cost contributes the largest cost component for the amine unit. A portion of the steam required for the amine unit is extracted from the steam turbine, and the remainder is assumed to be imported. About 44,000 lb/hr of steam is imported for the low-pressure case, and 113,500 lb/hr for the high-pressure case. Imports may be unnecessary if excess steam from elsewhere in the gasification unit is available.

The other major source of operating cost is the catalyst requirement for the tar cracker. The tar cracker specifics were determined by estimating the minimum fluidization velocity, required space velocity, and the required heat duty demanded of the regenerated catalyst. The total

amount of catalyst is equal to the settled bed volume of the two fluidized beds, plus an additional 10% for transfer line inventory. Due to the very high heat load and quantity of gas to be handled, the initial catalyst loading is substantial: ~300 tonnes in the HP case, and ~830 tonnes in the LP case.

The remaining catalyst and chemicals cost are in-line with the assumptions made by NREL; in fact, some of the costs used by NREL in the biomass to hydrogen report are used here either for consistency, or because little other information exists. For example, it is unknown what the cost will be of tar cracker catalyst that can perform as expected in the NREL "goal" design.

Nexant has not made assumptions for the total yearly operating cost at this time; this cost could vary considerably based on the assumptions made for plant performance and the assumptions for catalyst, chemicals, and power costs. An estimate for operating cost should be performed for an entire integrated gasification unit or biorefinery, instead of the clean-up unit as a stand-alone facility. Suggestions for proper estimation and reducing operating costs include:

- An availability of 85 to 90% would be appropriate for this design
- Both low and high pressure designs would likely require steam imports. This could come from purchases or excess steam production elsewhere in the gasification plant
- A 0.01% per day catalyst loss in the tar cracker, as assumed by NREL in the "goal" hydrogen design, is appropriate for initial cyclone operation, but will likely degrade over time. Typical catalyst assumptions and make-up rates for similar technologies range from 0.01% to 0.1%.

If a loss rate of 0.01% is assumed, and costs for the ZnO beds are amortized over the year, the daily catalyst and chemical cost is \$1931/day for the low-pressure case, and \$1457/day for the high pressure case. This takes into account tar cracker losses, ZnO bed replacement, and LO-CAT<sup>TM</sup> requirements. This is shown in Table 2-1 below.

Variable	Amount Required	Cost	Notes
Tar Reformer Catalyst	Low- Pressure Case: 1,820,000 lbs High-Pressure Case: 662,000 lbs	Price: \$4.67/lb (NREL H <sub>2</sub> Report)	No commercial catalyst is currently available for this operation. Assuming a GHSV of 2000/hr, and a catalyst volume equal to the settled bed volume of the two fluidized beds plus 10% for transfer lines.
ZnO Catalyst	Low-Pressure Case: 777 cubic feet High-Pressure Case: 707 cubic feet	Price: \$355/cubic foot (Johnson Matthey).	Initial fill then replaced every year. Catalyst inventory based on H <sub>2</sub> S removal capacity from 2 ppmv to 0.1 ppmv.
Sulfur Recovery Chemicals	Low-Pressure Case: 1.7 Tonnes/Day of Sulfur Removal	Price: \$191/tonne sulfur removed (GTP Quote)	Assumes price for all LO-CAT™ chemicals required. Does not include utility requirements.
	High-Pressure Case: 2.4 Tonnes/Day of Sulfur Removal		

#### TABLE 2-1 CATALYST AND CHEMICAL REQUIREMENTS

Steam, water, natural gas, and combustion air requirements are similar between both the high and low pressure cases. The main difference is in the power and cooling requirements. This is mostly due to the syngas compressor; the large energy and interstage cooling duty required adds considerably more to the utility requirements. Some of the cooling duty is recaptured in the steam system.

High-pressure case utility requirements can be seen in Table 2-2 below.

		Load BHP		Elect. Power						Water, GPM		Cooling MMBTU/HR	Nat. Gas	Combustion Air
ltem No	Item Name	Norm.	Max (3).	ĸw	445 psig	85 psig	5 psig	psig	Cond.	Proc.	C.W. circ. (2).	Water	MMSCFD	MMSCFD
H-200	Quench Water Recirculation Cooler										2,232	22.3		
H-302	Lean Solvent Cooler										13,487	135.0		
H-303	Amine Stripper Reboiler					243.9			244					
H-305	Acid Gas Condenser										8,520	85.3		
H-400A	K-400 Interstage Cooler										1,046	10.5		
H-401	MeOH Reactor Preheater					17.61			17.6					
H-501	Blowdown Cooler										84	0.8		
K-100	Combustion Air Blower	1,022		762										
K-320	Flue Gas Blower	207		154							2	0.02		
K-400	MeOH Compressor - 2 Stages	8,388		6,257							84	0.8		
P-201	Quench Water Recirculation Pump	3		2										
P-300	Lean Solvent Pump	1,474		1,100										
P-500	Condensate Make-up Water Pump	. 1		. 1										
P-501	Deaerator Feed Pump	8		6										
P-502	Boiler Feed Water Pump	710		530										
R-xxx	Gasifier				139.6									
R-100	Tar Reformer				26									
R-101	Catalyst Regenerator												7.8	84.4
	LO-CAT unit	1,004		749			0.9			2,500				
M-501	Extraction Steam Turbine/Generator	(19,721)		(14,712)	(165.6)	(148.0)								
	TOTAL	(6,903)		(5,150)	0	114	1		262	2,500	25,454	255	8	84

TABLE 2-2 HIGH-PRESSURE CASE UTILITY REQUIREMENTS

Low-pressure case utility requirements can be seen in Table 2-3.

			Load BHP		Steam M Pounds per Hour				Water, GPM		Cooling MMBTU/HR	Nat. Gas	Combustion Air	
ltem No	ltem Name	Norm.	Max (3).	ĸw	85 psig	35 psig	5 psig	psig	Cond.	Proc.	C.W. circ. (2).	Water	MMSCFD	MMSCFD
H-200	Quench Water Recirculation Cooler										2,213	22.2		
H-300A	1 st Stage intercooler										12,188	122.0		
H-300B	2nd Stage intercooler										3,276	32.8		
H-300C	3rd Stage intercooler										2,766	27.7		
H-300D	Post compressor cooler										1,819	18.2		
H-402	Lean Solvent Cooler										11,388	114.0		
H-403	Amine Stripper Reboiler					150.6			151					
H-405	Acid Gas Condenser										2,900	29.0		
H-500A	K-500 Interstage Cooler										1,105	11.1		
H-501	MeOH Reactor Preheater				18.8				18.8					
H-601	Blowdown Cooler										61	0.6		
K-100	Combustion Air Blower	910		679										
K-300	Syngas Compressor - 4 Stages	38,786		28,934										
K-420	Flue Gas Blower	347		259							3	0.03		
K-500	MeOH Compressor - 2 Stages	8,717		6,503							87	0.9		
P-201	Quench Water Recirculation Pump	20		15										
P-400	Lean Solvent Pump	802		599										
P-600	Condensate Make-up Water Pump	1		1										
P-601	Deaerator Feed Pump	7		5										
P-602	Boiler Feed Water Pump	570		425										
R-xxx	Gasifier					73.47								
R-100	Tar Reformer					53								
R-101	Catalyst Regenerator												7.0	74.8
	LO-CAT unit	639.9		477			0.56			1,800				
M-601	Extraction Steam Turbine/Generator	(26,019)		(19,410)										
	TOTAL	24,781		18,486	0	44	1		169	1,800	37,806	378	7	75
NOTES:	<ol> <li>All Figures shown above represen () indicates normal utility make * indicates intermittent usage or m 2. CWS temperature is 80 F and CW 3. Utility consumption for max. load c</li> </ol>	iake, not ir R tempera	icludeo iture is	d in totals 100 F. M				tower	is not s	shown				

## 2.4 DIFFERENCES WITH NREL BIOMASS TO HYDROGEN DESIGN

In general, the cost of the clean-up section of the biomass to chemicals designs is more expensive than for the NREL Biomass to Hydrogen design<sup>6</sup>. There are three main reasons for this: more equipment necessary in the chemicals designs, the increase in steel prices from 2002 to 2005, and different engineering assumptions made in the chemicals case. Information on each reason will be elaborated upon below.

#### 2.4.1 Added Equipment to Chemicals Design

The two major unit operations that are new to this design versus the hydrogen cases are the amine unit and the syngas compressor for methanol synthesis. In the hydrogen cases, a LO- $CAT^{TM}$  unit and ZnO bed was used for H<sub>2</sub>S removal, while the PSA removed carbon dioxide. The chemicals cases also use the LO-CAT<sup>TM</sup> and ZnO units, but instead of a PSA, an amine unit is used for the bulk H<sub>2</sub>S and CO<sub>2</sub> removal. The cost for the amine units is driven largely by the need for CO<sub>2</sub> removal; due to the low H<sub>2</sub>S content in the syngas, the cost of the amine unit would be roughly half as much if CO<sub>2</sub> removal was not required. The LO-CAT<sup>TM</sup> unit is used in this case for clean-up of the acid gas stream from the amine unit instead of bulk H<sub>2</sub>S removal. Because of the CO<sub>2</sub> content and different operating requirements versus the hydrogen case, the quote provided by GTP is roughly double the price used in the hydrogen case.

<sup>&</sup>lt;sup>6</sup> Spath, P.; Aden, A.; Eggeman, T.; Ringer, M.; Wallace, B.; Jechura, J. (2005). Biomass to Hydrogen Production Detailed Design and Economics Utilizing the Battelle Columbus Laboratory Indirectly-Heated Gasifier. 161 pp.; NREL Report No. TP-510-37408.

In order to compress the clean syngas up to methanol synthesis pressure, a  $\sim$ 8,000 HP compressor is required. This unit was not necessary in the hydrogen case, adding to the overall cost. Taking into account a \$12MM credit by not using the PSA, the LP cost increases by  $\sim$ \$8.5MM, while the HP cost increases by  $\sim$ \$18.5MM due specifically to the extra equipment needed.

#### 2.4.2 Increase in Steel Price

NREL used 2002 as the cost basis for the biomass to hydrogen designs, while Nexant is using Q2 2005. The increase in steel price between 2002 and 2005 has been significant, impacting the prices quoted in the Nexant design. The Q2 2005 basis for hot-rolled steel is  $\sim$ \$400 to \$450/ton, up from  $\sim$ \$250 to \$300/ton in 2002<sup>7</sup>. Steel prices have been very volatile in the last 3 years due to strong worldwide demand, a sharp rise in energy prices, consolidation in the US steel market, and a weak US dollar.

Because of this basis difference, the 2002 NREL basis would need to be escalated not only for inflation but also for steel price in order to put it on the same basis as this study. It is difficult to place a blanket escalation factor on the design due to the impacts that steel price has on different pieces of equipment; for example, this may make up much of the difference in price in equipment like vessels and exchangers, but have less of an impact on compressor prices. Each unit should be evaluated independently to determine the impact that steel price has on overall unit cost.

## 2.4.3 Engineering Assumptions

A side-by-side comparison of all the major process units was performed for the HP and LP cases versus the NREL hydrogen design. A few differences were noticed that are outlined below. A direct comparison cannot be performed on units that were lumped into the "Gas Cleanup" section of the NREL design and not explicitly sized. While the major differences are outlined here, only a brief attempt at determining the cost difference has been made.

#### **Reactors and Columns**

**ZnO Beds:** While the size of the ZnO beds in this design is smaller than the hydrogen case, the installed cost is roughly double. This is likely due to the difference in steel price.

*Tar Reformer/Regenerator:* In the hydrogen design, this is included in the "Cleanup" costs, so no explicit design information is available. The NREL assumption for "Cleanup" took the average of a number of different studies; however, only one of these studies, Weyerhaeuser (2000), had a tar cracker. The "Cleanup" section for the Weyerhaeuser study was ~\$9MM greater than the other designs, implying that the majority of the cost may be due to the tar cracker cost. The NREL "Cleanup" assumption may be low since the hydrogen design has a tar cracker, yet only one of the studies used to obtain the "Cleanup" cost also has a tar cracker.

<sup>&</sup>lt;sup>7</sup> For more information, see the Bureau of Labor Statistics "Producer Price Series", along with Lazaroff, Leon, "Steel Regains Some Luster", Detroit Free Press, 25 July 2005

## Cyclones

Since these were part of the "Cleanup" average, no explicit design numbers were provided as part of the hydrogen study. Design quotes from vendors are used for this part of the plant in the chemicals design.

## Vessels

The Nexant estimate is higher than the hydrogen design due to 1) the venturi and quench being included as part of the "Cleanup" estimate, 2) larger vessel sizes for the steam system than what was assumed in the hydrogen design, and 3) steel prices. Depending on the price assumed for the venturi /quench in the hydrogen design, the Nexant estimate appears to be ~\$3MM greater than the hydrogen case.

## Heat Exchangers

A number of differences exist between the hydrogen and chemicals designs, making the installed cost for exchangers in the chemical production case ~\$4MM to \$6MM higher than in the hydrogen case:

- There is a large cost discrepancy between the exchangers downstream of the tar reformer. The Nexant designs are larger and considerably more expensive; Nexant assumed refractory lining, while it is unclear if this assumption is made in the hydrogen design.
- The Nexant design has a number of exchangers not included in the hydrogen design: amine precoolers (HP case), methanol compressor coolers (both cases), and ZnO coolers (both cases).
- A few of the exchangers in the hydrogen design are included in the "Cleanup" section, so it is difficult to make a direction comparison.

## **Compressors and Blowers**

As mentioned earlier, the syngas compressor for methanol synthesis adds ~\$7MM to the installed cost relative to the hydrogen case. This compressor was not necessary in the NREL hydrogen design.

There is a major difference between the NREL and Nexant assumptions for the syngas compressor in the LP case. While NREL shows an installed cost of ~\$12MM for a 30,000 HP compressor, Nexant estimates that a ~38,000 HP compressor is required at an installed cost of ~\$37MM (\$15MM for the equipment alone). The equipment cost comes directly from Elliott Compressor; checks on the validity of the estimate using cost curves, ICARUS, and other vendors show that this is within the +/- 30% estimate desired by the study. The NREL study assumed that an integrally geared compressor type would be appropriate, while this report uses a horizontally split centrifugal compressor recommended by vendors. Analysis using cost estimating software shows that this assumption is the main reason for the cost difference.

## Pumps

Both Nexant and NREL designs are in agreement in regards to the pumps.

#### Steam Turbine

The Nexant estimate is slightly higher than the NREL estimate, ~\$12MM installed versus \$10MM. This difference is likely due to steel prices.

The other difference that should be pointed out between the hydrogen and chemicals cases is the assumption made for the installation factor. NREL used a 2.47 installation factor, which is derived from literature sources. Nexant used 2.57 in both the HP and LP cases, except on the process gas compressor, where 2.47 is used. These numbers are derived independently from previous experience and vendor engineering estimates. While the factors are very similar to one another, this difference can make a 4% difference (\$2MM) on an equipment cost of \$20MM.

## 2.5 CHANGING FLOWS, CONDITIONS, AND COMPOSITIONS

Per the scope of work outlined by NREL as part of this project, Nexant has been asked to provide input on how the design estimates will be adjusted if the syngas flowrates or compositions vary. Information for both the high and low-pressure cases, along with the scaling factors appropriate for each major piece of process equipment, are outlined below.

## 2.5.1 Flowrate Impacts

In general the limits on process equipment sizes are usually the result of manufacturing restraints, transportation limits, and maintenance restrictions. For this evaluation, it was assumed that the throughput would be increased by 50% and the equipment size or capacity would increase accordingly. The affects of this change are discussed below with respect to both the low- and high-pressure cases.

## Low-Pressure Syngas Design Cases

For the Low-Pressure Syngas Design Cases some of the equipment has already reached size limitations that required multiple trains or parallel equipment. Thus, increasing the capacity by 50% will require more parallel equipment and a more complex and expensive piping manifold. Examples include:

- Gasifier Cyclones (4 required for the base capacity)
- Tar Reformer SG Cooler/Steam Generator (2 required)
- Tar Reformer SG Cooler/BFW Preheater (2 required)
- Compressor Interstage Cooling 1st stage (2 required)
- Syngas Venturi Scrubber/Quench Tower (2 required)

Thus, for a 50% increase in capacity, the design would require 6 gasifier cyclones, 3 of each major heat exchanger, and 3 venturi scrubbers.

Other items, such as the 1st Stage KO Drum, may require either a parallel unit or field construction due to equipment size and weight limitations during transportation. While the limits for ground transportation vary from state to state, typically, codes limit standard transport sizes to ~14 feet in width and height, 53 feet long and 80,000 pounds. Locating this facility in Iowa will mean that most equipment will be transported to the site either by rail or truck. Access to the Mississippi or Missouri Rivers may allow larger vessels to be used. For the 1st Stage KO Drum, the inside diameter would increase to about 16 feet (from a 13 foot diameter) at a capacity 50% greater than the base case. However, when considering transportation by road, auxiliary equipment such as nozzles and flanges must be taken into consideration. This item would be well beyond most road transportation limits in the U.S. To manage this limitation, options are either transportation by rail or barge, parallel pieces of equipment, or field fabrication.

Other equipment may exceed the maximum recommended size for a single train, and would require a second, parallel unit. This includes items such as the Syngas Compressor and the shell and tube heat exchanger for the Flue Gas Cooler/Steam Superheater service. In the latter case, the size of the heat exchanger is actually a maintenance issue. The diameter of the tube bundle of these units is larger than a normal bundle puller could handle (maximum limit is about 6-7 feet diameter). It then becomes an economic question of bringing in special maintenance equipment during turnarounds or using smaller, parallel process equipment.

#### High-Pressure Syngas Design Cases

For the High-Pressure Syngas Design Cases, most of the equipment is smaller than the corresponding equipment for the Low-Pressure Syngas Design Cases as a result of the high pressure operation. Only a few items, when scaled by +50%, would require a parallel unit. Two major exchangers, the Tar Reformer SG Cooler/Steam Generator and Flue Gas Cooler/Steam Superheater, were discussed above. Another area is equipment within the LO-CAT<sup>TM</sup> unit. These include the Inlet Gas KO Drum and the LO-CAT<sup>TM</sup> Oxidizer Vessel. The former would require a vessel with an inside diameter of over 17 feet and the latter would required an inside diameter of about 16 feet. As noted previously, the outside diameter (including nozzles and flanges) would be well beyond most road transportation limits in the U.S. Vendors for process items of this nature can provide input for the appropriate process configuration for this service.

Appropriate vessel sizing for the amine system is also of concern in this design. The amine system contains two relatively large columns – the scrubber and the regenerator. Considering a 50% increase in capacity, the column diameters will increase by about 20 to 25%. In particular, the regeneration column may exceed the transportation size limitations and thus, require parallel trains or field fabrication.

#### **General Information**

A plant that is 50% larger will require more plot area not only due to the larger equipment and storage, but due to offsite considerations. For example, the flare will have to be designed for a load that is 50% larger. This will require either a taller flare or moving the flare further away from the main process units. A higher flare may meet with height restrictions. Thus, the area that is restricted around the flare may increase.

#### Estimating the Capital Investment Cost

In most cases the capital cost for a capacity increase or decrease of 50% can be estimated using exponential methods. That is, the new capital cost can be estimated by using capacity ratio exponents based on published correlations and the following formula:

$$C_2 = C_1 (q_2/q_1)^n$$

where C stands for cost, q for flowrate, and where the value of the exponent n depends on the type of equipment. In reviewing the literature for the various exponents, some discrepancies in published factors are apparent due to variation in definition, scope and size. Technology has also advanced over time, making it less expensive to produce larger machinery now than in years past. In addition, new regulations dictate expenditures for environmental control and safety not included in earlier equipment. In the table that follows, the most recent literature information is listed. Traditionally, when a specific value is not known, an exponent value of 0.6 is often used for equipment and a value of 0.7 for chemical process plants (usually expressed in terms of annual production capacity). Table 2-4 gives typical values of n for most of the equipment included in these designs.<sup>8,9,10,11,12</sup>

Equipment	Size Range	Units	Exponent**
Reactor – fixed beds	N/A		0.65-0.70
Column (including internals)	300-30,000	Feed rate, million lb/yr	0.62
Cyclone	20-8,000	Cubic feet/m	0.64
Vessel – vertical	100-20,000	US gallons	0.30
Vessel – horizontal	100-80,000	US gallons	0.62
Heat exchanger (S&T)	20-20,000	Square feet	0.59
Venturi scrubber	N/A		0.60
Compressor – centrifugal*	200-30,000	hp	0.62
Blower*	0.5 - 150	Thousand standard cubic feet per minute	0.60
Pump*	0.5-40	hp	0.30
	40-400		0.67
Turbine		hp	0.81
Pressure discharge	20-5,000		
Vacuum discharge	200-8,000		
Motor	10-25	hp	0.56

#### TABLE 2-4 EXAMPLES OF TYPICAL EXPONENTS FOR EQUIPMENT COST VERSUS CAPACITY

<sup>&</sup>lt;sup>8</sup> Perry, Robert H., and Green Don W., Perry's Chemical Engineers' Handbook, 7th edition, page 9-69.

<sup>&</sup>lt;sup>9</sup> Walas, Stanley M., "Chemical Process Equipment – Selection and Design," Butterworths, page 665

<sup>&</sup>lt;sup>10</sup> Blank, L. T. and A. J. Tarquin, "Engineering Economy," McGraw-Hill

<sup>&</sup>lt;sup>11</sup> Peters, Max S. and Timmerhaus, Klaus D., "Plant Design and Economics for Chemical Engineers," McGraw-Hill, page 170

Remer, Donald S. and Chai, Lawrence H., "Design Cost Factors for Scaling-up Engineering Equipment," *Chemical Engineering Progress*, August 1990, pp 77-82

Equipment	Size Range	Units	Exponent**
	25-200		0.77
Package unit	N/A		0.75
Other	N/A		0.6 - 0.7

excluding driver

\* this estimating method gives only the purchase price of the equipment; additional installation cost for labor, foundations and construction expenses will make the final cost higher.

#### 2.5.2 Composition Impacts

The major units that will be impacted by a large change in syngas composition are the tar reformer and the venturi scrubber. Due to the relatively low concentration of sulfur in the syngas stream, +/-50% fluctuations in the H<sub>2</sub>S content should not impact how the sulfur removal system is designed. Significant changes in the inlet H<sub>2</sub>/CO ratio may also require modifications of the design in order to establish the appropriate downstream composition.

The obvious change that will influence the design of the tar reformer is the amount of hydrocarbons in the syngas from the gasifier. Currently, the design is assuming that a separate reformer is not necessary, with the tar reformer converting most hydrocarbons exiting the gasifier. If either the hydrocarbon yield increases or the tar reformer conversion is lower than planned, a separate reformer for light hydrocarbons should be considered. The amount and type of hydrocarbons will affect the operating conditions which will in turn affect the water gas shift reaction. A change in the  $H_2$ /CO ratio may require divorcing the shift reaction from the tar reformer (i.e., a separate shift reactor instead of just adding steam to the tar reformer).

A 50% increase in particulates may require different/larger cyclones or a redesign of the venturi scrubber in order to handle the larger load. This is largely controlled by the gasifier operation; reliable performance data should be established prior to deciding upon a particulate removal scheme. Higher particulate loading than planned can significantly hurt overall plant performance.

A 50% increase in H<sub>2</sub>S will not affect the sulfur recovery processes. LO-CAT<sup>TM</sup> can handle between 150 lbs to 20 tonnes of sulfur per day, and concentrations between 100 ppm and about 10% H<sub>2</sub>S. Even at 50 percent more H<sub>2</sub>S, the concentration still remains within the operating limits for LO-CAT<sup>TM</sup>. In addition, the solvent circulation rate in the amine unit can be increased to remove additional H<sub>2</sub>S if the sulfur concentration is higher than expected.

## 2.6 FOLLOW-UP AND AREAS FOR FURTHER STUDY

The analysis performed sets the base case for the clean-up section of two different biomass-tochemicals designs. After in-depth analysis of these cases, the team has identified a number of areas for further study:

 Alternatives for Tar Removal: A number of assumptions have been made for sizing and costing of this unit. Greater study and analysis, both in the laboratory and through simulations, should be performed to determine if the methods used are valid. In addition, alternative tar removal technology should be considered, including:

- Introduction of tar cracking catalyst into the gasifier. Typically, this has not been done due to concerns with deactivation and erosion.
- Gasifier operation to reduce hydrocarbon yields.
- Using a water wash for tars, followed by a standard reformer for hydrocarbons.
   While this increases the cost of quenching and wastewater handling, the cost tradeoff may be economic.
- Process Integration, Gasification Systems and Biorefinery: Integration of the cleanup section with the other parts of the gasification plant will provide a better picture of the overall plant costs. In addition, use of this thermochemical platform has been considered for future application into an integrated "biorefinery". This base case could be used for a determination of the process requirements and offerings that a thermochemical platform could provide.
- Alternate CO<sub>2</sub>/Sulfur Removal Steps: Based on the design information provided and past studies that have been examined, the steps incorporated for CO<sub>2</sub> and sulfur removal has been determined to be appropriate at this stage. A cost comparison of amine versus physical solvents and new technologies for acid gas removal would provide additional data to confirm the appropriate use of amine in this design.

New technology is currently being explored to remove sulfur without having to cool to 110°F or below. Since none of this technology is currently commercial, it has not been evaluated for use in this design. If available however, warm sulfur clean-up may increase efficiency in this design, by reducing the amount of reheat necessary prior to entering the shift reactor.

• *Other Impurities in the Syngas:* For the low pressure case, a scrubber has been included to remove residual ammonia, and any metals, halides, or alkali remaining in the system. If it is deemed that the level of these impurities entering the scrubber will not adversely impact the FT or methanol catalysts, this step could be removed.

## Section 3

## 3.1 SUMMARY

The labor projections for the 2000 MTPD biomass gasification plant are based on a combination of 1) models developed from Emery Energy's 70MWe Gasification Plant design completed under prior DOE contracts, 2) additional "adders" for the scale and complexity (chemical plant nature / hydrogen production) of the 2000 MTPD plant being considered, and 3) previous experience of Nexant and other team members. The high pressure, oxygen-blown, 2000 MTPD plant requires labor skills with slightly greater operating experience than power-only facilities, and thus commands a premium for these skills.

The labor rates derived from Emery's 70 MWe Biomass IGCC (1200 MTPD plant) case were ~\$1,650,000 per year (not including subcontracted services) versus the \$2,274,720 projected for the labor costs for the 2000 MTPD biomass to chemicals design. This difference of roughly \$625,000 represents the higher level of experience needed for the larger plant, greater materials handling rates, and increased labor for plant maintenance. A discussion of the reasons for this difference, along with differences between the recent NREL Biomass to Hydrogen report, is contained below. Some of the main differences with the NREL Hydrogen report include different job descriptions, the use of a back-up shift crew, utilization of contract labor, and lower assumptions for overhead costs.

## 3.2 LABOR REQUIREMENTS

The following labor categories and positions will be required for the 2000 MTPD biomass plant.

- General Plant Manager: Responsible for all personnel and plant decisions, including new employee hiring, operator training, fuel contracts, maintenance contracts, general equipment purchases, external communications, and operating schedules. Engineering degree required, with 10+ years of chemical plant operating experience. Salary of \$100,000/yr.
- Administrative Assistant/Company Controller: Support the general plant manager, manages personnel records, completes company payroll, manages time accounting records, manages company benefits, employee investment accounts, and insurance enrollments. Accountant degree required with 5+ years of experience. Salary of \$45,000/yr.
- Secretary/Receptionist: Supports the General Plant Manager and Company Controller. Receives visitors, answers phone, and attends to office administrative duties. Salary/Wages of \$25,000/yr.
- *Laboratory Manager:* Oversees all laboratory equipment and laboratory technicians. Responsible for product quality; testing performed both on finished product and intermediate streams (via on-line equipment and sample draws). Works straight days, with some overtime possible. Salary/Wages of \$50,000/yr.
- *Laboratory Technician:* Responsible for sample gathering, analytical equipment maintenance, and laboratory testing. Works straight days, with some overtime

possible. Shift operating crew can assist with some sample gathering as necessary; contract equipment technicians can assist with analytical equipment repair as necessary. Salary/Wages of \$35,000/yr.

- *Shift Operating Crew:* The plant will be operated by a four-member crew shift each week, with responsibilities defined below:
- *Shift Superintendent.* The shift superintendent is the chief operator who mans the control station and simultaneously directs the activities of the shift crew. The shift superintendent is a degreed engineer who understands the plant, understands the technical and physical operations, and makes key operating decisions. The shift superintendent ensures compliance with plant quality, safety, industrial hygiene, and environmental requirements. 5-10 years of chemical plant operating experience is preferred for this position. Salary of \$75,000/yr.
- **Support Operator.** The support operator aids the shift superintendent with plant operation. The support operator is also tasked with bulk material handling such as feedstock receipts/inspection/weigh-in and ash weigh-out/disposal shipments. The support operator attends to feed and ash sampling/characterization, waste water disposal sampling, and provides general plant support in relief of the shift superintendent. The support operator is also tasked with monitoring plant emissions rates, including daily/weekly calibration of effluent gas monitors. The support operator verifies that plant operating records and daily logs are correct. This position coordinates fuel characterizations and waste water analyses. A novice degreed engineer or experienced technician is sufficient for this position. Salary of \$45,000/yr
- Millwright. The shift millwright conducts hourly and daily equipment inspections, safety rounds, completes scheduled equipment process maintenance, supports equipment maintenance and equipment replacements, contracts and supervises crafts such as pipe fitters, electricians, welders, and special instrument technicians when such functions exceed the millwright's capabilities. The millwright preferably has an associate degree in mechanical, industrial, or design engineering technology with 5-10 years experience. Salary of \$60,000.
- *Millwright Assistant/Yard Labor*. Supports millwright and accompanies millwright and contracted crafts, particularly during dangerous work activities, such as confined space entries and working from heights. The millwright assistant supports tool setup, job errands, and plant cleanup. Salary of \$35,000.

Shifts run for 12 hours with two crews per day. Crews report to work 30 minutes prior to the shift turnover to perform receive shift operating instructions and to pass information on critical operations and maintenance. Each crew member is allotted 30 minutes for a meal break. Thus, each shift extends 12.5 hours, with 0.5 hours meal break, or 12 hours of labor. Crews operate on a 4 days on / 4 days off rotation. This requires 84 hours on average per crew member for any two-week pay period.

Five complete shift teams are engaged. The fifth crew provides coverage for individual vacations, sick leave, and holidays. The fifth crew also fills in for continuing training and for

new hire training. The fifth crew also supports ongoing maintenance and periodic outage/turnaround planning. In addition, the fifth crew supports updates to control system programming, data collection, and instruments. The millwright assistant on the fifth crew supports plant cleanup and janitorial activities. The fifth crew works 40-hour straight days when not substituting for members of the four-crew rotation.

Table 3-1 summarizes the plant operating labor by category, salary, and total cost.

Position	Number	Base Salary or Hourly Rate	Annual Overtime and Holiday Hours	Overtime Rate	Total Annual Cost
General Plant Manager	1	\$100,000	N/A	N/A	\$100,000
Company Controller	1	\$45,000	N/A	N/A	\$45,000
Secretary/ Receptionist	1	\$25,000	None	N/A	\$25,000
Laboratory Manager	1	\$50,000	240	\$30	\$57,200
Laboratory Technician	2	\$35,000	240	\$22.50	\$80,800
Shift Superintendent	5	\$75,000	680	\$45	\$405,600
Support Operator	5	\$45,000	680	\$25	\$242,000
Millwright	5	\$60,000	680	\$32.50	\$322,100
Millwright Assistant	5	\$15.00/hr	560	\$22.50	\$144,000
Total Base Salaries and Wages					\$1,421,700
General Overhead and Benefits (60% of total salaries)					\$853,020
Total Base Wages and Benefits					\$2,274,720
Subcontracted Crafts					
Welder	\$80/hr	1200			\$96,000
Electrician	\$75/hr	640			\$48,000
Pipe Fitter	\$65/hr	600			\$39,000
Insulator/Painter	\$60/hr	400			\$24,000
Carpenter	\$55/hr	400			\$22,000
Instrument Technician	\$90/hr	400			\$36,000
Total Subcontracted Labor					\$265,000
Total Labor and Benefits (Operating Labor Cost)					\$2,539,720

#### TABLE 3-1 LABOR COSTS

## 3.3 DIFFERENCES WITH EMERY ENERGY 70 MWE CASE

Both the complexity and size of this facility increases the labor costs over what Emery Energy has assumed for their 70 MWe biomass gasification facility. The size of the unit (1200 MTPD vs. 2000 MTPD) slightly increases the number of shift workers and contract hours required, but does not increase the plant management or engineering requirements. This represents an economy-of-scale advantage enjoyed by larger gasification facilities; while the total labor requirement is greater than the 1200 MTPD facility, the marginal amount of labor required decreases as plant size increases.

This design contains additional equipment than what is assumed in Emery Energy's 70 MWe facility design. While this design does not contain a gas turbine, steam turbine, or HRSG, additional equipment includes enhanced sulfur removal (an amine system and ZnO beds), chemicals synthesis equipment, and tar cracking. It is this increase in complexity, rather than the increase in size, that adds the majority of the increase in labor costs.

## 3.4 DIFFERENCES WITH NREL BIOMASS TO HYDROGEN CASE

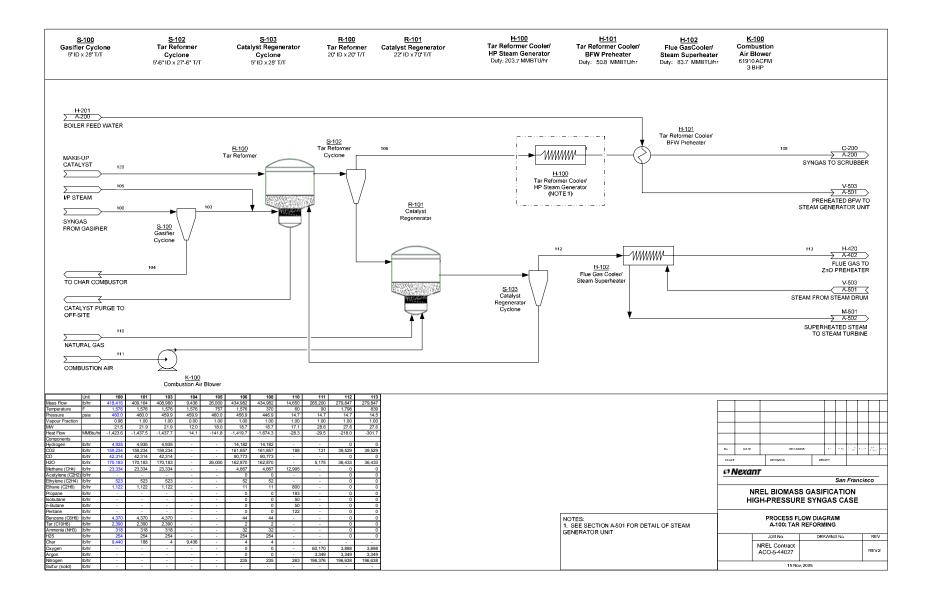
In the 2005 study, NREL made assumptions for the labor requirements necessary for a 2000 TPD wood gasification to hydrogen plant. The size being considered in this design is exactly the same, and the complexity is roughly the same as the NREL case. The only main difference is the inclusion of chemicals synthesis equipment, which takes the place of the PSA and related equipment required for hydrogen production.

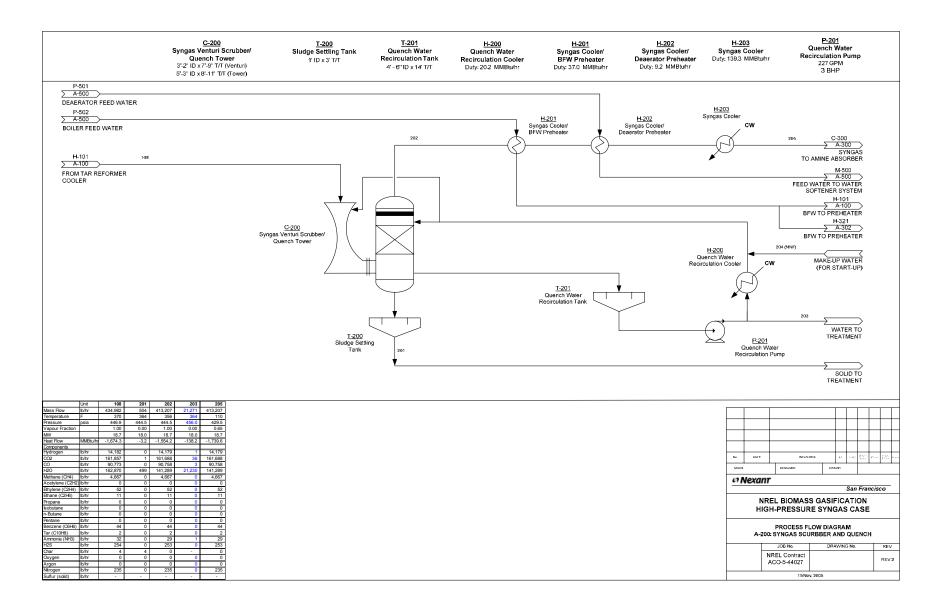
The labor requirements developed for the chemicals synthesis cases are lower by almost \$1.5MM due to the assumptions made by the Nexant team. The main differences are highlighted below:

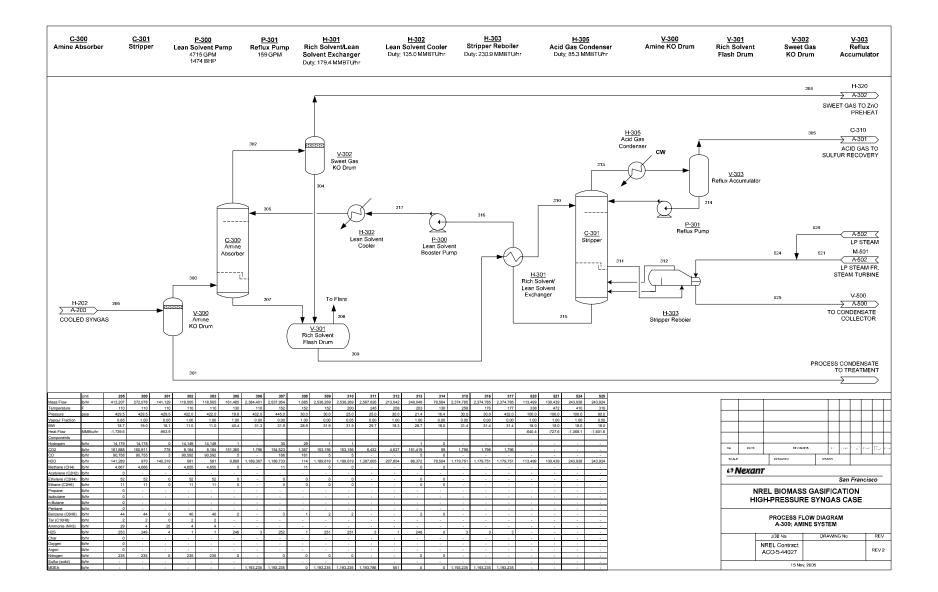
- *Salary Assumptions:* In general, slightly higher salaries are assumed in the chemicals synthesis design for employees such as the plant manager, engineers, and operators. Higher salaries may be necessary to attract workers to facilities employing complicated and novel technologies.
- *Administrative Assistants:* Instead of the three assistants assumed by NREL, this design assumes only two: the company controller/administrative assistant and the main receptionist. The main difference is that the truck handling work performed by the assistant in the NREL design will now be split amongst the millwrights and assistants.
- Work Assignments for Shift Workers: As mentioned in the job descriptions, it is assumed that support operators will assist with yard issues, feedstock delivery, and field work, while the superintendent will largely be responsible for control issues. This reduces the need for yard employees and operators whose sole job is to man control boards. The five crews effectively allow for additional personnel capable of supporting offloading and weighing of the biomass feedstock.
- *Subcontract Labor:* In order to reduce the need for full-time staff for part-time work, a number of specific skills, such as welders, electricians, and carpenters, will be

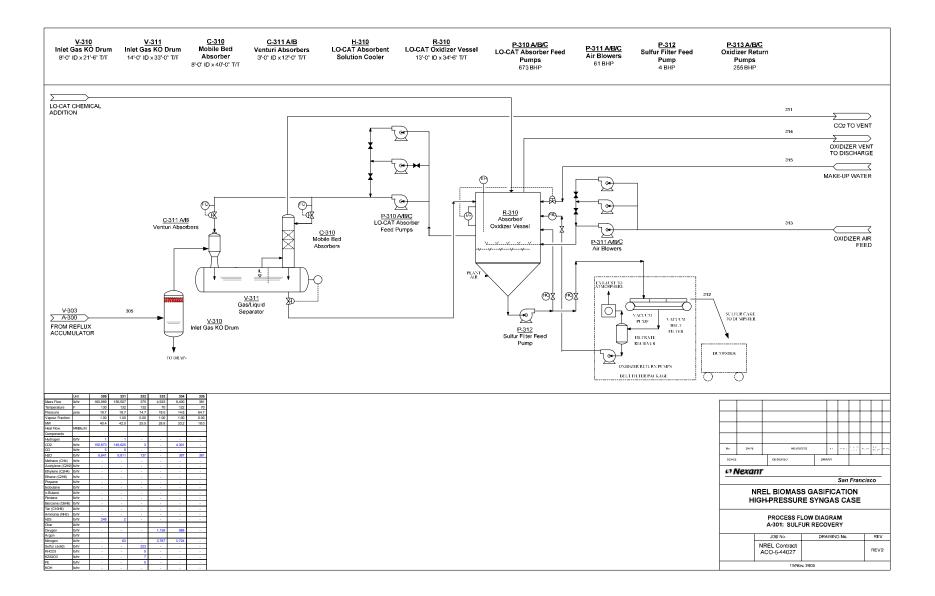
contracted out. This reduces the overall labor costs and overhead. No subcontract labor was assumed in the NREL hydrogen case.

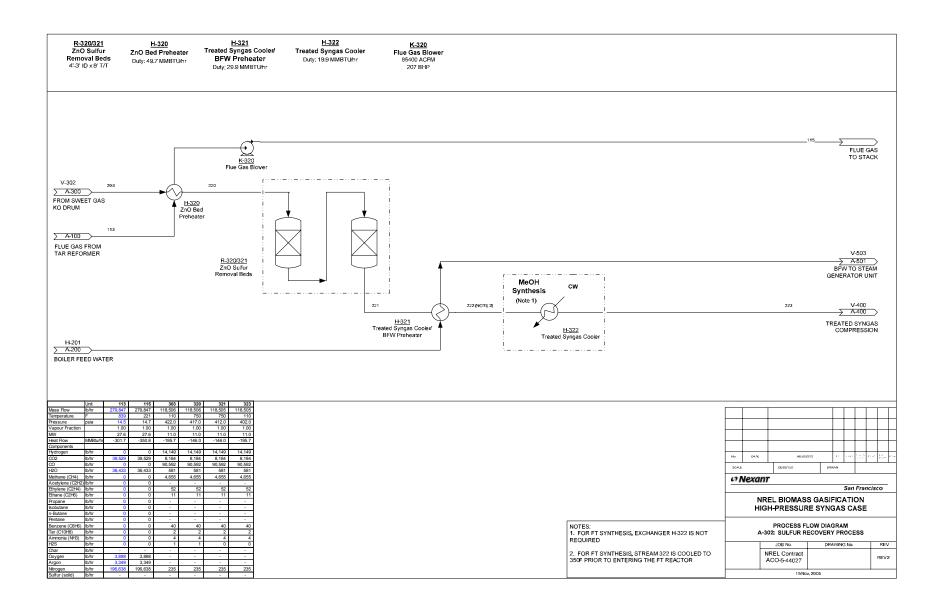
- **Overhead:** The labor estimate made in this case has roughly half as much full-time staff by utilizing more contract labor and changing the job description of day and shift employees. This is one reason that the estimate for overhead expenses (60%) is less than the biomass to hydrogen case (95%). In addition, the assumption has been made that a small firm will own and operate this facility. In general, overhead has been found to be less in smaller firms than in large multinationals; this assumption could be revised based on the ownership basis. This assumption for the overhead rate has been confirmed by Emery Energy, and is consistent with other small gasification companies that have limited facilities and indirect labor costs.
- **Overtime Assumptions:** The NREL hydrogen case assumed straight salaries for all employees, with no overtime. The chemicals case assumes ~2500 hours of overtime per year, roughly split over the 4 main shift worker categories. Allowing overtime reduces the number of full-time employees required, and decreases overall labor costs versus the NREL hydrogen case.
- Back-Up Shift Crew: Unlike the NREL hydrogen design, the back-up fifth shift team would be available to cover a number of different duties during the day shift, decreasing the need for specialty workers in each area.

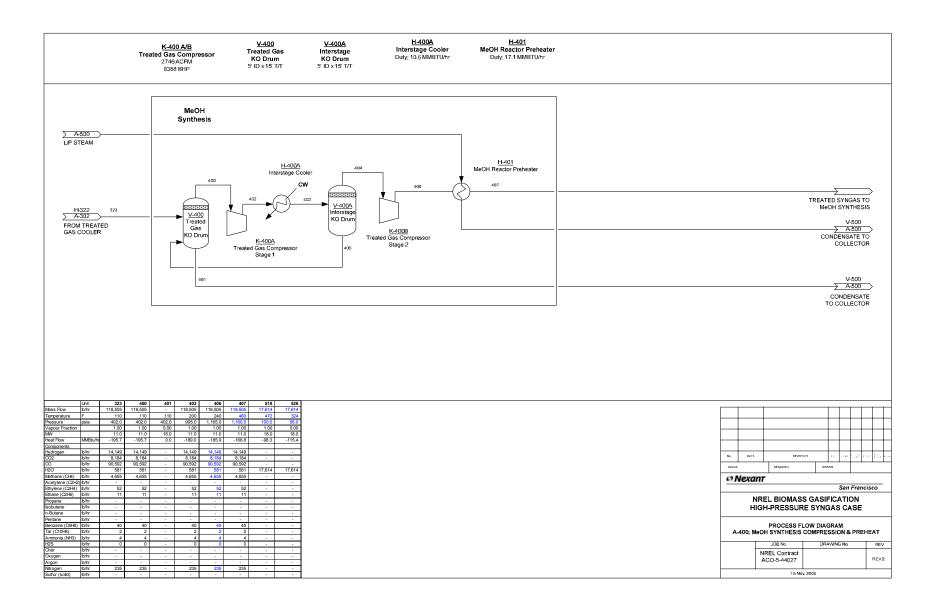


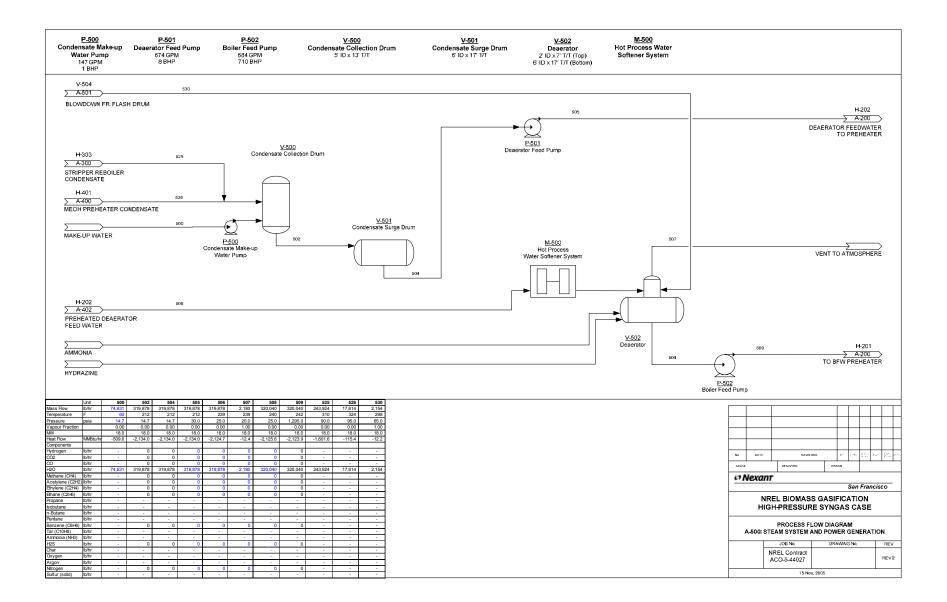




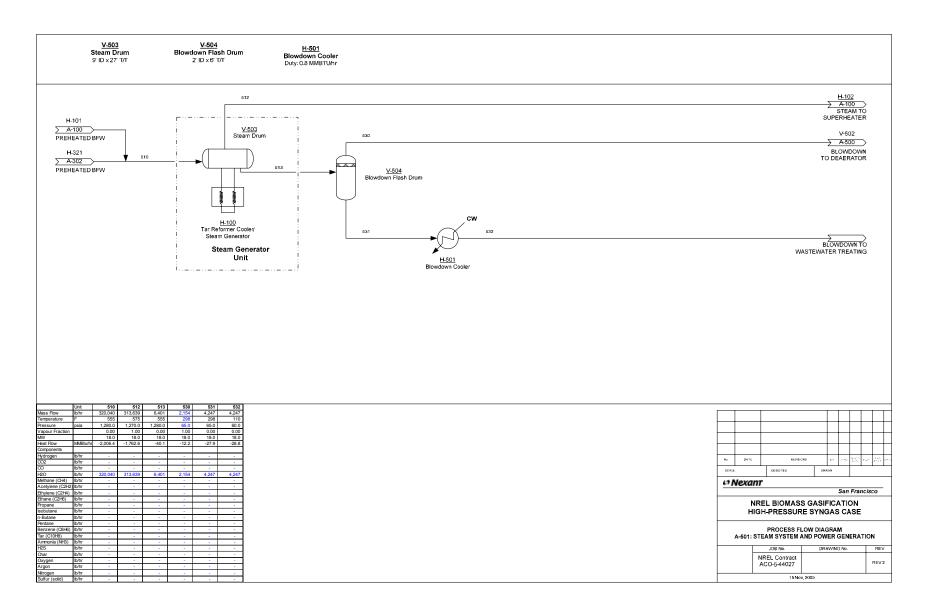


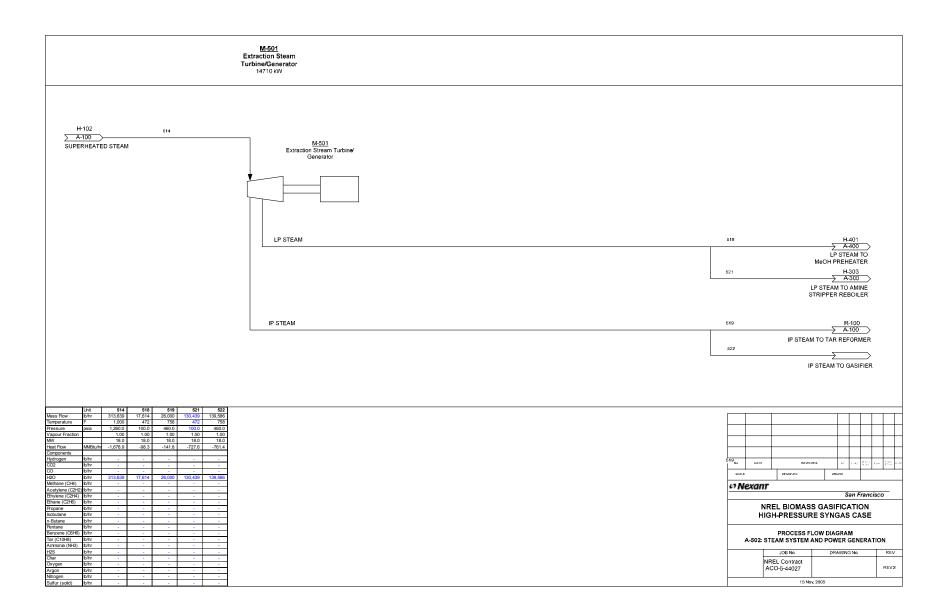


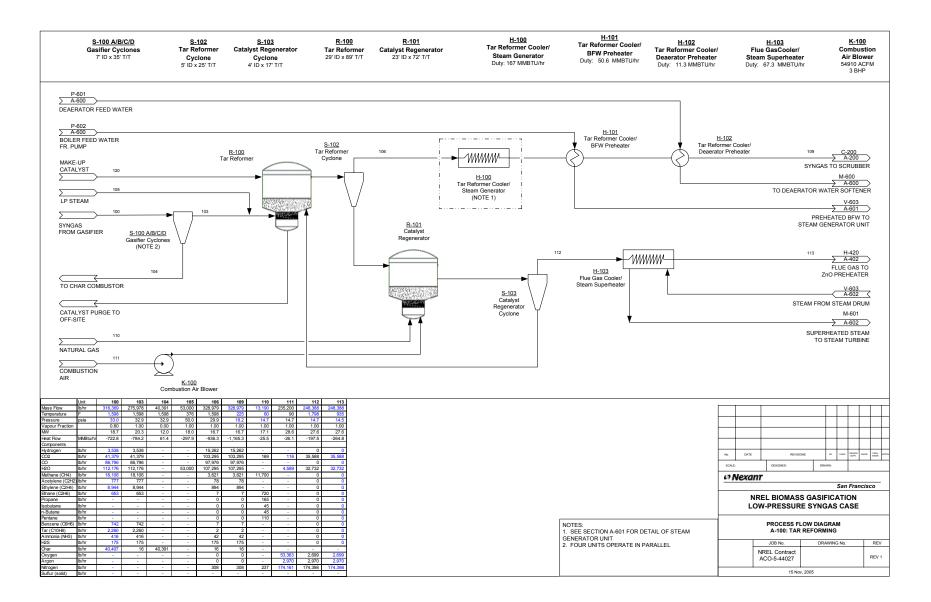


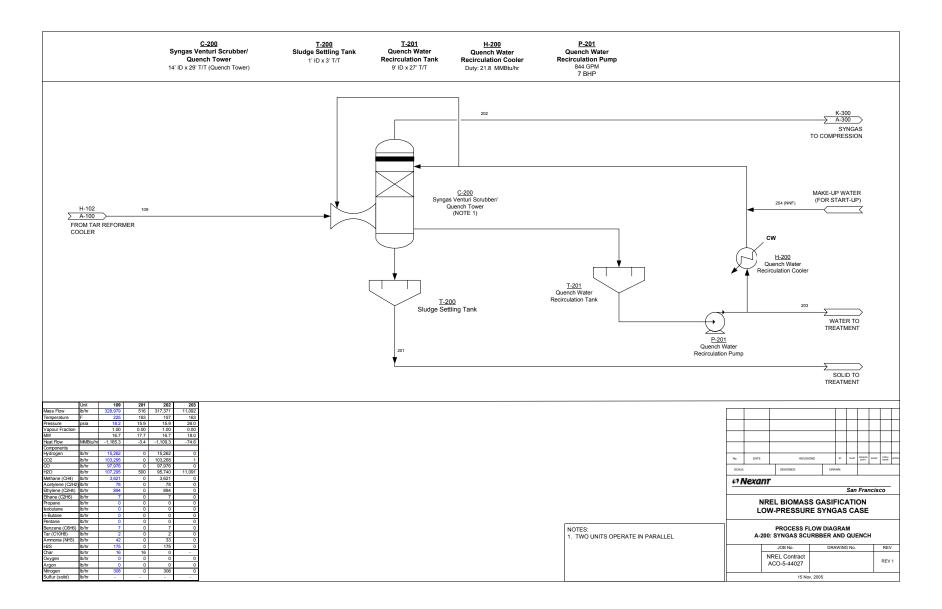


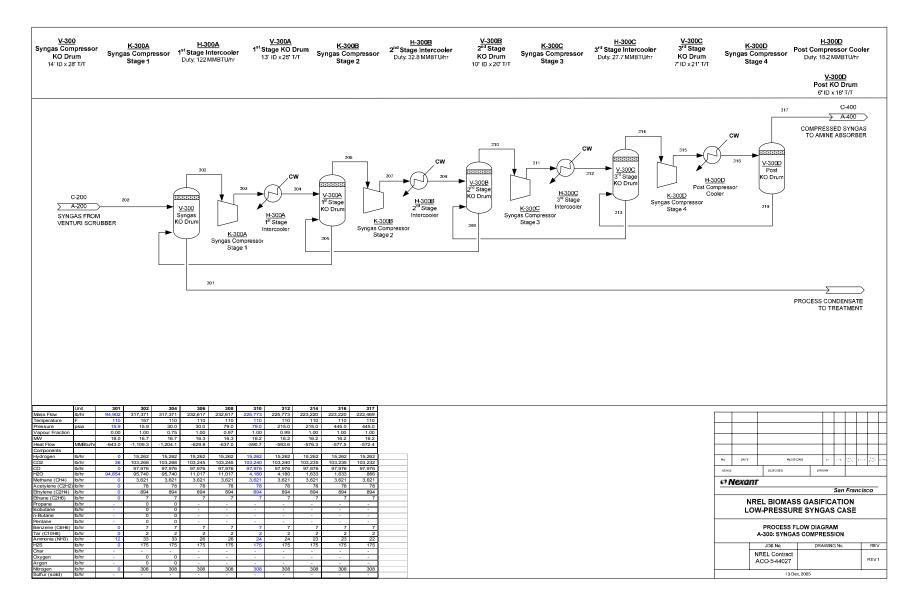


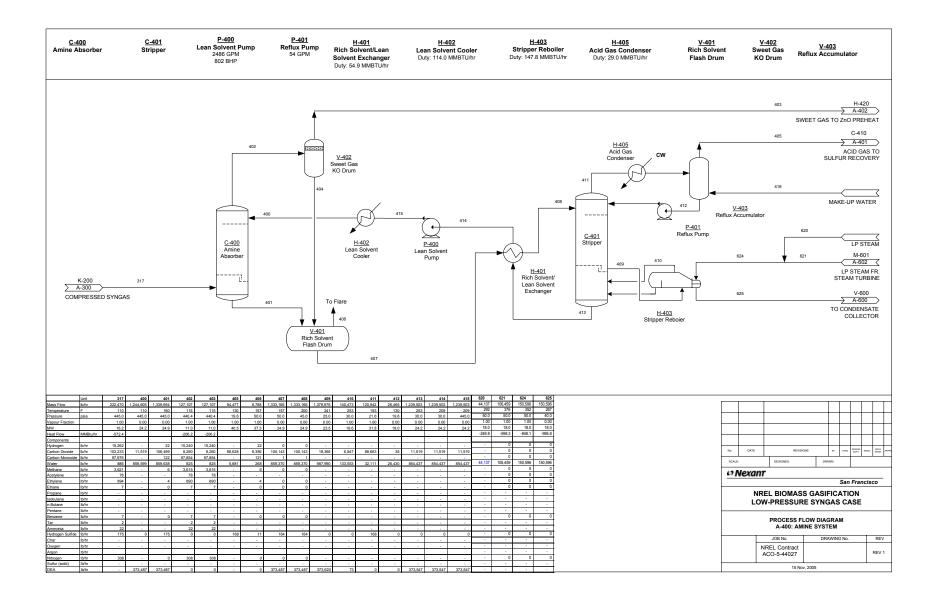


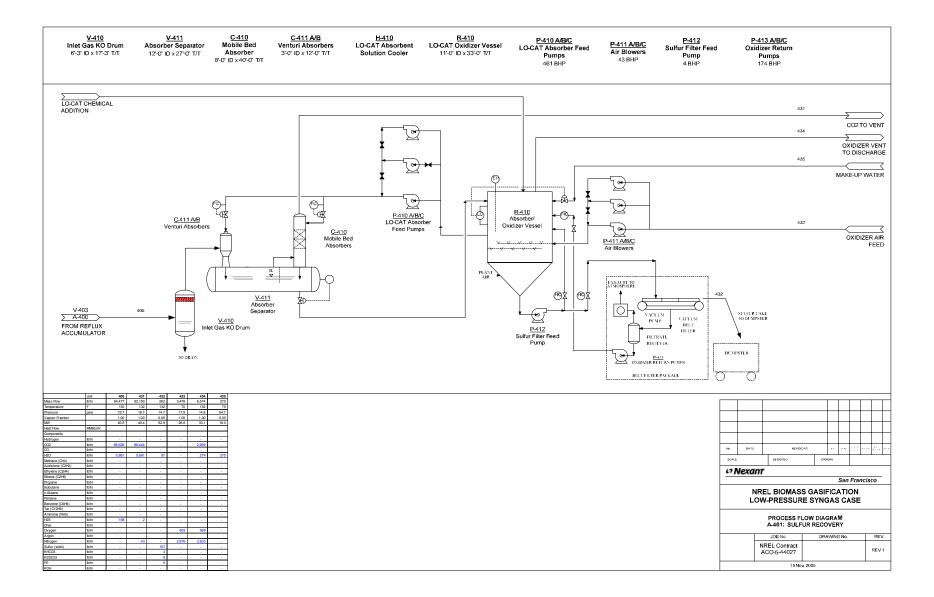


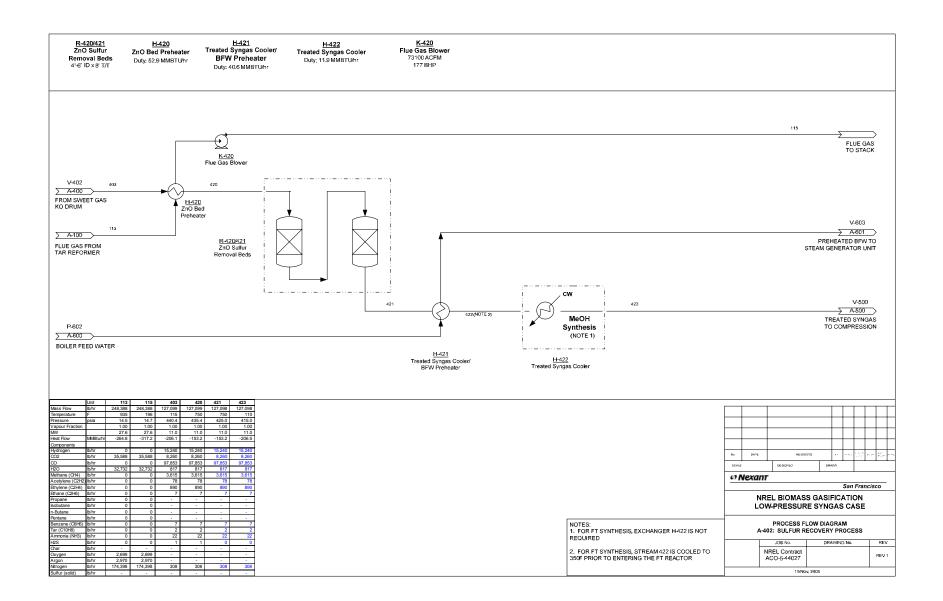


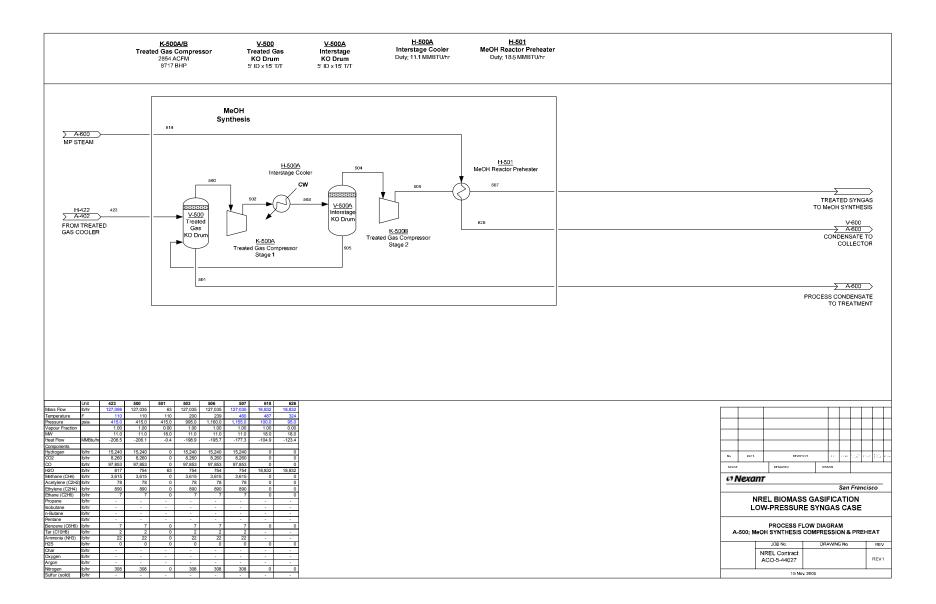


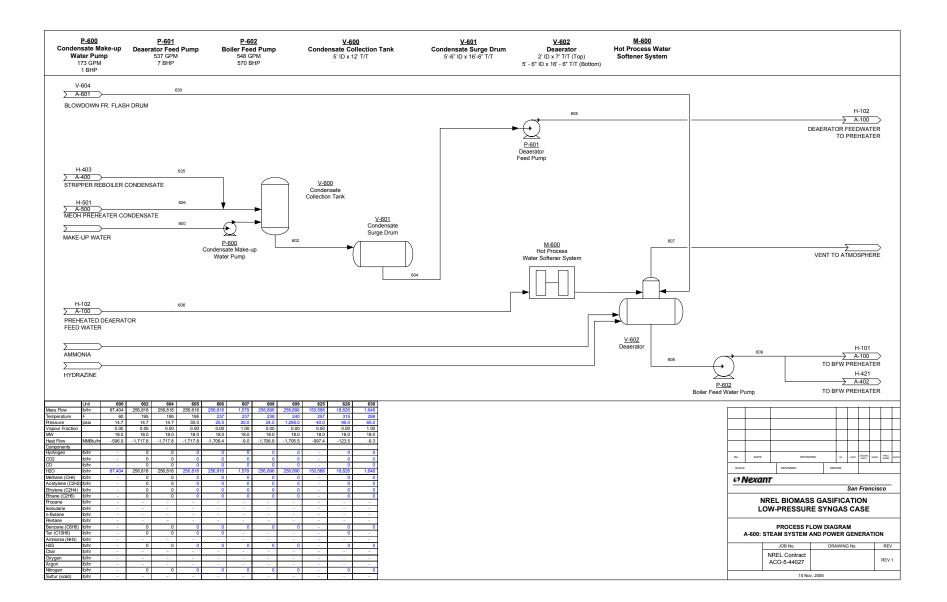




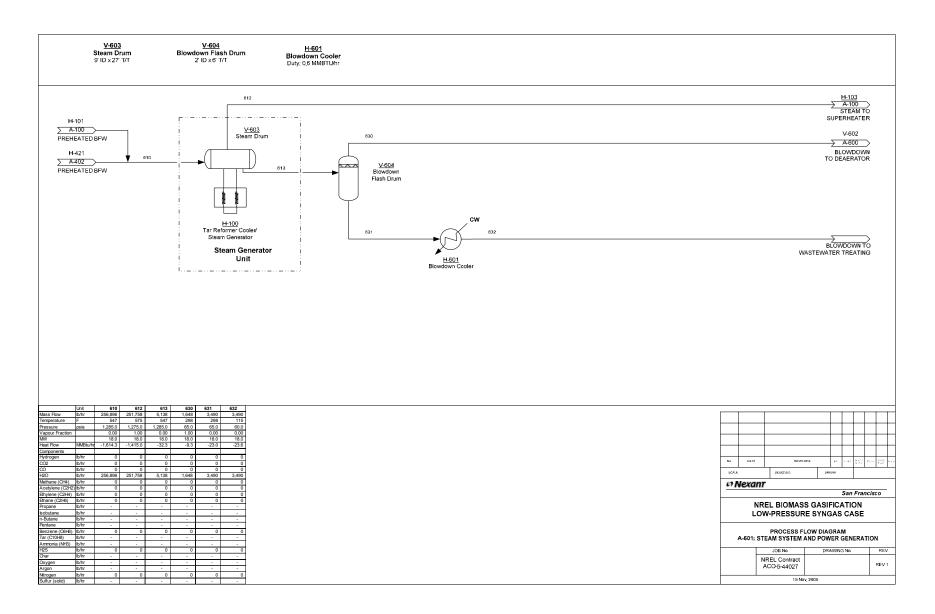


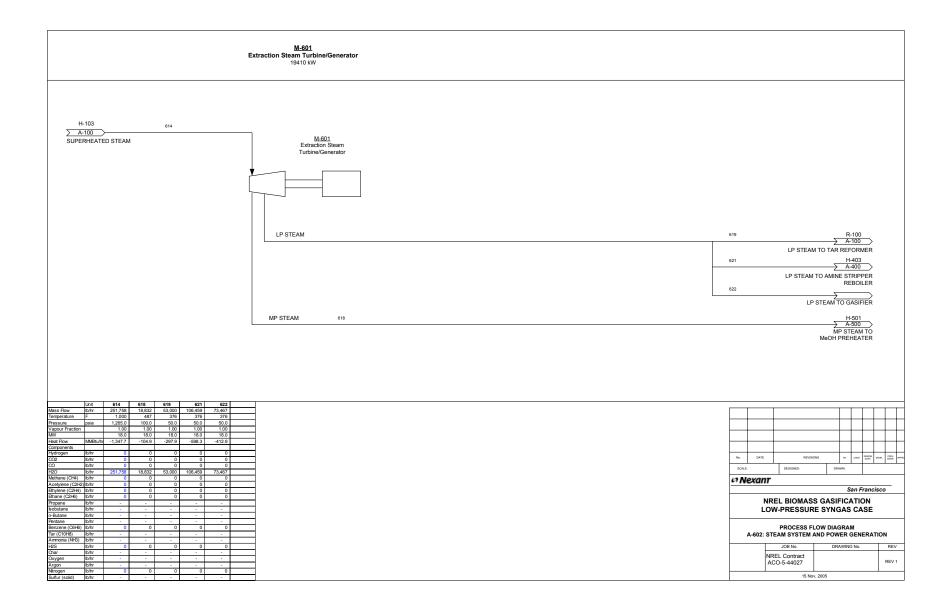












The following two appendices show the equipment lists for the high-pressure and low-pressure syngas design cases, along with detailed data sheets for some of the major pieces of equipment. No specific detail was developed for the tar cracking equipment due to the preliminary nature of its design. In addition, no additional information beyond what is presented in the equipment list was produced for vessels and pumps. Detailed equipment sheets are only shown for exchangers, cyclones, and compressors, where additional design data was developed.

								Desic		0.000	ating							
Item No	Description	Туре	Quantity Per Train	Size	Weight	Head	Design Duty	P	т	P	T	Power Useage	Materials	Price, total (unistalled)	Price Escalated, total (uninstalled)	Total Installed Cost	Quote Source	Comments
					lbs	PSI		PSIG	٩F	PSIG	۰F	(No.) HP		Q2 2004 Cost Index (US \$)	Q2 2005 Cost Index (US \$)	(US \$)		
Reactors													Refractory lined					
R-100	Tar Reformer	Fluidized Bed		20' ID x 20' T/T				490	1675	445	1576		CS		\$950,942		GTI	662,000 lbs catalyst req'd
R-101	Catalyst Regenerator			22' ID x 70' T/T				20	1950	-	1850		Refractory lined CS		\$329,616		GTI	
R-320	ZnO Beds	Vertical	1		43,856		2 ppmv H2S inlet	445	850	402	750		CS		\$219,280		Johnson Matthey	
R-321	ZnO Beds	Vertical	1	4' - 3* ID x 8' T/T	43,856		2 ppmv H2S inlet	445	850	402	750		CS		\$219,280		Johnson Matthey	707 ft3 total catalyst volume req'd
Total															\$1,719,118			
Cyclones																		
S-100	Gasifier Cyclone	Cyclone		5' ID x 25' T/T			3304 lb/hr dust loading	490	650	445	1576		CS w/ 4" refractory lining		\$355,000		Fisher Kosterman	Refractory lining will bring the shell temperature to 590F.
							1128 lb/hr dust						CS w/ 4" refractory					Refractory lining will bring the
S-102	Tar Reformer Cyclone	Cyclone	1	5' - 6" ID x 27' - 6" T/T			loading 1128 lb/hr dust	490	650	442	1576		lining CS w/ 4" refractory		\$410,000		Fisher Kosterman	shell temperature to 590F. Refractory lining will bring the
S-103	Catalyst Regenerator Cyclone	Cyclone	1	5' ID x 25' T/T			loading	490	650	442	1576		Lining		\$265,000		Fisher Kosterman	shell temperature to 590F.
Total															\$1,030,000			
Columns, V	essels & Tanks																	
				3" - 2" ID x 7" - 9" T-T (Venturi);														
0.000	Syngas Venturi Scrubber & Quench Tower	Vertical		5' - 3" ID x 8' - 11" T-T (Quench Tower)				485	420	432	370		CS		\$316.000		EPA Cost Curve	
C-200 V-400	Treated Gas KO Drum	Vertical	1	5' ID x 15' T-T	31700			400	420	432	110		CS	\$31 700	\$37,580		ICARUS	
V-400A	Interstage KO Drum	Vertical	1	5' ID x 15' T-T	29300			1030	250	980	200		CS	\$57,800	\$68,522		ICARUS	
V-500	Condensate Collection Drum	Vertical	1	5' ID x 13' T-T	4170			15	265	0	212		CS	\$14,745	\$17,480		ICARUS	
V-501	Condensate Surge Drum	Horizontal	1	6' ID x 17' T-T	6300			15	145	0	94		CS CS	\$22,195 \$31,350	\$26,312 \$37,165		ICARUS	
V-502	Deaerator	Horizontal	1	6' ID x 17' T-T; 2' ID x 7' T-T	7900			25	290	10	240						ICARUS	
	Steam Drum	Horizontal	1	9' ID x 27' T-T	139300			1335	625	1265	575		SA 302B	\$764,205	\$1,018,227		ICARUS	
V-504 T-200	Blowdown Flash Drum	Vertical Horizontal	1	2' ID x 6' T-T 1' ID x 3' T/T	1300 300			65 475	350 415	50 430	298 364		CS CS	\$8,200 \$4,800	\$9,721 \$5,690		ICARUS ICARUS	
T-200 T-201	Sludge Settling Tank Quench Water Recirculation Tank	Horizontal Horizontal	1	4' - 6" ID x 14' T/T	3600			475	415	430	364		CS	\$4,800 \$14,460	\$5,690		ICARUS	
Total	Querich water Redirculation Tank	Horizoniai		4-0 ID X 14 1/1	3000			4/5	360	430	311		63	\$14,400	\$1,553,840		ICARUS	
Used Freehouse																		
Heat Excha				5' - 7* ID x 12' T-T				T 1335	625	1270	575		CS					
H-100	Tar Reformer SG Cooler/Steam Generator	Shell & Tube	2	Surface area: 5206 SQFT			203.7 MMBTU/hr	S 485	1675	442	1576		CS - refractory	\$1,465,600	\$1,664,628		ICARUS	Refractory Lined
H-101	Tar Reformer SG Cooler/BFW Preheater	Shell & Tube	1	7' - 6" ID x 20' T-T Surface area: 23969 SQFT			50.84 MMBTU/hr	T 1335 S 485	600 675	1270 437	551 624		CS CS	\$513,500	\$583,233		ICARUS	
H-102	Flue Gas Cooler/Steam Superheater	Shell & Tube	1	8' - 4" ID x 14' T-T Surface area: 8915 SQFT			83.65 MMBTU/hr	T 1335 S 15	1100 1900	1255 0	1000 1798		316S CS - refractory	\$1,598,750	\$1,815,860		ICARUS	Refractory Lined
H-200	Quench Water Recirculation	Shell & Tube	1	3' - 6" ID x 10' T-T Surface area: 2867 SQFT			22.34 MMBTU/hr	T 485 S 20	415 150	441 5	364 100		CS CS	\$80,000	\$90,864		ICARUS	
				4' - 8" ID x 14' T-T				T 1335	400	1280	349		CS					
H-201	Amine Precooler/BFW Preheat	Shell & Tube	1	Surface area: 7511 SQFT 3' - 4" ID x 6' T-T			36.99 MMBTU/hr	S 470 T 30	410 300	427 15	356 239		CS CS	\$260,300	\$295,649		ICARUS	
H-202	Amine Precooler/Deaerator FW Preheat	Shell & Tube	1	Surface area: 585 SQFT 8' ID x 8' T-T			9.24 MMBTU/hr	S 465 T 65	400 150	422 50	338 100		CS CS	\$16,260	\$18,468		ICARUS	
H-203	Amine Precooler	Shell & Tube	1	Surface area: 11541 SQFT 8' ID x 8' T-T			139.3 MMBTU/hr	S 460 T 450	350 800	432	305 750		CS CS	\$309,600	\$351,644		ICARUS	
H-320	ZnO Preheater	Shell & Tube	1	Surface area: 19400 SQFT 5' ID x 16' T-T			49.69 MMBTU/hr	S 15 T 1335	910 615	0 1270	839		CS CS	\$288,000	\$327,110		ICARUS	-
H-321	ZnO SG Cooler/BFW Preheater	Shell & Tube	1	Surface area: 5440 SQFT			29.85 MMBTU/hr	S 440	800	397	565 750		CS	\$192,600	\$218,755		ICARUS	
H-322	Post ZnO Syngas Cooler	Shell & Tube	1	3' ID x 8' T-T Surface area: 1620 SQFT			19.91 MMBTU/hr	T 65 S 435	150 420	50 393	100 370		CS CS	\$56,100	\$63,718		ICARUS	
H-400A	MeOH Compressor Interstage Cooler	Shell & Tube	1	1' - 11" ID x 6' T-T Surface area: 476 SQFT			10.47 MMBTU/hr	T 1035 S 65	390 150	985 100	338 50		CS CS	\$32,200	\$36,573		ICARUS	
			<u> </u>	6' ID x 18' T-T				T 1210	515	1150	460		CS					
H-401	MeOH Syngas Preheat	Shell & Tube	1	Surface area: 16212 SQFT 1' - 3" ID x 4' T-T			17.14 MMBTU/hr	S 100 T 65	525 150	85	472		CS CS	\$355,140	\$403,368		ICARUS	
H-501 Total	Blowdown Cooler	Shell & Tube	1	Surface area: 130 SQFT			0.84 MMBTU/hr	S 65	350	50	298		ČŠ	\$19,100	\$21,694 \$5,891,565		ICARUS	
Total															\$5,691,565			
Compresso	rs & Blowers																011 01 -	
K-100	Combustion Air Blower	Blower	2	61910 ACFM		5				0	90	1800	CS		\$274,305		Chicago Blower Corp./ ICARUS	Used ICARUS to cost motor. 2 - 100% blowers
	Flue Gas Blower	Blower	2	85400 ACFM		0.4				0	214	207			\$233.875		Scaled fr. Chicago Blower	2 - 100% blowers
K-400	MeOH Compressor - 2 Stages	Centrifugal	1	2746 ACFM	74,500	758			i	387	110	8388	CS CS	\$2,133,200	\$2,522,936		ICARUS	
Total										1					\$3,031,115			
Pumps			1			•				1								
P-201	Quench Water Recirculation	Centrifugal	2	282 GPM	420	14		475	360	430	311	3	CS	\$10,600	\$11,021		ICARUS	2 - 100% pumps
P-500	Condensate Make-up Water Pump	Centrifugal	2	147 GPM	440	5		20	110	0	60	1.3	CS	\$5,400	\$5,614		ICARUS	2 - 100% pumps
P-501	Deaerator Feed Pump	Centrifugal	2	674 GPM	680	15		30	150	0	98	8	CS	\$17,200	\$17,883		ICARUS	2 - 100% pumps
P-502 Total	Boiler Feed Water Pump	Centrifugal	2	684 GPM	9,000	1,270		1345	290	11	240	710	CS	\$325,000	\$337,903 \$372,421		ICARUS	2 - 100% pumps
			1						t	1					\$3/2,421			
Steam Turb	ine																	
M-501	Steam Turbine	Steam Turbine	1		172,900	-1,160				1245	1000	(14710 kW)	CS	\$4,534,500	\$5,362,953		ICARUS	
Total			I							+					\$5,362,953			
Package Un										+			<u> </u>					
	Amine Unit	I	1	1 1						1						\$22,413,600	GRI Cost Curve	1
	LO-CAT Unit		1	1						1					\$3,998,550		Gas Technology Products	1
										1								
TOTAL EQU	JIPMENT COST, (excld. Package units)														\$18,961,012	\$48,729,802		Installation factor of 2.57 used
TOTAL INST	TALLED COST															\$76,491,952		
										-								

## HIGH PRESSURE SYNGAS DESIGN CASE

Task 2: Gas Cleanup Design and Cost Estimates, Wood Feedstock Final Report United States Department of Energy/National Renewable Energy Laboratory

C-2

								Des	lan	Oper	eting							
-		1	1	T	r –	r –		Des	lign	Oper	ating		r	Price, total	Price Escalated,	Total Installed		
Item No	Description	Type	Quantity	Size, each	Weight	Head	Design Duty, total	Р	т	Р	т	Power Useage	Materials	(unistalled)	total (uninstalled)	Cost	Quote Source	Comments
					lbs	PSI		PSIG	۰F	PSIG	٩F	(No.) HP		Q2 2004 Cost Index (US \$)	Q2 2005 Cost Index (US \$)	(US \$)		
Reactors					Ibs	PSI		PSIG	9°	PSIG	4	(No.) HP		(US \$)	(US\$)	(US\$)		
		Fluidized Bed	1										Refractory lined					
R-100	Tar Reformer	Fluidized Bed	1	29' ID x 89' T/T				30	1700	15	1598		CS		\$921,786		GTI	1,820,000 lbs catalyst reg'd
R-101	Catalyst Regenerator		1	23' ID x 72' T/T				30	1700	15	1598		Refractory lined CS		\$545.886		GTI	
	ZnO Beds	Vertical	1	4' - 6" ID x 8' T/T	44,522		2 ppmv H2S inlet	455	850	415	750		CS		\$222,610		Johnson Matthey	
	ZnO Beds	Vertical	1	4' - 6" ID x 8' T/T	44,522		2 ppmv H2S inlet	455	850	415	750		CS		\$222,610 \$1,912,892		Johnson Matthey	777 ft <sup>3</sup> total catalyst volume req'd
Total															\$1,912,892			
Cyclones																		
S-100							14,142 lb/hr dust		650 (see				CS w/ 4*					Refractory lining will bring the shell
A/B/C/D	Gasifier Cyclone	Cyclone	4	7" ID x 35" T/T			loading	33	comments)	18	1598		refractory lining		\$1,225,000		Fisher Kosterman	temperature to 590F.
S-102	Tar Reformer Cyclone	Cyclone	1	5' ID x 25' T/T			1,000 lb/hr dust loading	33	650 (see comments)	15	1598		CS w/ 4* refractory lining		\$370,000		Fisher Kosterman	Refractory lining will bring the shell temperature to 590F.
			<u> </u>				1,000 lb/hr dust	33	650 (see				CS w/ 4*					Refractory lining will bring the shell
S-103	Catalyst Regenerator Cyclone	Cyclone	1	4' ID x 17' T/T			loading	33	comments)	15	1598		refractory lining		\$250,000		Fisher Kosterman	temperature to 590F.
Total															\$1,845,000			
Columns, V	essels & Tanks																	
C-200	Syngas Venturi Scrubber & Quench Tower	Vertical	2	14' ID x 29' T/T				19	275	4	225		CS		\$340,000		Croll Reynolds	
V-300 V-300A	Syngas KO Drum	Vertical	2	14' ID x 28' T/T 13' ID x 26' T/T	31,500 25,500			16 30	210 160	1	157 110		CS CS	\$306,800 \$73,400	\$363,711 \$87,016		ICARUS	
	1st Stage KO Drum 2nd Stage KO Drum	Vertical	1	10' ID x 20' T/T	25,500			79	160	64	110		CS	\$73,400	\$64,373		ICARUS	
V-300C	3rd Stage KO Drum	Vertical	1	7" ID x 21' T/T	21,900			220	160	200	110		CS	\$41,800	\$49,554		ICARUS	
V-300D	Post KO Drum	Vertical	1	6' ID x 18' T/T	23,600			475	160	430	110		CS	\$45,400	\$53,822		ICARUS	
V-500 V-500A	Treated Gas KO Drum Interstage KO Drum	Vertical Vertical	1	5' ID x 15' T/T 5' ID x 15' T/T	14,900 29,300			440	160 250	400 980	110 200		CS CS	\$31,800 \$57,800	\$37,699 \$68,522		ICARUS ICARUS	
V-600	Condensate Collection Tank	Vertical	1	5' ID x 12' T/T	3,990			1,030	250	0	195		CS	\$14,100	\$16,716		ICARUS	
V-601	Condensate Surge Drum	Horizontal	1	5' - 6" ID x 16' - 6" T/T	5,483			15 25	245	Ö	195		CS	\$19,320	\$22,904		ICARUS	
	Deserator	Vertical	1	6" ID x 18" T/T; 2" ID x 6" T/T	7,800				290	10	237		CS	\$35,700	\$42,322		ICARUS	
V-603 V-604	Steam Drum Blowdown Flash Drum	Horizontal Vertical	1	9' ID x 27' T/T 2' ID x 6' T/T	139,300			1335 65	625 350	1270 50	575 298		SA 302B CS	\$764,205 \$7.500	\$1,018,227 \$8,891		ICARUS	1
T-200	Sludge Settling Tank	Horizontal	1	1' ID x 3' T/T	300			16	180	1	128		CS	\$4,000	\$4,742		ICARUS	
T-201 Total	Quench Water Recirculation Tank	Horizontal	1	9' ID x 27' T/T	15,300			16	180	1	128		CS	\$60,700	\$71,960 \$2,250,458		ICARUS	
Total															\$2,250,458			
Heat Excha	ngers																	
				6' ID x 14' T/T				T 1335	625	1270	575		CS	1				
H-100	Tar Reformer SG Cooler/Steam Generator	Shell & Tube	2	Surface area: 5354 SQFT 4' - 9" ID x 14' T/T			167 MMBTU/hr	S 30 T 1335	1700 600	15	1598 542		CS - refractory	\$989,400	\$1,129,202		ICARUS	Refractory Lined
H-101	Tar Reformer SG Cooler/BFW Preheater	Shell & Tube	2	4 - 9 ID x 14 1/1 Surface area: 6667 SQFT			50.61 MMBTU/hr	S 20	675	1,280 12	624		CS CS	\$682,550	\$775,240		ICARUS	
			-	6' - 3" ID x 14' T/T				T 30	280	15	227		CS					
H-102	Tar Reformer Cooler/Deaerator FW Preheat	Shell & Tube	1	Surface area: 5621 SQFT			11.34 MMBTU/hr	S 20	350	9	300		CS	\$104,600	\$118,805		ICARUS	
H-103	Flue Gas Cooler/Steam Superheater	Shell & Tube	1	7' - 6" ID x 14' T/T Surface area: 5770 SQFT			67.26 MMBTU/hr	T 1335 S 15	1100	985	1275 1798		316S CS - refractory	\$1.016.858	\$1,154,947		ICARUS	Refractory Lined
				5" - 11" ID x 10' T/T				T 30	150	5	100		CS					renderory enice
H-200	Quench Water Recirculation Cooler	Shell & Tube	1	Surface area: 9232 SQFT			22.2 MMBTU/hr	S 30	215	11	161		CS	\$203,800	\$231,476		ICARUS	
H-300A	Compressor Interstage Cooling	Shell & Tube		6' - 10" ID x 12' T/T Surface area: 14235 SQFT			122 MMBTU/hr	T 35 S 65	400 150	20 50	344 100		CS CS	\$802,600	\$911,593		ICARUS	
H-300A	Compressor interstage Cooling	Silei a Tube	-	3' - 11" ID x 10' T/T			122 MMD10/11	T 65	150	50	100		CS	\$802,000	3911,053		IGAROS	
H-300B	Compressor Interstage Cooling	Shell & Tube	1	Surface area: 3435 SQFT			32.79 MMBTU/hr	S 85	400	69	350		CS	\$72,300	\$82,118		ICARUS	
H-300C	Compressor Interstage Cooling	Shell & Tube	1	4' - 3" ID x 10' T/T			27.69 MMBTU/br	T 230 S 65	400	205	349 100		CS	\$95,000	\$107,901		ICARUS	
H-300C	Compressor Interstage Cooling	Shell & Tube	1	Surface area: 4368 SQFT 3' - 6" ID x 10' T/T			27.69 MMB1U/hr	S 65 T 485	330	435	277		CS	\$95,000	\$107,901		ICARUS	
H-300D	Compressor Interstage Cooling	Shell & Tube	1	Surface area: 2934 SQFT			18.21 MMBTU/hr	S 65	150	50	100		CS	\$74,900	\$85,071		ICARUS	
				7' - 6" ID x 8' T/T				T 465	800	420	750		CS					
H-420	ZnO Preheater	Shell & Tube	1	Surface area: 14480 SQFT 5' - 4" ID x 12' T/T			52.90 MMBTU/hr	S 15 T 1335	990 600	0	945 542		CS CS	\$289,300	\$328,587		ICARUS	
H-421	ZnO Syngas Cooler/BFW Preheat	Shell & Tube	1	Surface area: 6915 SQFT			40.57 MMBTU/hr	S 455	800	410	750		CS	\$244.300	\$277,476		ICARUS	
				2' - 6" ID x 8' T/T				T 65	150	50	100		CS					
H-422	ZnO Syngas Cooler	Shell & Tube	1	Surface area: 1190 SQFT 2' - 6" ID x 8' T/T			11.86 MMBTU/hr	S 450 T 1.035	315	405	265		CS	\$41,210	\$46,806		ICARUS	
H-500A	MeOH Compressor Interstage Cooling	Shell & Tube	1	Surface area: 511 SQFT			11.06 MMBTU/br	S 65	150	50	100		CS	\$33,800	\$38,390		ICARUS	
				6' ID x 14' T/T				T 1,261	515	1,145	460		CS					
H-501	MeOH Syngas Preheat	Shell & Tube	1	Surface area: 12712 SQFT			18.45 MMBTU/hr	S 100	540	85	487		CS	\$278,500	\$316,320		ICARUS	
H-601	Blowdown Cooler	Shell & Tube	1	1' ID x 4' T/T Surface area: 89 SQFT			0.609 MMBTU/hr	T 65 S 65	150 350	50 50	100 298		CS CS	\$18,400	\$20,899		ICARUS	
Total	, 100 (100 - 100 -														\$5,624,833		100.0100	
		1	1							I –								
Compresso	6	1	1	+					-				l	+			Chicago Blower Corp./	Used ICARUS to cost motor. 2 -
K-100	Combustion Air Blower	Blower	2	54910 ACFM		3				0	90	1600	CS		\$256,425		ICARUS	100% blowers
K-420	Flue Gas Blower	Blower	2	73100 ACFM		0.4				0	176	177	CS		\$202,375		Scaled fr. Chicago Blower	2 - 100% blowers
K-300 K-500	Syngas Compressor- 4 stages MeOH Compressor- 2 stages	Centrifugal Centrifugal	1	131800 ACFM 2854 ACFM	333,100 31,100	434 745				1 399	157 115	38,786 8,717	CS CS		\$15,000,000 \$2,369,000	\$37,050,000	Elliott Ariel Corp.	2.47 installation factor
Total		Genunuyar		2004 AGEM	31,100	740				399	110	0,/1/	03	L	\$17,827,800		Aller Corp.	
Pumps P-201	Quench Recirculation Pump	Centrifugal	1	2422 CDM	900	10.10		26	211	1.18	160.9	20	C.8	\$91.000	804.812		ICADUS	2 100% ритер
P-600	Condensate Make-up Water Pump	Centrifugal	2	2423 GPM 172 GPM	800 440	10.12		20	211	0	60	20	CS CS	\$6.320	\$94,613 \$6,571		ICARUS ICARUS	2 - 100% pumps 2 - 100% pumps
P-601	Deaerator Feed Pump	Centrifugal	2	172 GPM 537 GPM	810	15.3		30	160	15	108.8	7	CS	\$17,400	\$18,091		ICARUS	2 - 100% pumps
P-602 Total	Boiler Feedwater Pump	Centrifugal	2	548 GPM	8,900	1275		1350	278	20	227.9	570	CS	\$316,800	\$329,377 \$448,651		ICARUS	2 - 100% pumps
rotai		1		+	-										\$448,651			
Steam Turb							<u> </u>											
	Steam Turbine- 2 extraction stages	Steam Turbine	1		221,200	-1200				1,250	1,000	(19410 kW)	CS	\$5,459,900	\$6,457,424		ICARUS	
Total				+									-	+	\$6,457,424			
Package Ur	its	1	1	1			1			I			1	1		1		1
A-400	Amine Unit															\$12,452,000	GRI Cost Curve	
A-401	LO-CAT Unit		<u> </u>	+						<u> </u>				+	\$3,733,550	\$5,003,550	Gas Technology Products	
		1	1	1						-				1				
1		1	1							I								Installation factor of 2.57 used on all
	JIPMENT COST, (excld. Package units)	I								L					\$36,367,057	\$91,963,336		equipment except syngas compressor
TOTAL INS	TALLED COST						<u> </u>									\$109,418,886		
												-						

## LOW PRESSURE SYNGAS DESIGN CASE

Task 2: Gas Cleanup Design and Cost Estimates, Wood Feedstock Final Report United States Department of Energy/National Renewable Energy Laboratory C-3

## DATA SHEETS, HIGH PRESSURE DESIGN

Job No.           Customer         NREL         Ref No.         HP Syngas Case           Address         Proposal No.         Date         Rev. 0           Plant Location         Date         Rev. 0         Service of Unit         Tar Reformer SG Cooler/HP Steam Generator         Item No.         H-100           Size 67x 144         Type         BEM - HORZ Connected in         2 Parallel         1 Series           SurfUnit (Eff)         10411 ft <sup>a</sup> Shells/Unit 2         Surface/Shell (Effective)         5206 ft <sup>a</sup> Fluid Allocation         PERFORMANCE OF ONE UNIT         Probaside         Fluid Name         Probaside           Fluid Name         Syngas fr Tar Reformer         Preheated BFW         Total Fluid Entering         10/1           Vapor         435.000         0         0         1313.900           Uquid Density (In/Out)         Ib/ft <sup>a</sup> 0.000/0.000         46.162/45.419           Liquid Density (In/Out)         Ib/ft <sup>b</sup> 0.0000         0.313.900           Liquid Density (In/Out)         Btu/h-ft-F         0.000         0.320           Vapor Moleght (In/Out)         Btu/h-ft-F         0.000         0.320           Vapor Meight (In/Out)         Btu/h-ft-F         0.0285         0.0216.25		h	leat Exchanger Spec	fication shee	et	
Address         Proposal No.           Plant Location         Date         Rev. 0           Service of Unit         Tar Reformer SG Cooler/HP Steam Generator         Item No         H-100           Size 67x 144         Type         BEM - HORZ Connected Item No         2 Parallel         1 Series           Suff/Unit (Eff)         10411 ft <sup>a</sup> Shells/Unit 2         Sufface/Shell (Effective)         5206 ft <sup>a</sup> PERFORMANCE OF ONE UNIT         Fluid Allocation         Shellside         Tubeside         Tubeside           Fluid Name         Syngas fr Tar Reformer         Preheated BFW         313,900         0         1313,900           Vapor         435,000         0         313,900         143,1900         143,1900           Vapor         435,000         0         313,900         146,162,45,419         149           Liquid Density (In/Out)         Ib/ft <sup>a</sup> 0.000         0.061         164,445,419           Liquid Density (In/Out)         Ib/ft <sup>a</sup> 0.000         0.031         164,41           Liquid Density (In/Out)         CP         0.000         0.041         164,45,419           Liquid Density (In/Out)         Ib/ft-F         0.000         0.0320         Vapor Moleight (In/Out)         18,661,866 <t< th=""><th></th><th></th><th></th><th></th><th></th><th></th></t<>						
Plant Location         Date         Rev. 0           Service of Unit         Tar Reformer SG Cooler/HP Steam Generator         Item No         H-100           Size 67x 144         Type         BEM - HORZ Connected in         2 Parallel         1 Series           StarfUnit (Eff)         10411 ft <sup>2</sup> Shells/Unit         2 Surface/Shell (Effective)         5206 ft <sup>2</sup> Fluid Allocation         PERFORMANCE OF ONE UNIT         Tubeside         Tubeside           Fluid Allocation         Syngas fr Tar Reformer         Preheated BFW           Total Fluid Entering         Ib/hr         435,000         0           Vapor         435,000         0         313,900           Vapor         435,000         0         313,900           Steam         0         313,900         1444           Uquid Density (In/Out)         Ib/ft <sup>1</sup> 0.000         0.091           Liquid Density (In/Out)         Ib/ft <sup>1</sup> 0.000         0.320           Vapor Viscosity         cP         0.000         0.320           Vapor Specific Heat         Btu/b-F         0.000         0.320           Vapor Specific Heat         Btu/b-F         0.067         0.025           Temperature (In/Out)         cP         1		EL			HP Syngas C	ase
Service of Unit         Tar Reformer SG Cooler/HP Steam Generator         Item No         H-100           Size 67x 144         Type         BEM - HORZ Connected in         2 Parallel         1 Series           SurfUnit (Eff)         10411 ft <sup>a</sup> Shells/Unit         2         Surface/Shell (Effective)         5206 ft <sup>a</sup> Fluid Allocation         PERFORMANCE OF ONE UNIT         Preheated BFW         Tubeside         Tubeside           Fluid Name         Syngas fr Tar Reformer         Preheated BFW         313,900         0           Vapor         435,000         0         0         13,900           Vapor         435,000         0         0         313,900           Steam         0         313,900         0         0         13,900           Liquid Density (In/Out)         Ib/ft <sup>a</sup> 0.000/0.000         4.616245.419         0           Liquid Specific Heat         Btu/b-F         0.000         1.644         1           Liquid Viscosity         CP         0.0025         0.0200         0.025           Temperature (In/Out)         Btu/b-F         0.492         0.774         0.025           Vapor Viscosity         CP         0.025         0.0200         1.285.000         0.025						
Size 67x 144         Type         BEM - HORZ Connected in         2 Parallel         1 Series           Surf/Unit (Eff)         10411 ft <sup>2</sup> Shells/Unit         2         Surface/Shell (Effective)         5206 ft <sup>2</sup> Fluid Allocation         Shells/Ide         Tubeside         Tubeside           Fluid Allocation         Shells/Ide         Tubeside         Tubeside           Fluid Name         Syngas ft Tar Reformer         Preheated BFW         Preheated BFW           Vapor         435,000         0         313,900           Liquid         0         313,900         Steam         0         313,900           Noncondensable         0         313,900         Liquid Specific Heat         Btu/b-F         0.000         0.091           Liquid Specific Heat         Btu/b-F         0.000         0.320         Vapor Noi Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200         Vapor Specific Heat         Btu/b-F         0.482         0.774           Vapor Viscosity         cP         0.0285         0.0200         Vapor Specific Heat         Btu/b-F         0.482         0.774           Vapor Viscosity         cP         0.0285         0.0000						Rev. 0
Surf/Unit (Eff)         10411 ft²         Shells/Unit         2         Surface/Shell (Effective)         5206 ft²           PERFORMANCE OF ONE UNIT         Pereheated BFW           Fluid Allocation         Shellside         Tubeside           Fluid Name         Syngas fr Tar Reformer         Preheated BFW           Otal Fluid Entering         Ib/hr         435,000         313,900           Vapor         435,000         0         313,900           Liquid Density (In/Out)         Ib/ht?         0.000/0.000         46.162/45.419           Liquid Density (In/Out)         Ib/ht?         0.0000         0.0311           Liquid Viscosity         CP         0.000         0.0311           Liquid Viscosity         CP         0.000         0.0311           Liquid Density (In/Out)         Ib/ht?         0.0000         0.0311           Uiguid Specific Heat         Bt//ht-Ft         0.000         0.320           Vapor Mol. Weight (In/Out)         Bt//ht-Ft         0.492         0.774           Vapor Mol. Weight (In/Out)         Ft/sec         4.35.04         8.337           Vapor Mol. Weight (In/Out)         Ft/sec         4.35.04         8.337           Vapor Mol. Weight (In/Out)         Ft         1.576.0624.0						
PERFORMANCE OF ONE UNIT           Fluid Allocation         Shellside         Tubeside           Fluid Allocation         Syngas IF Tar Reformer         Preheated BFW           Total Fluid Entering         Ib/hr         435,000         313,900           Vapor         435,000         0           Liquid         0         313,900           Steam         0         313,900           Noncondensable         0         313,900           Liquid Density (In/Out)         Ib/ht <sup>P</sup> 0.000         0.091           Liquid Specific Heat         Btu/ht-F         0.000         0.091           Liquid Specific Heat         Btu/ht-F         0.000         0.320           Vapor Microstity         CP         0.000         0.320           Vapor Specific Heat         Btu/ht-F         0.000         0.320           Vapor Specific Heat         Btu/ht-F         0.067         0.025           Vapor Specific Heat         Btu/ht-F         0.067         0.025           Temperature (In/Out)         ft/s62,003/3245         5.000/133         F           Operating Pressure Drop (Allow/Calc)         psi         5.000/3245         5.000/133           Presure Drop Allow/Calc)         psi						1 Series
Fluid Allocation         Shellside         Tubeside           Fluid Name         Syngas fr Tar Reformer         Preheated BFW           Total Fluid Entering         Ib/hr         435,000         313,900           Vapor         435,000         0         313,900           Vapor         435,000         0         313,900           Steam         0         313,900         313,900           Noncondensable         0         313,900         4435,000         0           Fluid Vaporized or Condensed         0         313,900         4435,000         0.000           Liquid Density (In/Out)         Ib/ftP         0.0000         46,162/45,419         1444           Liquid Specific Heat         Bt//lb-F         0.000         0.320         990           Vapor Mound Weight (In/Out)         Bt//lb-F         0.0285         0.0200         0.774           Vapor Specific Heat         Bt//lb-F         0.492         0.774         90.025           Temperature (In/Out)         rF         1,576.0624.0         556.0/575.0         0.025           Vapor Thermal Conductivity         Bt//lb-F         0.492         0.774         90.00         0.0255         0.0001.00         0.00500         1.285.000         Velocity	Surf/Unit (Eff) 104	11 ft <sup>2</sup> Shells/Unit		( )	5206 ft <sup>2</sup>	
Fluid Name         Syngas fr Tar Reformer         Preheated BFW           Total Fluid Entering         lb/hr         435,000         313,900           Vapor         435,000         0           Liquid         0         313,900           Steam         0         313,900           Noncondensable         0         313,900           Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         lb/ft*         0.0000         46.162/45.419           Liquid Specific Heat         Btu/hr-Ft         0.000         0.0291           Liquid Specific Heat         Btu/hr-Ft         0.000         0.320           Vapor Thermal Conductivity         Btu/hr-Ft         0.0285         0.0200           Vapor Specific Heat         Btu/hr-Ft         0.067         0.025           Vapor Specific Heat         Btu/hr-Ft         0.067         0.025           Temperature (In/Out)         FF         1,576.0/624.0         556.0/575.0           Operating Pressure         psi (Abs)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Fresure Drop (Allow/Calc)         psi         5.000/1.3360         0.005000			PERFORMANCE OF ONE	UNIT		
Total Fluid Entering         Ib/hr         435,000         313,900           Vapor         435,000         0           Liquid         0         313,900           Steam         0         313,900           Noncondensable         0         313,900           Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         Ib/ft <sup>P</sup> 0.0000         46.162/45.419           Liquid Density (In/Out)         Ib/ft <sup>P</sup> 0.0000         0.091           Liquid Specific Heat         Btu/lb-F         0.000         0.320           Vapor Mol. Weight (In/Out)         Btu/hr-ft-F         0.000         0.320           Vapor Viscosity         CP         0.0285         0.0200           Vapor Viscosity         CP         0.0285         0.0200           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         "F         1.576.0/624.0         556.0/575.0           Operating Pressure         psi(Abos)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133	Fluid Allocation		Shellside			Tubeside
Vapor         435,000         0           Liquid         0         313,900           Steam         0         313,900           Noncondensable         0         313,900           Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         Ib/ft <sup>4</sup> 0.000/0.000         46.162/45.419           Liquid Specific Heat         Bt/lb-F         0.000         0.320           Vapor Mol. Weight (In/Out)         Bt/lb-F         0.000         0.320           Vapor Specific Heat         Bt/lb-F         0.0285         0.0200           Vapor Specific Heat         Bt/lb-F         0.492         0.774           Vapor Specific Heat         Bt/lb-F         0.492         0.774           Vapor Thermal Conductivity         Bt/lb-F         0.492         0.774           Vapor Thermal Conductivity         Bt/lb-F         0.492         0.774           Vapor Thermal Conductivity         Bt/lb-F         0.0265         0.0200           Vapor Thermal Conductivity         Bt/lb-F         0.925         0.0200           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.0007.1323         5.0007.1333	Fluid Name		Syngas fr Tar Re	former		Preheated BFW
Liquid         0         313,900           Steam         0         313,900           Noncondensable         0         313,900           Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         ib/ft*         0.000/0.000         46,162/45,419           Liquid Vaporized or Condensed         0         0.091         0.091           Liquid Vaporized or Condensed         0         0.091         1644           Liquid Vaporized or Conductivity         Btu/hz-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18,86/18,86         0.0/18.02         0.320           Vapor Mol. Weight (In/Out)         18,86/18,86         0.0/18.02         0.774           Vapor Molemal Conductivity         Btu/hz-F         0.492         0.774           Vapor Thermal Conductivity         Btu/hz-F         0.492         0.774           Vapor Thermal Conductivity         Btu/hz-F         0.067         0.025           Temperature (In/Out)         *F         1,576.0/624.0         5.600/575.0           Operating Pressure         psi(Abs)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Fouling resistance	Total Fluid Entering	lb/hr	435,000			313,900
Steam         Noncondensable           Noncondensable         0           Fluid Vaporized or Condensed         0           Liquid Density (In/Out)         Ib/ft <sup>®</sup> Liquid Density (In/Out)         Ib/ft <sup>®</sup> Liquid Specific Heat         Btu/hr-F           Liquid Specific Heat         Btu/hr-F           Liquid Thermal Conductivity         Btu/hr-F           Vapor Mol. Weight (In/Out)         18.66/18.66           Vapor Specific Heat         Btu/hr-F           Vapor Specific Heat         Btu/hr-F           Vapor Specific Heat         Btu/hr-F           O.0285         0.0200           Vapor Thermal Conductivity         Btu/hr-F           Outhor F         0.0267           Temperature (In/Out)         °F           Operating Pressure         psi(Abs)           Operating Pressure         psi(Abs)           Velocity         ft/sec           Velocity         ft/sec           Pressure Drop (Allow/Calc)         psi           Pressure Drop (Allow/Calc)         psi           Pressure Construction OF ONE SHELL         CONSTRUCTION OF ONE SHELL           Construction OF ONE SHELL         Sketch           Design/Test Pres. psi         500/	Vapor		435,000			0
Steam         Noncondensable           Noncondensable         0         313,900           Liquid Density (In/Out)         Ib/ft <sup>0</sup> 0.000/0.000         46.162/45.419           Liquid Density (In/Out)         Ib/ft <sup>0</sup> 0.000         0.091           Liquid Specific Heat         Btu/hr-FF         0.000         1.644           Liquid Thermal Conductivity         Btu/hr-FF         0.000         0.320           Vapor Viscosity         CP         0.0285         0.0200           Vapor Viscosity         CP         0.0285         0.0200           Vapor Viscosity         CP         0.0285         0.0200           Vapor Viscosity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/675.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft <sup>2</sup> -F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr-ft         F           Transfer Rate, Service	Liquid		0			313,900
Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         Ib/ft <sup>®</sup> 0.000/0.000         46.162/45.419           Liquid Viscosity         cP         0.000         0.091           Liquid Specific Heat         Btu/lb-F         0.000         1.644           Liquid Thermal Conductivity         Btu/hr-ft-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200           Vapor Viscosity         cP         0.0285         0.0200           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure Drop (Allow/Calc)         psi         5.000/1.33         5.000/1.133           Fouling resistance         hr.ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         Shelliside         Tubeside         Sketch           Design/Test Pres.						
Fluid Vaporized or Condensed         0         313,900           Liquid Density (In/Out)         Ib/ft <sup>®</sup> 0.000/0.000         46.162/45.419           Liquid Viscosity         cP         0.000         0.091           Liquid Specific Heat         Btu/lb-F         0.000         1.644           Liquid Thermal Conductivity         Btu/hr-ft-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200           Vapor Viscosity         cP         0.0285         0.0200           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure Drop (Allow/Calc)         psi         5.000/1.33         5.000/1.133           Fouling resistance         hr.ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         Shelliside         Tubeside         Sketch           Design/Test Pres.						
Liquid Density (In/Out)         Ib/ft <sup>3</sup> 0.000/0.000         46.162/45.419           Liquid Viscosity         CP         0.000         0.091           Liquid Specific Heat         Btu/lb-F         0.000         1.644           Liquid Thermal Conductivity         Btu/lb-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft <sup>2</sup> /F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr           De		densed	0		1	313.900
Liquid Viscosity         CP         0.000         0.091           Liquid Specific Heat         Btu/lb-F         0.000         1.644           Liquid Thermal Conductivity         Btu/lb-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/74           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Mol. Weight (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft <sup>2</sup> -F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft <sup>2</sup> -F           Consections         In         1-19.0         6.0         0           Design Temp.			-	)		
Liquid Specific Heat         Btu/lb-F         0.000         1.644           Liquid Thermal Conductivity         Btu/hr-ft-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200           Vapor Viscosity         cP         0.0285         0.0200           Vapor Viscosity         cP         0.0492         0.774           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           Constructions         In         1-19.0         6.0         6.0           Size &         <				,	l	
Liquid Thermal Conductivity         Btu/hr-ft-F         0.000         0.320           Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200           Vapor Specific Heat         Btu/hr-Ft         0.492         0.774           Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203.700,000 Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2 Btu/hr-ft²-F           Construction of Nor Dor SHELL         Shellside         Tubeside         Sketch           Design/Test Pres. psi         500/         1,360/         500         500           No. Passes per Shell         1         6         6         6         6					l	
Vapor Mol. Weight (In/Out)         18.66/18.66         0.0/18.02           Vapor Viscosity         cP         0.0285         0.0200           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Thermal Conductivity         Btu/lb-F         0.067         0.025           Temperature (In/Out)         °F         1.576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1.285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr.ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1.360/         1.360/           Design Temp.         °F         1675         6000         No. Passes per Shell         1         6           Connections         In         1-17.0         12.0         1.17.0         12.0         1.17.0<						-
Vapor Viscosity         CP         0.0285         0.0200           Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Thermal Conductivity         Btu/lb-F         0.492         0.774           Vapor Thermal Conductivity         Btu/lb-F         0.026         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/lb-r         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/lb-rft²-F            500/         1,360/         Design Temp. °F         1675         600           No. Passes per Shell         1         6         600         600         600         600           No. Passes per Shell         1         1.10         6         600         70         70				2		
Vapor Specific Heat         Btu/lb-F         0.492         0.774           Vapor Thermal Conductivity         Btu/lb-F         0.067         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft <sup>2</sup> -F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft <sup>2</sup> -F           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6         6           Cornosion Allow.         in         1-19.0         6.0         6           Size & Out         1-17.0         12.0         1         1           Rating         Intermediate         0         0         0				)	l	
Vapor Thermal Conductivity         Btu/hr-ft-F         0.067         0.025           Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6         6           Corrosion Allow. in         0.0625         0.0625         0.0625           Connections         In         1-17.0         12.0         1           Rating         Intermediate         0         0         0         1           Tube No         1912         OD 1.000 in         Thk 0.065 <td></td> <td></td> <td></td> <td></td> <td></td> <td></td>						
Temperature (In/Out)         °F         1,576.0/624.0         556.0/575.0           Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6         6           Corrosion Allow. in         0.0625         0.0625         0.0625           Connections         In         1-17.0         12.0         1           Rating         Intermediate         0         0         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 3						
Operating Pressure         psi(Abs)         457.000         1,285.000           Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1,360/           Design/Test Pres. psi         500/         1,360/           No. Passes per Shell         1         6           Corrosion Allow. in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover						
Velocity         ft/sec         43.504         8.337           Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Design/Test Pres. psi         500/         1,360/         Sketch           Design Temp.         °F         1675         600         Sketch           No. Passes per Shell         1         6         Connections         In         1-19.0         6.0           Size &         Out         117.0         12.0         Rating         Nternediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         Shell         Shell           Shell         I.D 67.00 OD in         Shell Cover         INT         Channel Cover         INT				.0		
Pressure Drop (Allow/Calc)         psi         5.000/3.245         5.000/1.133           Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Design/Test Pres.         psi         500/         1,360/           Design Temp.         °F         1675         600         Sketch           No. Passes per Shell         1         6         Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0         12.0         12.0         12.0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         1.0         6.0         10         10.           Shell         1.D 67.00 OD in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.		,				
Fouling resistance         hr-ft²-F/Btu         0.001000         0.005000           Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Shellside         Tubeside         Sketch           Design/Test Pres. psi         500/         1,360/         Sketch           Design Temp.         °F         1675         600         Sketch           No. Passes per Shell         1         6         Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT						
Heat Exchanged         203,700,000         Btu/hr         mtd (corr)         307.674 °F           Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Construction of one shead         Sketch           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6           Corrosion Allow. in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT				5		5.000/1.133
Transfer Rate, Service         63.6         Clean         128.2         Btu/hr-ft²-F           CONSTRUCTION OF ONE SHELL           Construction of one shell         Sketch           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6         6           Corrosion Allow. in         0.0625         0.0625         6.0           Size &         Out         1-17.0         12.0         12.0           Rating         Intermediate         0         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         ID 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT			0.001000			0.005000
CONSTRUCTION OF ONE SHELL         Shellside       Tubeside       Sketch         Design/Test Pres. psi       500/       1,360/         Design Temp.       °F       1675       600         No. Passes per Shell       1       6         Corrosion Allow. in       0.0625       0.0625         Connections       In       1-19.0       6.0         Size &       Out       1-17.0       12.0         Rating       Intermediate       0       0         Tube No       1912       OD 1.000 in       Thk 0.065       Length 12.00 ft       Pitch 1.25000 / 30.         Tube Type       PLAIN       Material       Shell       I.D 67.00 OD in       Shell Cover       INT         Channel or Bonnet       Channel Cover       INT       Channel Cover       INT	Heat Exchanged 203	,700,000 Btu/hr	mtd (corr)			
Shellside         Tubeside         Sketch           Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6           Corrosion Allow. in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT	Transfer Rate, Service	63.6	Clean	128.2 Btu/hr-	ft²-F	
Design/Test Pres. psi         500/         1,360/           Design Temp.         °F         1675         600           No. Passes per Shell         1         6           Corrosion Allow. in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT			CONSTRUCTION OF ON	SHELL		
Design Temp.         °F         1675         600           No. Passes per Shell         1         6           Corrosion Allow.         in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length         12.00 ft         Pitch         1.25000 / 30.           Tube Type         PLAIN         Material         Internal         Interna		She	llside Tube	side		Sketch
Design Temp.         °F         1675         600           No. Passes per Shell         1         6           Corrosion Allow.         in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length         12.00 ft         Pitch         1.25000 / 30.           Tube Type         PLAIN         Material         Internal         Interna	Design/Test Pres. psi	500/	1,36	0/		
No. Passes per Shell         1         6           Corrosion Allow.         in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912           OD         1.000 in         Thk 0.065         Length         12.00 ft           Tube Type         PLAIN         Material         Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT         Channel Cover         INT		1675	60	00		
Corrosion Allow.         in         0.0625         0.0625           Connections         In         1-19.0         6.0           Size &         Out         1-17.0         12.0           Rating         Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         Material         INT           Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT		1		6		
Connections Size & Rating         In         1-19.0         6.0           Dut         1-17.0         12.0           Intermediate         0         0           Tube No         1912         OD 1.000 in         Thk 0.065         Length 12.00 ft         Pitch 1.25000 / 30.           Tube Type         PLAIN         Material         INT           Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT		0.0625	0.062	25		
Size & Out 1-17.0 12.0 Rating Intermediate 0 0 Tube No 1912 OD 1.000 in Thk 0.065 Length 12.00 ft Pitch 1.25000 / 30. Tube Type PLAIN Material Shell I.D 67.00 OD in Shell Cover INT Channel or Bonnet Channel Cover						
Rating     Intermediate     0     0       Tube No     1912     OD 1.000 in     Thk 0.065     Length 12.00 ft     Pitch 1.25000 / 30.       Tube Type     PLAIN     Material       Shell     I.D 67.00 OD in     Shell Cover     INT       Channel or Bonnet     Channel Cover     INT					-	
Tube No       1912       OD 1.000 in       Thk 0.065       Length 12.00 ft       Pitch 1.25000 / 30.         Tube Type       PLAIN       Material         Shell       I.D 67.00 OD in       Shell Cover       INT         Channel or Bonnet       Channel Cover       INT						
Tube Type         PLAIN         Material           Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT	ittaing	0				
Tube Type         PLAIN         Material           Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT	Tube No 191	2 OD 1 000 in	Thk 0.065	Length 12.00	ft	Pitch 1 25000 / 30 0°
Shell         I.D 67.00 OD in         Shell Cover         INT           Channel or Bonnet         Channel Cover         INT				2011901 12.00		
Channel or Bonnet Channel Cover	21				INIT	
		1.0 07.00 01		or.	IINI	
Tubaabaat Statianany						
Tubesheet-Stationary Tubesheet-Floating					VEC	
Floating Head Cover Impingement Protection YES						20.1
Baffles Cross Type VERT- SEG %Cut 19.1 (Area) Spacing-cc 29.1		iype vERI-		Alea)	Spacing-cc	29.1
Baffles-Long Seal Type	Barries-Long		, ,	<b>T</b>		
Supports-Tube U-Bend Type						
Bypass Seal Arrangement Tube-Tubesheet Joint		ent		neet Joint		
Expansion Joint Type						
Rho-V2 Inlet Nozzle2,412Bundle Entrance1,266Bundle Exit2,915		2,412	Bundle Entrance	1,266		1
Gasket-Shellside Tubeside Floating Head					Floating Head	
Code Requirement ASME Section 8, Divsion 1 TEMA Class R	Code Requirement	ASME Section	on 8, Divsion 1		TEMA Class	R
Weight/Shell Filled with Water Bundle						

		Н	eat Exchar	nger Specif	ication shee	t	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Tar Reformer S	G Cooler/BF	W Preheater		Item No	H-101	
Size 90x 240		Туре	<b>BEM - HORZ</b>	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	23969 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		23969 ft <sup>2</sup>	
	20000 11		-	NCE OF ONE		20000	
-luid Allocation				Shellside			Tubeside
Fluid Name			Svn	gas fr Tar Ref	ormor		BFW
Total Fluid Enterin	~	lb/br	Syn		JIIIEI		208,600
	y	lb/hr		435,000			,
Vapor				435,000			0
Liquid				0			208,600
Steam							
Noncondensa							
Iuid Vaporized or				0			0
iquid Density (In/	Out)	lb/ft³		0.000/0.000			55.214/46.316
iquid Viscosity		cP		0.000			0.116
iquid Specific Hea	at	Btu/lb-F		0.000			1.368
iquid Thermal Co		Btu/hr-ft-F		0.000			0.358
apor Mol. Weight				18.66/18.66			0.0/0.0
/apor Viscosity	(	сP		0.0199			0.0000
apor Specific Heat	at	Btu/lb-F		0.461			0.000
apor Thermal Co		Btu/hr-ft-F		0.044			0.000
		°F		624.0/370.0			349.0/551.0
emperature (In/O							
Operating Pressur	e	psi(Abs)		452.000			1,285.000
/elocity		ft/sec		33.096			-
Pressure Drop (All	,	psi		10.000/8.600			5.000/0.359
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.001000			0.005000
Heat Exchanged	50,840,000 Bt	u/hr		mtd (corr)	41.736 °F		
Fransfer Rate, Ser	vice	50.8		Clean	86.9 Btu/hr-ft <sup>2</sup>	-F	
			CONSTRUCT	TION OF ONE	SHELL		
		Shel	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	500/		1,350	/		
Design Temp.	°F	675		600			
No. Passes per Sh		1			<u>,</u>		
Corrosion Allow.	in	0.0625		0.062			
Connections	In	1-19.0	n	6.0	,		
		1-19.0	-				
Size &	Out		0	6.0			
Rating	Intermediate	0		0			
		<u> </u>				<i>.</i>	
Tube No	6830	OD 0.750 in		Thk 0.065	Length 20.00	tt	Pitch 1.00000 / 30.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 90.00 OE	) in	Shell Cover		INT	
Channel or Bonnet				Channel Cove			
<b>Subesheet-Station</b>	ary			Tubesheet-Fl			
Floating Head Cov	er			Impingement	Protection	YES	
Baffles Cross		Type VERT-	SEG	%Ċut 13.8 (A	vrea)	Spacing-cc	24.1
Baffles-Long		21		Seal Type	,		
Supports-Tube			U-Bend		Туре		
Sypass Seal Arran	aement			Tube-Tubesh			
71	gement						
Expansion Joint	2	E 104	Dundle Cater	Туре	1 4 4 0	Dundla Euit	4 007
Rho-V2 Inlet Nozz	e	5,194	Bundle Entrar	ice	1,440	Bundle Exit	4,997
Gasket-Shellside			Tubeside			Floating Head	
odo Hoguiromon	t	ASME Sectio	n 8, Divsion 1			TEMA Class	к
Code Requiremen							
Veight/Shell Remarks:			Filled with Wa	ater		Bundle	

Address Plant Location Service of Unit F Size 100x 168	IREL				Job No. Ref No.		200
Address Plant Location Service of Unit F Size 100x 168 Surf/Unit (Eff) 8 Fluid Allocation					Ref No		200
Plant Location Service of Unit F Size 100x 168 Surf/Unit (Eff) 8 Fluid Allocation						HP Syngas Ca	ise
Service of Unit F Size 100x 168 Surf/Unit (Eff) 8 Fluid Allocation					Proposal No.		
Size 100x 168 Surf/Unit (Eff) 8 Fluid Allocation					Date		Rev. 0
Surf/Unit (Eff) 8 Fluid Allocation	lue Gas Cool	er/Steam Sup	erheater		Item No	H-102	
Fluid Allocation		Туре	<b>BEM - HORZ</b>	Connected in	1 Parallel		1 Series
Fluid Allocation	915 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	8915 ft <sup>2</sup>	
			PERFORMAN				
				Shellside			Tubeside
			Flue	e Gas fr. Tar R	leaen	Su	perheated Steam
Total Fluid Entering		lb/hr		280.200			313,900
Vapor				280,200		1	313,900
Liquid				0			0
Steam				0			0
Noncondensab	0						
	-			0		───	0
Fluid Vaporized or C		IL /443				┥────	-
Liquid Density (In/Ou	utj	lb/ft <sup>3</sup>		0.000/0.000		───	0.000/0.000
Liquid Viscosity		cP		0.000		───	0.000
Liquid Specific Heat		Btu/lb-F		0.000		<u> </u>	0.000
Liquid Thermal Cond		Btu/hr-ft-F		0.000		L	0.000
Vapor Mol. Weight (	In/Out)			27.56/27.56			18.02/18.02
Vapor Viscosity		cP		0.0399			0.0254
Vapor Specific Heat		Btu/lb-F		0.314			0.676
Vapor Thermal Cond		Btu/hr-ft-F		0.039			0.036
Temperature (In/Out	t)	°F		1,798.0/839.0	)		575.0/1,000.0
Operating Pressure		psi(Abs)		14.700			1,270.000
Velocity		ft/sec		211.463			4.576
Pressure Drop (Allow	w/Calc)	psi		2.000/1.798			5.000/0.484
Fouling resistance	,	hr-ft²-F/Btu		0.001000			0.005000
Heat Exchanged 8	3 650 000 Bt			mtd (corr)	482.751 °F	4	
Transfer Rate, Servi		19.4		Clean	22.7 Btu/hr-ft	2_F	
	00	10.4	CONSTRUCT			<u> </u>	
		Shel		Tubes		π	Sketch
Design/Test Pres. p	vei	30/	13100	1,350		-	OKeten
	F	1900		1,000		-	
	-	1900			1	-	
No. Passes per She		•			-	-	
Corrosion Allow. in		0.0625	<u>`</u>	0.062	)	-	
	n	1-61.0	-	15.0		_	
	Dut	1-55.0	)	15.0		-	
Rating	ntermediate	0		0			
	900	OD 0.750 in		Thk 0.065	Length 14.00	tt	Pitch 1.25000 / 45.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 100.00 O	D in	Shell Cover		INT	
Channel or Bonnet				Channel Cove	er		
Tubesheet-Stationar				Tubesheet-Flo			
Floating Head Cover	r			Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 40.7 (A	vrea)	Spacing-cc	69.9
Baffles-Long				Seal Type			
Supports-Tube			U-Bend	••	Туре		
Bypass Seal Arrange	ement			Tube-Tubesh			
Expansion Joint				Туре			
Rho-V2 Inlet Nozzle		880	Bundle Entrar		3,144	Bundle Exit	1,037
			Tubeside		- ,	Floating Head	,
Gaskel-Snellsine		ASME Sectio	n 8, Divsion 1			TEMA Class	R
Gasket-Shellside							••
Code Requirement Weight/Shell			Filled with Wa	ter		Bundle	

		Н	leat Exchar	nger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Cs	ae
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Quench Water	Recirculation			Item No	H-200	
Size 42x 120		Туре	<b>BEM - HORZ</b>	Connected in			1 Series
Surf/Unit (Eff)	2867 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		2867 ft <sup>2</sup>	
	2007 10			NCE OF ONE		2007 10	
-luid Allocation				Shellside		1	Tubeside
Fluid Name				Cooling Wate	r	-	Quench Water
	~	lh/hr		1,117,000	I		
Fotal Fluid Enterin	ig	lb/hr		0			105,700 0
Vapor				-			-
Liquid				1,117,000			105,700
Steam							
Noncondensa							
Iuid Vaporized or				0			0
iquid Density (In/	Out)	lb/ft³		61.436/61.06	)		57.041/61.765
iquid Viscosity		cP		0.510			0.301
iquid Specific He	at	Btu/lb-F		1.005			1.017
iquid Thermal Co		Btu/hr-ft-F		1.122		1	0.381
/apor Mol. Weigh				0.0/0.0		1	0.0/0.0
apor Viscosity		cP		0.0000			0.0000
/apor Specific He	at	Btu/lb-F		0.000			0.000
apor Specific fie apor Thermal Co		Btu/hr-ft-F		0.000			0.000
Temperature (In/C		°F		80.0/100.0			311.0/110.0
Operating Pressur	re	psi(Abs)		20.000			456.000
/elocity		ft/sec		3.475			-
Pressure Drop (Al	,	psi		5.000/3.632			5.000/0.424
Fouling resistance		hr-ft²-F/Btu		0.002000			0.001000
Heat Exchanged	22,340,000 Bt	u/hr		mtd (corr)	92.789 °F		
Fransfer Rate, Sei	rvice	84.0		Clean	115.0 Btu/hr-	ft²-F	
			CONSTRUCT	TION OF ONE	SHELL		
		Shel	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	35/		500	1		
Design Temp.	°F	150		41	5		
No. Passes per SI		1			1		
Corrosion Allow.	in	0.0625		0.062			
Connections	In	1-12.	0	4.0			
Size &	Out	1-12.	-	4.0		-	
			0	4.0			
		0					
	Intermediate	0		0			
Rating	Intermediate					0	Ditate 0.00750 / 00.00
Rating	Intermediate	OD 0.750 in		Thk 0.065	Length 10.00	ft	Pitch 0.93750 / 30.0°
Rating Fube No Fube Type	Intermediate	OD 0.750 in PLAIN		Thk 0.065 Material	Length 10.00		Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell	Intermediate 1558 F	OD 0.750 in	D in	Thk 0.065 Material Shell Cover		ft	Pitch 0.93750 / 30.0°
Rating Fube No Fube Type Shell Channel or Bonne	Intermediate 1558 F	OD 0.750 in PLAIN	Din	Thk 0.065 Material Shell Cover Channel Cove	er		Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior	Intermediate 1558 F t ary	OD 0.750 in PLAIN	) in	Thk 0.065 Material Shell Cover Channel Cover Tubesheet-Fl	er pating	INT	Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Toating Head Cov	Intermediate 1558 F t ary	OD 0.750 in PLAIN 1.D 42.00 OE		Thk 0.065 Material Shell Cover Channel Cover Tubesheet-Fl Impingement	er pating Protection	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Toating Head Cov	Intermediate 1558 F t ary	OD 0.750 in PLAIN		Thk 0.065 Material Shell Cover Channel Cover Tubesheet-Fl	er pating Protection	INT	Pitch 0.93750 / 30.0° 24.0
Rating ube No ube Type Shell Channel or Bonne ubesheet-Statior Toating Head Cov Baffles Cross	Intermediate 1558 F t ary	OD 0.750 in PLAIN 1.D 42.00 OE		Thk 0.065 Material Shell Cover Channel Cover Tubesheet-Fl Impingement	er pating Protection	INT	
Rating ube No ube Type Shell Channel or Bonne ubesheet-Statior Toating Head Cov Baffles Cross Baffles-Long	Intermediate 1558 F t ary	OD 0.750 in PLAIN 1.D 42.00 OE		Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 (Å	er pating Protection wrea)	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Toating Head Cov Baffles Cross Baffles-Long Supports-Tube	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE	SEG	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 ( <i>P</i> Seal Type	er pating Protection area) Type	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Toating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE	SEG	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 ( <i>P</i> Seal Type Tube-Tubesh	er pating Protection area) Type	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Toating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE Type VERT-	SEG U-Bend	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 (A Seal Type Tube-Tubesh Type	er pating Protection wrea) Type eet Joint	INT YES Spacing-cc	24.0
Rating Fube No Fube Type Shell Channel or Bonne Fubesheet-Statior Floating Head Cov Baffles Cross Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE	SEG U-Bend Bundle Entrar	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 (A Seal Type Tube-Tubesh Type	er pating Protection area) Type	INT YES Spacing-cc Bundle Exit	24.0
Rating Fube No Fube Type Shell Channel or Bonne Fubesheet-Statior Floating Head Cov Baffles Cross Baffles Cross Baffles Cross Baffles Cross Baffles Cons Baffles	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE Type VERT- 2,540	SEG U-Bend Bundle Entrar Tubeside	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fli Impingement %Cut 22.8 (A Seal Type Tube-Tubesh Type	er pating Protection wrea) Type eet Joint	INT YES Spacing-cc Bundle Exit Floating Head	24.0 3,750
Rating Fube No Fube Type Shell Channel or Bonne Fubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside Code Requiremen Weight/Shell	Intermediate 1558 F t hary ver	OD 0.750 in PLAIN 1.D 42.00 OE Type VERT- 2,540	SEG U-Bend Bundle Entrar	Thk 0.065 Material Shell Cover Channel Cove Tubesheet-Fl Impingement %Cut 22.8 ( <i>P</i> Seal Type Tube-Tubesh Type Tube-Tubesh	er pating Protection wrea) Type eet Joint	INT YES Spacing-cc Bundle Exit	24.0 3,750

		Н	eat Exchar	iger Specif	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev.
Service of Unit	Amine Precool	er/BFW Prehe	eat		Item No	H-201	
Size 56x 168		Туре	<b>BEM - HORZ</b>	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	7511 ft²	Shells/Unit		Surface/Shell		7511 ft²	
			PERFORMAN				
Fluid Allocation				Shellside	•••••		Tubeside
Fluid Name				BFW		Synge	is to Amine Absorber
Total Fluid Enterin	a	lb/hr		320,300		Cynge	414,200
Vapor	'Y	10/11		020,000			414,200
Liquid				320,300			0
				520,500			0
Steam							
Noncondens							40.000
Fluid Vaporized or		11 16:0		0	4		40,260
Liquid Density (In/	Out)	lb/ft <sup>3</sup>		58.527/55.20	1		0.000/56.407
Liquid Viscosity		cP		0.188			0.150
Liquid Specific He	at	Btu/lb-F		1.086			1.037
Liquid Thermal Co		Btu/hr-ft-F		0.393			0.404
Vapor Mol. Weigh	t (In/Out)			0.0/0.0			18.69/18.69
Vapor Viscosity		cP		0.0000			0.0176
Vapor Specific He	at	Btu/lb-F		0.000			0.467
Vapor Thermal Co		Btu/hr-ft-F		0.000			0.040
Temperature (In/C		°F		242.0/349.0			356.0/338.0
Operating Pressu	<u>е</u>	psi(Abs)		1.295.000			442.000
Velocity	0	ft/sec		0.893			18.179
Pressure Drop (Al	low/Calc)	psi		5.000/0.697			5.000/0.635
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.002000			0.001000
Heat Exchanged				mtd (corr)	34.052 °F		0.001000
				( )	300.2 Btu/hr-1	u2 F	
Transfer Rate, Se	vice	144.6	CONSTRUCT	Clean		IF	
		Ohal				1	Ohatah
			lside	Tubes			Sketch
Design/Test Pres.		1,425/		1,360			
Design Temp.	°F	410		40	0		
No. Passes per Sl		1			1		
Corrosion Allow.	in	0.0625		0.062	5		
Connections	In	1-8.0	)	23.0			
Size &	Out	1-8.0	)	23.0			
Rating	Intermediate	0		0			
Tube No	3030	OD 0.750 in		Thk 0.065	Length 14.00	ft	Pitch 0.93750 / 30.0°
Tube Type	F	PLAIN		Material	-		
Shell		I.D 56.00 OE	) in	Shell Cover		INT	
Channel or Bonne	t			Channel Cove	er		
Tubesheet-Station				Tubesheet-Fl			
Floating Head Cov				Impingement		YES	
Baffles Cross	~ ~ ·	Type VERT-	SEG	%Cut 10.0 (A		Spacing-cc	11.1
Baffles-Long		· )po / Litt-	020	Seal Type		Spacing-00	
Supports-Tube			U-Bend	ocar iype	Туре		
	acmont		0-Denu	Tube-Tubesh	Type		
Bypass Seal Arran	igement						
Expansion Joint	1	4.440	Dunalla Est	Туре	407	Dunella Eritt	4 505
Rho-V2 Inlet Nozz	le	1,110	Bundle Entrar	ice	167	Bundle Exit	1,585
Gasket-Shellside			Tubeside			Floating Head	
Code Requiremen Weight/Shell	t	ASME Section	n 8, Divsion 1 Filled with Wa			TEMA Class Bundle	R

		Н	leat Exchan	iger Speci	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas C	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Amine Precool	er/Deaerator	FW Preheat		Item No	H-202	
Size 40x 72		Туре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	585 ft²	Shells/Unit	1	Surface/Shel	I (Effective)	585 ft²	
			PERFORMAN	ICE OF ONE	ÛNIT		
Fluid Allocation				Shellside			Tubeside
Fluid Name			Synga	as to Amine A	bsorber	Dea	aerator Feed Water
Total Fluid Enterin	a	lb/hr	, , ,	414,200			320,000
Vapor	0			373,940			0
Liquid				40.260			320,000
Steam				-,			/
Noncondensa	able						
Fluid Vaporized or				9,444			0
Liquid Density (In/		lb/ft³		55.290/55.49	2		59.180/58.595
Liquid Viscosity	Call	cP		0.092	-	ł	0.262
Liquid Specific He	at	Btu/lb-F		1.111		<del> </del>	1.020
Liquid Thermal Co		Btu/hr-ft-F		0.395		ł	0.385
Vapor Mol. Weight		Dtu/III-It-F		18.96/18.943	6	<del> </del>	0.0/0.0
Vapor Viscosity		сР		0.0179			0.0000
Vapor Specific He	at	Btu/lb-F		0.0179			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.445			0.000
		°F		338.0/332.0	1		212.0/239.4
Temperature (In/C		psi(Abs)		437.000			
Operating Pressur	e	,		25.051			30.000
Velocity		ft/sec					-
Pressure Drop (All		psi		5.000/1.075	)		5.000/0.287
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged				mtd (corr)	108.950 °F		
Transfer Rate, Ser	vice	145.0	CONSTRUCT	Clean	322.3 Btu/hr-	ft²-F	
		Shal	Iside			1	Sketch
Dealan/Test Dres	noi	480/		45		-	Skelch
Design/Test Pres.	°F	480/		30		4	
Design Temp.		400			-	4	
No. Passes per Sh					1		
Corrosion Allow.	in	0.0625		0.062	ວ	4	
Connections	ln Out	1-23.	-	8.0		4	
Size &	Out	1-19.	U	8.0		4	
Rating	Intermediate	0		0			
Tube No	550	OD 0.750 in		Thk 0.065	Length 6.00 f	ł	Pitch 1.25000 / 45.0°
		PLAIN		Material		L	Filon 1.20000 / 40.0
Tube Type Shell	F	1.D 40.00 OE	) in	Shell Cover		INT	•
Channel or Bonne	ł	1.D 40.00 OL	ווו כ	Channel Cover	or	IINI	
	-						
Tubesheet-Station				Tubesheet-Fl		YES	
Floating Head Cov			850	Impingement			20.0
Baffles Cross		Type VERT-	326	%Cut 49.0 (/		Spacing-cc	38.9
Baffles-Long			II Dond	Seal Type	Tuno		
Supports-Tube			U-Bend	Tube Tube 1	Type		
Bypass Seal Arran	igement			Tube-Tubesh	ieet Joint		
Expansion Joint		4 400	<u> </u>	Туре	0.400	<u> </u>	0.500
Rho-V2 Inlet Nozz	le	1,486	Bundle Entrar	ice	2,490	Bundle Exit	2,529
Gasket-Shellside			Tubeside			Floating Head	
Code Requiremen	t	ASME Section	on 8, Divsion 1			TEMA Class	R
Weight/Shell			Filled with Wa	iter		Bundle	

		Н	eat Exchar	nger Specif	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Amine Precool	er			Item No	H-203	
Size 96x 96		Туре	<b>BEM - HORZ</b>	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	11541 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		11541 ft <sup>2</sup>	
0011/0111 (L11)			•	NCE OF ONE			
Fluid Allocation				Shellside	•••••	1	Tubeside
Fluid Name			Syna	as to Amine Al	hsorber		Cooling Water
Total Fluid Entering	2	lb/hr	Synga	414,200	0301061		6,965,000
Vapor	9			364,537			0,903,000
				,			-
Liquid				49,663			6,965,000
Steam							
Noncondensa							
Fluid Vaporized or				97,296		ļ	0
Liquid Density (In/0	Out)	lb/ft³		55.608/62.12	0		62.000/61.573
Liquid Viscosity		cP		0.211			0.627
Liquid Specific Hea		Btu/lb-F		1.063			1.001
Liquid Thermal Co		Btu/hr-ft-F		0.384			0.365
Vapor Mol. Weight				18.8591/18.9	6		0.0/0.0
Vapor Viscosity	· /	cP		0.0168			0.0000
Vapor Specific Hea	at	Btu/lb-F		0.424			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.041			0.000
Temperature (In/O		°F		332.0/110.0		1	80.0/100.0
Operating Pressure		psi(Abs)		432.000			65.000
Velocity	5	ft/sec		13.546			-
Pressure Drop (All				5.000/1.874			5.000/0.592
<b>.</b>	ow/Calc)	psi					
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.001000			0.002000
Heat Exchanged				mtd (corr)	98.751 °F		
Transfer Rate, Ser	vice	122.2		Clean	210.0 Btu/hr-	ft²-F	
			CONSTRUCT				
		Shel	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	475/		80	/		
Design Temp.	°F	350		150	0	1	
No. Passes per Sh	ell	1			1	1	
Corrosion Allow.	in	0.0625		0.062	5	1	
Connections	In	1-23.	0	31.0		1	
Size &	Out	1-17.	-	31.0		4	
Rating	Intermediate	0	•	0		4	
lating	Internediate	Ű		Ű			
Tube No	8842	OD 0.750 in		Thk 0.065	Length 8.00 f	t	Pitch 0.93750 / 30.0°
Tube Type		PLAIN		Material	Length 0.001	ι	1 1101 0.007 00 7 00.0
	Г	I.D 96.00 OE	) in	Shell Cover		INT	
		1.0 90.00 OL	7 11 1			IINT	
Shell							
Shell Channel or Bonnet				Channel Cove			
Shell Channel or Bonnet Tubesheet-Station	ary			Tubesheet-Fl	oating		
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov	ary	_		Tubesheet-Fl Impingement	oating Protection	YES	
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross	ary	Type VERT-	SEG	Tubesheet-Fl Impingement %Cut 18.6 (A	oating Protection	YES Spacing-cc	39.8
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long	ary	Type VERT-		Tubesheet-Fl Impingement	oating Protection Area)		39.8
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	ary er	Type VERT-	SEG U-Bend	Tubesheet-Fl Impingement %Cut 18.6 (A Seal Type	oating Protection Area) Type		39.8
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	ary er	Type VERT-		Tubesheet-Fl Impingement %Cut 18.6 (A	oating Protection Area) Type		39.8
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran	ary er	Type VERT-		Tubesheet-Fl Impingement %Cut 18.6 (A Seal Type	oating Protection Area) Type		39.8
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint	ary er gement	Type VERT-		Tubesheet-Fl Impingement %Cut 18.6 ( <i>F</i> Seal Type Tube-Tubesh Type	oating Protection Area) Type		39.8 3,610
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl	ary er gement		U-Bend Bundle Entrar	Tubesheet-Fl Impingement %Cut 18.6 ( <i>F</i> Seal Type Tube-Tubesh Type	oating Protection Area) Type eet Joint	Spacing-cc Bundle Exit	3,610
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl Gasket-Shellside	ary er gement e	1,463	U-Bend Bundle Entrar Tubeside	Tubesheet-Fl Impingement %Cut 18.6 ( <i>F</i> Seal Type Tube-Tubesh Type	oating Protection Area) Type eet Joint	Spacing-cc Bundle Exit Floating Head	3,610
Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl Gasket-Shellside Code Requirement Weight/Shell	ary er gement e	1,463	U-Bend Bundle Entrar	Tubesheet-Fl Impingement %Cut 18.6 (A Seal Type Tube-Tubesh Type nce	oating Protection Area) Type eet Joint	Spacing-cc Bundle Exit	3,610

1		н	eat Exchanger	Specification a	sheet	
				Job No.		
Customer	NREL			Ref No.	HP Syngas C	ase
Address				Proposal	No.	
Plant Location				Date		Rev. 0
Service of Unit	ZnO Preheater	r		Item No	H-320	
Size 96x 96		Туре	BEM - HORZ Conne	ected in 1 Pa	arallel	1 Series
Surf/Unit (Eff)	19400 ft <sup>2</sup>	Shells/Unit	1 Surfa	ce/Shell (Effective)	) 19400 ft <sup>2</sup>	
			PERFORMANCE O	F ONE ÙNIT		
Fluid Allocation	,		Sh	ellside		Tubeside
Fluid Name			Flue Gas	fr. Tar Regen		Sweet Syngas
Total Fluid Enterin	a	lb/hr		30.200		118,500
Vapor	5			30,200		118,500
Liquid				0		0
Steam				0		0
Noncondensa	able					
Fluid Vaporized or				0		0
Liquid Density (In/		lb/ft <sup>3</sup>	0.00	0/0.000		0.000/0.000
1 2 1	July	cP				
Liquid Viscosity	ot	-		0.000		0.000
Liquid Specific Hea		Btu/lb-F		0.000		0.000
Liquid Thermal Co		Btu/hr-ft-F		0.000		0.000
Vapor Mol. Weight	t (In/Out)			56/27.56		10.99/10.99
Vapor Viscosity		cP		.0157		0.0182
Vapor Specific Hea		Btu/lb-F		).312		0.659
Vapor Thermal Co		Btu/hr-ft-F		0.012		0.076
Temperature (In/O		°F	839.	.0/214.0		100.0/750.0
Operating Pressur	e	psi(Abs)	14	4.500		422.000
Velocity		ft/sec	64	4.628		2.701
Pressure Drop (All	ow/Calc)	psi	2.00	0/1.675		5.000/0.488
Fouling resistance	,	hr-ft <sup>2</sup> -F/Btu	0.0	02000		0.002000
Heat Exchanged	49,960,000 Bt	tu/hr	mtd (d	corr) 96.31 °F		
Transfer Rate, Ser		26.55	Clean	,		
			CONSTRUCTION C		• •	
		She	Iside	Tubeside		Sketch
Design/Test Pres.	psi	30/		465/		
Design Temp.	°F	910		800		
No. Passes per Sh		1		1		
Corrosion Allow.	in	0.0625		0.0625		
CONDSION ANOW.						
Connections			1			
	In	1-53.	-	12.0		
Size &	In Out	1-53. 1-47.	-	12.0 15.0		
Size &	In	1-53.	-	12.0		
Size & Rating	In Out Intermediate	1-53. 1-47. 0	D	12.0 15.0 0	2 00 #	
Size & Rating Tube No	In Out Intermediate 14190	1-53. 1-47. 0 OD 0.750 in	D Thk (	12.0 15.0 0 0.065 Length 8	3.00 ft	Pitch 1.25000 / 45.0°
Connections Size & Rating Tube No Tube Type	In Out Intermediate 14190	1-53. 1-47. 0 OD 0.750 in PLAIN	D Thk ( Mater	12.0 15.0 0 0.065 Length 8		Pitch 1.25000 / 45.0°
Size & Rating Tube No Tube Type Shell	In Out Intermediate 14190 F	1-53. 1-47. 0 OD 0.750 in	D Thk ( Mater D in Shell	12.0 15.0 0 0.065 Length 8 rial Cover	3.00 ft	Pitch 1.25000 / 45.0°
Size & Rating Tube No Tube Type Shell Channel or Bonnet	In Out Intermediate 14190 F	1-53. 1-47. 0 OD 0.750 in PLAIN	D Thk ( Mater D in Shell Chan	12.0 15.0 0 0.065 Length 8 rial Cover nel Cover		Pitch 1.25000 / 45.0°
Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station	In Out Intermediate 14190 F t tary	1-53. 1-47. 0 OD 0.750 in PLAIN	D Thk ( Mater D in Shell Chan Tubes	12.0 15.0 0 0.065 Length 8 rial Cover nel Cover sheet-Floating	INT	Pitch 1.25000 / 45.0°
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov	In Out Intermediate 14190 F t tary	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating gement Protection	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross	In Out Intermediate 14190 F t tary	1-53. 1-47. 0 OD 0.750 in PLAIN	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area)	INT	Pitch 1.25000 / 45.0° 65.0
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long	In Out Intermediate 14190 F t tary	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut Seal	12.0 15.0 0 0.065 Length 8 rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area) Type	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	In Out Intermediate 14190 F t t hary /er	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area)	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	In Out Intermediate 14190 F t t hary /er	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut Seal	12.0 15.0 0 0.065 Length 8 rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area) Type	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran	In Out Intermediate 14190 F t t hary /er	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut Seal	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating gement Protection 36.0 (Area) Type Type	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint	In Out Intermediate 14190 F t tary ver	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater D in Shell Chan Tubes Impin SEG %Cut Seal U-Bend Tube-	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating gement Protection 36.0 (Area) Type Type	INT	
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozz	In Out Intermediate 14190 F t tary ver	1-53. 1-47. 0 OD 0.750 in PLAIN I.D 163.00 C	D Thk ( Mater Mater D in Shell Chan Tubes Impin SEG %Cut Seal <sup>7</sup> U-Bend Tube- Type	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area) Type Type Tubesheet Joint	INT YES Spacing-cc Bundle Exit	65.0
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside	In Out Intermediate 14190 F t hary /er	1-53. 1-47. 0 OD 0.750 in PLAIN 1.D 163.00 C Type VERT- 900	D Thk ( Mater D in Shell Chanı Tubes Impin SEG %Cut Seal U-Bend U-Bend Tube- Type Bundle Entrance Tubeside	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area) Type Type Tubesheet Joint	INT YES Spacing-cc	65.0
Size & Rating Tube No Tube Type Shell Channel or Bonnel Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozz	In Out Intermediate 14190 F t hary /er	1-53. 1-47. 0 OD 0.750 in PLAIN 1.D 163.00 C Type VERT- 900	D Thk ( Mater Mater D in Shell Chan Tubes Impin SEG %Cut Seal U-Bend U-Bend Tube- Type Bundle Entrance	12.0 15.0 0 0.065 Length & rial Cover nel Cover sheet-Floating igement Protection : 36.0 (Area) Type Type Tubesheet Joint	INT YES Spacing-cc Bundle Exit Floating Head	65.0

		Н	leat Exchar	iger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	ZnO SG Coole	r/BFW Prehea	ater		Item No	H-321	
Size 60x 192		Туре	<b>BEM - HORZ</b>	Connected in	1 Paralle		1 Series
Surf/Unit (Eff)	5440 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	5440 ft <sup>2</sup>	
			PERFORMA	ICE OF ONE	<u>ÙNIT</u>		
Fluid Allocation				Shellside			Tubeside
Fluid Name	-		Sv	ngas fr ZnO B	eds		BFW
Total Fluid Entering	a	lb/hr	- ,	118,500			111,600
Vapor	5	10/11		118,500			0
Liquid				0			111,600
Steam				0			111,000
Noncondensa	blo						
				0			0
Fluid Vaporized or		IL /443		-			•
Liquid Density (In/	Jul)	lb/ft <sup>3</sup>		0.000/0.000			54.688/45.460
Liquid Viscosity	- 1	cP		0.000			0.115
Liquid Specific Hea		Btu/lb-F		0.000			1.429
Liquid Thermal Co		Btu/hr-ft-F		0.000		ļ	0.352
Vapor Mol. Weight	(In/Out)			10.99/10.99		L	0.0/0.0
Vapor Viscosity		cP		0.0203			0.0000
Vapor Specific Hea		Btu/lb-F		0.663			0.000
Vapor Thermal Co	nductivity	Btu/hr-ft-F		0.086			0.000
Temperature (In/O	ut)	°F		750.0/370.0			349.0/565.0
Operating Pressure	е	psi(Abs)		412.000			1,285.000
Velocity		ft/sec		30.448			-
Pressure Drop (All	ow/Calc)	psi		5.000/3.935			5.000/0.407
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged				mtd (corr)	75.373 °F		
Transfer Rate, Ser		72.8		Clean	99.1 Btu/hr-ft	<sup>2</sup> -F	
	100	12.0	CONSTRUCT	TON OF ONE		•	
		Shal	Iside	Tubes			Sketch
Design/Test Pres.	nei	455/	13106	1,350			OREICH
Design Temp.	°F	800		615		-	
		1				-	
No. Passes per Sh Corrosion Allow.					•		
	in	0.0625	0	0.0625	)		
Connections	In	1-15.	-	4.0			
Size &	Out	1-13.	0	6.0			
Rating	Intermediate	0		0			
						-	
Tube No	1902	OD 0.750 in		Thk 0.065	Length 16.00	tt	Pitch 1.25000 / 30.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 60.00 OE	) in	Shell Cover		INT	
Channel or Bonnet				Channel Cove	er		
Tubesheet-Station				Tubesheet-Flo			
Floating Head Cov	er			Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 14.0 (A	vrea)	Spacing-cc	14.5
Baffles-Long				Seal Type			
			U-Bend		Туре		
Supports-Tube	aement			Tube-Tubesh			
Supports-Tube Bypass Seal Arran	gement			Туре			
	igement						
Bypass Seal Arran Expansion Joint	0	2.063	Bundle Entrar		272	Bundle Fxit	2.203
Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl	0	2,063	Bundle Entrar Tubeside		272	Bundle Exit Floating Head	2,203
Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl Gasket-Shellside	le		Tubeside		272	Floating Head	, ,
Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl	le			nce	272		

		Н	leat Exchai	nger Specif	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Post ZnO Syng	gas Cooler			Item No	H-322	
Size 36x 96		Туре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	1620 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	1620 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE			
Fluid Allocation				Shellside			Tubeside
Fluid Name			Sv	ngas fr. ZnO E	Beds		Cooling Water
Total Fluid Enterin	a	lb/hr		118,500			995,500
Vapor	5			118,500			0
Liquid				0			995,500
Steam				•			000,000
Noncondens	ahla						
Fluid Vaporized or				0		ļ	0
Liquid Density (In/		lb/ft <sup>3</sup>		0.000/0.000			62.000/62.000
	ourj	cP					
Liquid Viscosity	ot	Etu/lb-F		0.000		<b> </b>	0.762
Liquid Specific He				0.000			1.000
Liquid Thermal Co		Btu/hr-ft-F		0.000		ļ	0.363
Vapor Mol. Weigh	t (in/Out)			10.99/10.99		ļ	0.0/0.0
Vapor Viscosity		cP		0.0148			0.0000
Vapor Specific He		Btu/lb-F		0.647			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.065			0.000
Temperature (In/C		°F		370.0/110.0			80.0/100.0
Operating Pressur	e	psi(Abs)		407.000			65.000
Velocity		ft/sec		47.403			-
Pressure Drop (Al		psi		5.000/3.747			5.000/0.585
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged	19,910,000 Bt	u/hr		mtd (corr)	109.229 °F		
Transfer Rate, Se	vice	112.6		Clean	183.2 Btu/hr-	ft²-F	
			CONSTRUC	TION OF ONE	SHELL		
		She	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	450/		80	)/		
Design Temp.	°F	420		15	0		
No. Passes per SI	nell	1			1		
Corrosion Allow.	in	0.0625		0.062	5		
Connections	In	1-13.	0	12.0	-		
Size &	Out	1-12.		12.0			
Rating	Intermediate	0	•	0			
rtating	intorniouluto	Ű		Ű			
Tube No	1102	OD 0.750 in		Thk 0.065	Length 8.00 f	t	Pitch 0.93750 / 30.0°
Tube Type	-	PLAIN		Material	Longth 0.001		1.1.511 0.007 00 7 00.0
Shell	F	I.D 36.00 OE	) in	Shell Cover		INT	
Channel or Bonne	+	1.D 00.00 OL		Channel Cove	ar	1111	
Tubesheet-Station	-						
Floating Head Cov				Tubesheet-FI Impingement		YES	
0		Type VERT-	SEC				24.0
Baffles Cross		Type VERT-	320	%Cut 24.3 (A Seal Type	nica)	Spacing-cc	2 <del>4</del> .U
Baffles-Long			II Dond	Sear Type	Tuno		
Supports-Tube	romont		U-Bend	Tubo Tubo -	Type		
Bypass Seal Arrar	igement			Tube-Tubesh	eet Joint		
Expansion Joint	1	0.500		Туре	1.001	<u> </u>	0.075
Rho-V2 Inlet Nozz	le	2,539	Bundle Entra	nce	1,981	Bundle Exit	3,675
Gasket-Shellside			Tubeside			Floating Head	
Code Description	t	ASME Section	on 8, Divsion 1			TEMA Class	R
Code Requiremen	-						
Weight/Shell Remarks:	- 		Filled with Wa	ater		Bundle	

		Н	eat Exchar	nger Specif	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	MeOH Compre	essor Interstag	je Cooler		Item No	H-400A	
Size 23x 72		Туре	<b>BEM - HORZ</b>	Connected in	1 Paralle		1 Series
Surf/Unit (Eff)	476 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	476 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE			
Fluid Allocation				Shellside			Tubeside
Fluid Name				Cooling wate	r		Syngas
Total Fluid Enterin	ומ	lb/hr		537,000			118,500
Vapor	.9			0			118,500
Liquid				537,000			0
Steam				337,000			0
Noncondens	ablo						
			<b> </b>	0			0
Fluid Vaporized or		11, 1613	l	0	0		0
Liquid Density (In/	Out)	lb/ft <sup>3</sup>		62.000/62.00	U	ļ	0.000/0.000
Liquid Viscosity		cP	<b> </b>	0.762			0.000
Liquid Specific He		Btu/lb-F	L	1.000			0.000
Liquid Thermal Co		Btu/hr-ft-F	L	0.363			0.000
Vapor Mol. Weigh	t (In/Out)			0.0/0.0			10.99/10.99
Vapor Viscosity		cP		0.0000			0.0155
Vapor Specific He		Btu/lb-F		0.000			0.655
Vapor Thermal Co		Btu/hr-ft-F		0.000			0.068
Temperature (In/C		°F		80.0/100.0			338.0/200.0
Operating Pressur	re	psi(Abs)		65.000			1,000.000
Velocity		ft/sec		4.236			25.340
Pressure Drop (Al	low/Calc)	psi		5.000/2.578			5.000/0.675
Fouling resistance	,	hr-ft²-F/Btu		0.002000			0.001000
Heat Exchanged	10 470 000 Bt			mtd (corr)	172.318 °F		
Transfer Rate, Se	rvice	127.7		Clean	216.4 Btu/hr-	ft²_F	
	11100	121.1	CONSTRUCT	TION OF ONE			
		Shol	Iside	Tubes		1	Sketch
Design/Test Pres.	nci	80/					Sketch
	°F	150	•••		1,050/ 390		
Design Temp.					-	-	
No. Passes per Sl		0.0625		1		-	
Corrosion Allow.	inections In 1			0.0625			
Connections			1-10.0		12.0		
	Out	1-10.	0	10.0			
	Out Intermediate	1-10.0 0	0	10.0 0			
Rating	Intermediate	0		0			
Rating Tube No	Intermediate 442	0 OD 0.750 in		0 Thk 0.065	Length 6.00 f	t	Pitch 0.93750 / 30.0°
Size & Rating Tube No Tube Type	Intermediate 442	0 OD 0.750 in PLAIN		0 Thk 0.065 Material	Length 6.00 f		Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell	Intermediate 442 F	0 OD 0.750 in		0 Thk 0.065 Material Shell Cover		t INT	Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne	Intermediate 442 F	0 OD 0.750 in PLAIN		0 Thk 0.065 Material Shell Cover Channel Cover	er		Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior	Intermediate 442 F at nary	0 OD 0.750 in PLAIN		0 Thk 0.065 Material Shell Cover	er	INT	Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior	Intermediate 442 F at nary	0 OD 0.750 in PLAIN		0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement	er oating Protection		Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov	Intermediate 442 F at nary	0 OD 0.750 in PLAIN	Din	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-Fl	er oating Protection	INT	Pitch 0.93750 / 30.0°
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross	Intermediate 442 F at nary	0 OD 0.750 in PLAIN I.D 23.25 OE	Din	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement	er oating Protection	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long	Intermediate 442 F at nary	0 OD 0.750 in PLAIN I.D 23.25 OE	Din	0 Thk 0.065 Material Shell Cover Channel Cove Tubesheet-FI Impingement %Cut 23.5 ( <i>P</i>	er oating Protection Area)	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	Intermediate 442 F et nary ver	0 OD 0.750 in PLAIN I.D 23.25 OE	) in SEG	0 Thk 0.065 Material Shell Cover Channel Cove Tubesheet-FI Impingement %Cut 23.5 ( <i>P</i>	er oating Protection Area) Type	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles -Long Supports-Tube Bypass Seal Arrar	Intermediate 442 F et nary ver	0 OD 0.750 in PLAIN I.D 23.25 OE	) in SEG	0 Thk 0.065 Material Shell Cover Channel Cove Tubesheet-FI Impingement %Cut 23.5 ( <i>I</i> Seal Type Tube-Tubesh	er oating Protection Area) Type	INT	
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint	Intermediate 442 F et nary ver	0 OD 0.750 in PLAIN T.D 23.25 OE Type VERT-	D in SEG U-Bend	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 23.5 ( <i>F</i> Seal Type Tube-Tubesh Type	er oating Protection Area) Type eet Joint	INT YES Spacing-cc	16.3
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	Intermediate 442 F et nary ver	0 OD 0.750 in PLAIN I.D 23.25 OE	D in SEG U-Bend Bundle Entrar	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 23.5 ( <i>F</i> Seal Type Tube-Tubesh Type	er oating Protection Area) Type	INT YES Spacing-cc Bundle Exit	16.3
Rating Tube No Tube Type Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside	Intermediate 442 F et nary ver ngement zle	0 OD 0.750 in PLAIN I.D 23.25 OE Type VERT- 1,206	D in SEG U-Bend Bundle Entrar Tubeside	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 23.5 ( <i>I</i> Seal Type Tube-Tubesh Type nce	er oating Protection Area) Type eet Joint	INT YES Spacing-cc Bundle Exit Floating Head	16.3
Rating Tube No	Intermediate 442 F et nary ver ngement zle	0 OD 0.750 in PLAIN I.D 23.25 OE Type VERT- 1,206	D in SEG U-Bend Bundle Entrar	0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 23.5 ( <i>I</i> Seal Type Tube-Tubesh Type nce	er oating Protection Area) Type eet Joint	INT YES Spacing-cc Bundle Exit	16.3

		н	eat Exchai	nger Speci	fication shee	et	
				-	Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	MeOH Syngas	Preheat			Item No	H-401	
Size 72x 216		Туре	BEM - HORZ	Connected in		-	1 Series
	16212 ft <sup>2</sup>	Shells/Unit	1	Surface/Shel		16212 ft <sup>2</sup>	
		oriologi orint	-	NCE OF ONE			
Fluid Allocation				Shellside	•••••	1	Tubeside
Fluid Name				Steam		Svr	igas to MeOH Rxn
Total Fluid Entering	N	lb/hr		17,610		Oyi	118,500
Vapor	1	10/11		17,610			118,500
				0			0
Liquid				0			0
Steam	<b>b</b> . <b>1</b> -						
Noncondensa				47.040			
Iuid Vaporized or				17,610	2		0
_iquid Density (In/C	Dut)	lb/ft <sup>3</sup>		0.000/54.780	J	ļ	0.000/0.000
Liquid Viscosity		cP		0.128			0.000
_iquid Specific Hea		Btu/lb-F		1.157			0.000
_iquid Thermal Cor		Btu/hr-ft-F		0.393			0.000
Vapor Mol. Weight	(In/Out)			18.02/18.02			10.99/10.99
Vapor Viscosity		cP		0.0161			0.0170
Vapor Specific Hea	ıt	Btu/lb-F		0.483			0.660
/apor Thermal Cor		Btu/hr-ft-F		0.020			0.074
Temperature (In/O		°F		471.7/324.0	)		240.0/460.0
Operating Pressure		psi(Abs)		100.000			1,165.000
Velocity	,	ft/sec		4.482			2.118
Pressure Drop (Allo		psi		5.000/0.586			5.000/0.430
Fouling resistance	Jw/Calc)	hr-ft²-F/Btu		0.005000			0.001000
<u>v</u>	17 4 40 000 D						0.001000
Heat Exchanged				mtd (corr)	45.146 °F	~ _	
Transfer Rate, Serv	/ice	23.4		Clean	27.4 Btu/hr-ft	<b>-</b> ⊢	
				TION OF ONE		-	
			lside	Tubes			Sketch
Design/Test Pres.		130/		1,225			
Design Temp.	°F	545		51	5		
No. Passes per Sh	ell	1			1		
Corrosion Allow.	osion Allow. in 0.0625		0.0625				
Connections	ections In 1-8.0		10.0		1		
Size &	Out	1-2.0	)	12.0			
Rating	Intermediate	0		0			
<b>J</b>							
Fube No	5044	OD 0.750 in		Thk 0.065	Length 18.00	ft	Pitch 0.93750 / 30.0°
Tube Type		PLAIN		Material	Longar 10.00	it i	
Shell	•	I.D 72.00 OE	) in	Shell Cover		INT	
Channel or Bonnet		1.D 72.00 OL		Channel Cover	or		
					÷.		
Tubesheet-Stationa				Tubesheet-F	0	NO	
Floating Head Cove	31		050	Impingement		NO Secondaria	11.0
Baffles Cross		Type VERT-	SEG	%Cut_10.2 (/	Area)	Spacing-cc	14.3
Baffles-Long				Seal Type	<del>.</del>		
Supports-Tube			U-Bend		Туре		
Bypass Seal Arran	gement			Tube-Tubesh	eet Joint		
Expansion Joint				Туре			
	e	1,057	Bundle Entra	nce	1,398	Bundle Exit	1,158
Rho-V2 Inlet Nozzle			Tulsasida			Floating Head	
			Tubeside			i louding i loud	
Rho-V2 Inlet Nozzl Gasket-Shellside Code Requirement		ASME Sectio	n 8, Divsion 1			TEMA Class	R
Gasket-Shellside		ASME Sectio					

		Н	eat Exchar	nger Specif	ication she	et	
					Job No.		
Customer	NREL				Ref No.	HP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Blowdown Coo	oler			Item No	H-501	
Size 15x 48		Туре	BEM - HORZ	Connected in			1 Series
Surf/Unit (Eff)	130 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		130 ft <sup>2</sup>	
040(2)			PERFORMA	NCE OF ONE		100 10	
Fluid Allocation				Shellside	•	1	Tubeside
Fluid Name				Blowdown			Cooling water
Total Fluid Enterin	~	lb/hr		3,987			41,985
Vapor	y	ID/III		0			0
				-			
Liquid				3,987			41,985
Steam							
Noncondensa							
Fluid Vaporized or				0			0
Liquid Density (In/	Out)	lb/ft³		56.607/62.00	0		62.000/62.000
Liquid Viscosity		cP		0.311			0.762
Liquid Specific He		Btu/lb-F		1.059			1.000
Liquid Thermal Co		Btu/hr-ft-F		0.382			0.363
Vapor Mol. Weight				0.0/0.0			0.0/0.0
Vapor Viscosity	( )	сP		0.0000			0.0000
Vapor Specific He	at	Btu/lb-F		0.000			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.000			0.000
Temperature (In/C		°F		298.0/110.0			80.0/100.0
Operating Pressur		psi(Abs)		65.000			65.000
Velocity	e	ft/sec		0.143			0.528
Pressure Drop (All		psi		5.000/0.154			5.000/0.206
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged				mtd (corr)	89.027 °F		
Transfer Rate, Sei	vice	72.7		Clean	97.5 Btu/hr-ft	.²-F	
				TION OF ONE	SHELL		
		Shel	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	80/		80	/		
Design Temp.	°F	350		150	)		
No. Passes per Sh	nell	1			1		
Corrosion Allow.	in	0.0625		0.0625			
Connections	In	1-1.0	)	3.0	m		
Size &	Out	1-1.0		3.0			
Rating	Intermediate	0	•	0.0		-	
rading	intermediate	Ŭ		Ŭ			
Tube No	170	OD 0.750 in		Thk 0.065	Length 4.00 f	4	Pitch 0.93750 / 30.0°
Tube Type	-	PLAIN		Material	Length 4.001	ι	T II.01 0.33730 / 30.0
Shell	F	I.D 15.25 OE	) in			INT	
	4	1.0 13.25 UL	וויל	Shell Cover Channel Cove	or .	INT	
Channel or Bonne				Tubesheet-Fl			
Tubesheet-Station				Impingement	Protection	YES	
Tubesheet-Station Floating Head Cov		_		1 0		<b>.</b> .	
Tubesheet-Station Floating Head Cov Baffles Cross		Type VERT-	SEG	%Cut 8.6 (Ar		Spacing-cc	3.0
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long		Type VERT-		1 0	ea)	Spacing-cc	3.0
Tubesheet-Station Floating Head Cov Baffles Cross		Type VERT-	SEG U-Bend	%Cut 8.6 (Ar		Spacing-cc	3.0
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long	ver	Type VERT-		%Cut 8.6 (Ar	ea) Type	Spacing-cc	3.0
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	ver	Type VERT-		%Cut 8.6 (Ar Seal Type	ea) Type	Spacing-cc	3.0
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint	igement		U-Bend	%Cut 8.6 (Ar Seal Type Tube-Tubesh Type	ea) Type		3.0
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	igement	Type VERT-	U-Bend Bundle Entrar	%Cut 8.6 (Ar Seal Type Tube-Tubesh Type	ea) Type eet Joint	Bundle Exit	423
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside	ngement	728	U-Bend Bundle Entrar Tubeside	%Cut 8.6 (Ar Seal Type Tube-Tubesh Type	ea) Type eet Joint	Bundle Exit Floating Head	423
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	ngement	728	U-Bend Bundle Entrar	%Cut 8.6 (Ar Seal Type Tube-Tubesh Type nce	ea) Type eet Joint	Bundle Exit	423

COMPRESSO				K-100					
SERVICE				Combustion Air					
GAS HANDLE	D			Air					
NORMAL FLO	w	SCFM		58,597					
NORMAL FLO	w	LB/HR		265,200					
DESIGN FLOW	v	SCFM							
MOL WT.				28.63					
C_/C_		Value		1.4					
		@ F / P	SIA	90 / 14.7					
SUCTION CON									
	PRESSURE	PSIA		14.7					
	FACTOR @ SUCTION	ACFM		0.999 61,910					
ORIGIN	SUCTION	PSIA		01,910					
TEMPERA	ATURE	F		90					
LINE LOS		PSI	(2)						
OTHER L		PSI	(1, 2)		1				
CONTING		PSI							
DISCHARGE C	CONDITIONS								
DISCH. P	RESSURE	PSIA		20					
	EMPERATURE	F	(2)	157					
	FACTOR @ DISCH.			0.999					
DELIVER		PSIA						-	
LINE LOS		PSI	(2)						
	GER LOSS	PSI	(2)						
HEATER		PSI	(2)						
OTHER L	L VALVE LOSS	PSI	(2)						
		PSI PSI	(2)						
TOTAL LO	OSSES	PSI	(2)	1.36					
TOTAL LO	OSSES		(2)	1.36					
TOTAL LO	OSSES			1.36 0.75 1800					
TOTAL LO COMPRESSIO EFFICIENCY	OSSES ON RATIO		(2)	0.75					
TOTAL LO COMPRESSIO EFFICIENCY BHP	OSSES IN RATIO R TYPE		(2)	0.75					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI	(2)	0.75 1800					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O	(2)	0.75 1800 3.1					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub>	(2)	0.75 1800 3.1 20.3					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub>	(2)	0.75 1800 3.1 20.3					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE	OSSES IN RATIO R TYPE	PSI H <sub>2</sub> O O <sub>2</sub> Ar	(2)	0.75 1800 3.1 20.3 0.9					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOR DRIVER TYPE GAS COMPOS	OSSES ON RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub>	(2) (2) (2)	0.75 1800 3.1 20.3 0.9 75.7					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER	L DESIGN				
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI	0.75 1800 3.1 20.3 0.9 75.7 NUBBER					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR ED AND MU	(2) (2) (2) DISCHARGE SI JST BE VERIFIE	0.75 1800 3.1 20.3 0.9 75.7 UBBER D BY FINAL MECHANICA					
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO R TYPE SITION: Vol. %	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR	(2) (2) (2) DISCHARGE SI JST BE VERIFIE	0.75 1800 3.1 20.3 0.9 75.7 NUBBER	L DESIGN	CLIENT			
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR ED AND MU	(2) (2) (2) DISCHARGE SI JST BE VERIFIE	0.75 1800 3.1 20.3 0.9 75.7 UBBER D BY FINAL MECHANICA	PROJ.			ntract ACO-5-440	
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR ED AND MU	(2) (2) (2) DISCHARGE SI JST BE VERIFIE	0.75 1800 3.1 20.3 0.9 75.7 UBBER D BY FINAL MECHANICA	PROJ.		JOB NO NREL Co		227 REV
TOTAL LC COMPRESSIO EFFICIENCY BHP COMPRESSOF DRIVER TYPE GAS COMPOS	OSSES IN RATIO	PSI H <sub>2</sub> O O <sub>2</sub> Ar N <sub>2</sub> JCTION OR ED AND MU	(2) (2) (2) DISCHARGE SI JST BE VERIFIE	0.75 1800 3.1 20.3 0.9 75.7 UBBER D BY FINAL MECHANICA	PROJ.				

COM	IPRESSOR				K-320				
SER	VICE				Flue Gas Blower				
GAS	HANDLED	)			Flue Gas				
NOF	RMAL FLOW	v	SCFM		64,194				
NOF	RMAL FLOW	v	LB/HR		279,800				
	GN FLOW		SCFM						
MOL	<u>wt</u> .				27.58				
C <sub>p</sub> /C <sub>v</sub>			Value		1.365				
			@ F / P\$	SIA	202.5 / 14.3				
500	TION CONI	PRESSURE	PSIA		14.3				
		ACTOR @ SUCTION	FUA		0.9985				
	FLOW AT		ACFM		85,400				
	ORIGIN		PSIA						
	TEMPERA	TURE	F		214				
	LINE LOSS	S	PSI	(2)					
	OTHER LC	DSSES	PSI	(1, 2)					
	CONTING	ENCY	PSI						
								+	
DISC		ONDITIONS			417			├	
	DISCH. PR		PSIA F	(2)	14.7 221			╂────┤─	
		MPERATURE ACTOR @ DISCH.	_ F	(2)	0.9985				
	DELIVERY		PSIA		0.8900			<u> </u>	
	LINE LOSS		PSI	(2)					
	EXCHANG		PSI	(2)					
	HEATER L		PSI	(2)					
	CONTROL	VALVE LOSS	PSI	(2)					
	OTHER LC	DSSES	PSI	(2)					
	CONTING		PSI	(2)					
	TOTAL LO		PSI	(2)					
	<b>IPRESSION</b>	N RATIO			1.03				
				(2)	0.75				
BHP	, IPRESSOR	TVDE		(2)	207				
	VER TYPE								
		TION: Vol. %							
			CO <sub>2</sub>		14.33				
			H₂O		10.93				
			O <sub>2</sub>		1.03				
			Ar		0.73				
			N <sub>2</sub>		72.98				
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		ALLOWANCE FOR SU			BBER Y FINAL MECHANICAL	_ DESIGN			
								-	
	DATE		REVIS	IONS	PROC	PROJ.	CLIENT	-	
NO	DATE		REVIS	IONS	PROC	PROJ.	CLIENT	JOB NO NREL Contract	ACO-5-44027
NO	DATE							JOB NO NREL Contract	ACO-5-44027
NO	DATE	NREL BIO			PROC essure Syngas Case (G				

	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE TION: Vol. % ALLOWANCE FOR SL ULATED IS ESTIMATIO	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	IST BE VERIFIED BY		PROJ.	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002	JOB NO NREL DRAWING NO	Contract ACO	5-44027 REV
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005 0.000001 0.00005 0.000001 0.00002 0.08 ER FINAL MECHANICAL	1.0 1.0 1.1 0.7 1.2 655 1. 30, 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000000	17 75 86 5 08 27 7 92 003 005 0001 002 18	JOB NO NREL		5-44027
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005 0.000001 0.00005 0.000001 0.00002 0.08 ER FINAL MECHANICAL	1.0 1.0 1.1 0.7 1.2 655 1. 30, 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000000	17 75 86 5 08 27 7 92 003 005 0001 002 18			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
COMH COMH COMH COMH COMH COMH COMH COMH	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
COMH COMH COMH COMH COMH COMH COMH COMH	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.0000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
COMH COMH COMH COMH COMH COMH COMH COMH	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
COMH COMH COMH COMH COMH COMH COMH COMH	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LOS TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ACTOR @ DISCH. ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00003 0.00005 0.000001 0.00002 0.08 ER	1.0 1.0 1.1 1.1 0.7 1.2 655 1.1 300 0.2 2. 0.0 0.00 0.0000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.000000 0.00000000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
COMH COMH COMH COMH COMH	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL CONTROL CONTING TOTAL LO PRESSION CIENCY PRESSOR ER TYPE COMPOSI	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS SSES ENCY SSES I RATIO TYPE TION: Vol. %	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005 0.000001 0.00002 0.08	1.0 1.0 1.1 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00 0.000 0.000 0.000 0.000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005 0.00005 0.00001 0.00002	1.0 1.0 1.1 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00 0.000 0.000 0.000 0.000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005 0.00005 0.00001 0.00002	1.0 1.0 1.1 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00 0.000 0.000 0.000 0.000	17 75 86 .1 5 08 27 7 7 02 003 005 0001 002			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.002 0.00003 0.00005 0.000001	1.0 1.0 1.1 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00 0.000 0.000 0.000	17 75 86 .1 5 08 27 7 7 22 003 005 0001			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.00003 0.00005	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00 0.00 0.00	17 75 86 .1 5 08 27 7 7 02 003 005			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02 0.002 0.00003	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0 0.00	17 75 86 .1 5 08 27 7 7 22 003			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70 0.02	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2 2. 0.0	17 75 86 .1 .5 .08 .27 .7 .22			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27 2.70	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2 2.	17 75 86 5 08 27 7			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2	17 75 86 .1 5 08 27			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08 0.27	1.0 1.0 1.1 0.7 1.2 65 1. 30. 0.2	17 75 86 .1 5 08 27			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50 30.08	1.0 1.0 1.1 0.7 1.2 65 1. 30.	17 75 86 .1 5 08			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10 1.50	1.0 1.0 1.1 0.7 1.2 65 1.	17 75 86 .1 5			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102 65.10	1.0 1.0 1.1 0.7 1,2 65	17 75 86			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75 7,102	1.0 1.1 1.1 0.7 1.2	17 75 86			
DISCI DISCI C C C C C C C C C C C C C C C C C C	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION CIENCY PRESSOR ER TYPE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY SSES I RATIO TYPE	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75	1.0 1.1 1.1	17 75			
DISCI DISCI COMP COMP COMP	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LO CONTINGE TOTAL LO PRESSION	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS VALVE LOSS SSES SSES SSES I RATIO	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75	1.0 1.1 1.1	17 75			
DISCI DISCI COMP COMP EFFIC BHP	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LO OTHER LO OTHER LO PRESSION CIENCY	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS VALVE LOSS SSES SSES SSES I RATIO	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49 0.75	1.0 1.1 1.1	17 75			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO PRESSION	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DISSES ENCY SSES	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2) (2)	1.022 2.49	1.0	17			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE TOTAL LO	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DISSES ENCY SSES	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2)	1.022	1.0				
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC CONTINGE	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES ENCY	F PSIA PSI PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2) (2) (2)			26			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL OTHER LC	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS DSSES	F PSIA PSI PSI PSI PSI PSI	(2) (2) (2) (2) (2) (2)			26			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L CONTROL	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS VALVE LOSS	F PSIA PSI PSI PSI PSI	(2) (2) (2) (2) (2)			26			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG HEATER L	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS OSS	F PSIA PSI PSI PSI	(2) (2) (2)			26			
	DISCH. TE COMPR. F DELIVERY LINE LOSS EXCHANG	ESSURE MPERATURE ACTOR @ DISCH. S ER LOSS	F PSIA PSI PSI	(2) (2)			26			
	DISCH. TE COMPR. F DELIVERY LINE LOSS	EESSURE MPERATURE ACTOR @ DISCH.	F PSIA PSI	(2)			26			
	DISCH. TE COMPR. F. DELIVERY	ESSURE MPERATURE ACTOR @ DISCH.	F PSIA				26			
DISCI	DISCH. TE COMPR. F	ESSURE MPERATURE ACTOR @ DISCH.	F	(2)			26			
DISCI	DISCH. TE	ESSURE MPERATURE		(2)			26			
DISCI		ESSURE		(2)	334.8	240		1		
DISCI					.,	,.				
(			PSIA		1,000	1,1	65			
	HARGE CO	ONDITIONS						İ		
								1		
(	CONTINGE		PSI	(., -)						
	OTHER LC		PSI	(1, 2)				1		
	LINE LOSS		PSI	(2)	110	20				
	TEMPERA	TURE	F		110	20	0	1		
			PSIA		1,007	1,3	00			
	COMPR. F.	ACTOR @ SUCTION	ACFM		1.006	1.0 1,3				
			PSIA		402	99				
	TION CONI									
-			@ F / PS	SIA	110 / 402	200 /	995			
C <sub>p</sub> /C <sub>v</sub>			Value		1.418	1.4				
MOL	WT.				10.99	10.				
DESI	GN FLOW		SCFM							
NOR	MAL FLOW	V	LB/HR		118,500	118,	500			
	MAL FLOW		SCFM		68,247	68,2				
GAS	HANDLED	1			Treated Syngas	Treated				
SERV	/ICE				MeOH Comp-1	MeOH 0	Comp-2			
					K-400A					
сом	PRESSOR	NUMBER				K-40				

SERVIC GAS H/ NORM/ DESIGI MOL W C <sub>p</sub> /C <sub>v</sub> SUCTIC SUCTIC SUCTIC CCC CCC DISCH/ DISCH/ DISCH/ DISCH/ CCC CCC DISCH/ CCC CCC CCC CCC CCC CCC CCC	ANDLED AL FLOW AL FLOW AL FLOW SN FLOW NT. ION CONDITIONS SUCTION PRESSURE COMPR. FACTOR @ SUCTION SUCTION PRESSURE COMPR. FACTOR @ SUCTION SUCTION PRESSURE SUCTION SUCTION PRESSURE SUCTION SUCTION PRESSURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUSCH. TEMPERATURE SUSCHANGER LOSS SUSCHANGER LOSS SUSTROL VALVE LOSS SUSTROL VALVE LOSS SUSCHANGER LOSS SUSCH	SCFM LB/HR SCFM Value @ F / PS PSIA PSIA PSI PSI PSI PSIA F PSIA F	SIA (2) (1, 2) (2)	M-501A Steam Turbine - Extraction Stage 1 Steam 110,138 313,600 18.02 1.384 1000 / 1260 1260 0.9334 3,369 1000 1000 460 758	M-501 Steam Tu Extraction 5 Stear 51,97 148,11 18,02 1,35: 758 / 4 460 0.952 3,702 758 	rbine - Stage 2 m 79 00 2 3 3 460 21 9 9			
GAS H/ NORM/ DESIGI MOL W C <sub>p</sub> /C <sub>v</sub> SUCTIC SUCTIC SUCTIC SUCTIC OF TE CCC DISCH/ DI DISCH/ DI CCC CC DISCH/ CCC CCC TC CCC	ANDLED AL FLOW AL FLOW AL FLOW SN FLOW NT. ION CONDITIONS SUCTION PRESSURE COMPR. FACTOR @ SUCTION SUCTION PRESSURE COMPR. FACTOR @ SUCTION SUCTION PRESSURE SUCTION SUCTION PRESSURE SUCTION SUCTION PRESSURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUCH. TEMPERATURE SUSCH. TEMPERATURE SUSCHANGER LOSS SUSCHANGER LOSS SUSTROL VALVE LOSS SUSTROL VALVE LOSS SUSCHANGER LOSS SUSCH	LB/HR SCFM Value @ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F	(2) (1, 2)	Extraction Stage 1 Steam 110,138 313,600 18.02 1.384 1000 / 1260 1260 0.9334 3,369 1000 1000 460	Extraction 3 Stear 51,97 148,11 18.0: 1.35: 758 / 4 460 0.952 3,70: 758	Stage 2 n '9 00 2 2 3 3 460 21 9 9			
NORM/           NORM/           DESIGI           MOL           DESIGI           MOL           SUCTION           SUCTION           SUCTION           SUCTION           SUCTION           SUCTION           SUCTION           DISCH/           DIS	IAL FLOW IAL FLOW IAL FLOW SN FLOW NT. ION CONDITIONS UUCTION PRESSURE COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION INE LOSS INTHER LOSSES CONTINGENCY INE LOSS INE LOSS INE LOSS INE LOSS INE LOSS INTROL VALVE LOSS	LB/HR SCFM Value @ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F	(2) (1, 2)	Steam           110,138           313,600           18.02           1.384           1000 / 1260           1260           0.9334           3,369           1000           460	Stear           51,97           148,11           1.35:           758 / 4           460           0.952           3,70*           758	n			
NORM/ DESIGI MOL W C <sub>p</sub> /C <sub>v</sub> SUCTIC CC CC FL CC CC FL CC CC DISCH/ DISCH/ DISCH/ DISCH/ CC CC CC CC CC CC CC CC CC C	IAL FLOW SN FLOW NT. ION CONDITIONS UCTION PRESSURE COMPR. FACTOR @ SUCTION SLOW AT SUCTION COMPR. FACTOR @ SUCTION STOP AT SUCTION DRIGIN EMPERATURE INE LOSS CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS SIXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	LB/HR SCFM Value @ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F	(2) (1, 2)	313,600 18.02 1.384 1000 / 1260 1260 0.9334 3,369 1000 460	148,11 18.0: 1.35: 758 / 4 460 0.952 3,70: 758	00 2 3 460 21 9 9			
	IN FLOW NT. ION CONDITIONS IUCTION PRESSURE COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION INE LOSS DITHER LOSSES CONTINGENCY IARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS INE LOSS IEATER LOSS CONTROL VALVE LOSS DITHER LOSSES	SCFM Value @ F / PS PSIA ACFM PSIA F PSI PSI PSI F PSIA F PSIA F	(2) (1, 2)	313,600 18.02 1.384 1000 / 1260 1260 0.9334 3,369 1000 460	148,11 18.0: 1.35: 758 / 4 460 0.952 3,70: 758	00 2 3 460 21 9 9			
MOL W           Cp/Cv           SUCTIC           SUCTIC           SUCTIC           FL           OI           DISCH/           DISCH/           DI           DISCH/           DI           DI           DI           CC           DI           CC           OT           CC           OT           CC           OT           CC           OT           CC           OT           CC           OT           CC	NT. ION CONDITIONS IUCTION PRESSURE COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION COMPR. FACTOR @ SUCTION INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY JINE LOSS INE LOSS CONTROL VALVE LOSS DTHER LOSSES	Value @ F / PS PSIA PSIA F PSIA PSI PSI PSIA F PSIA F SIA PSIA	(2) (1, 2)	1.384 1000 / 1260 1260 0.9334 3,369 1000 460	1.35: 758 / 4 460 0.952 3,70: 758	3 460 21 9 3			
C <sub>p</sub> /C <sub>v</sub> SUCTIC SUCTIC SUC CC OF CC DISCH/ DI CC CC DISCH/ DI CC CC CC CC CC CC CC CC CC C	ION CONDITIONS UCTION PRESSURE COMPR. FACTOR @ SUCTION SLOW AT SUCTION PRIGIN TEMPERATURE INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS INE LOSS CONTROL VALVE LOSS DTHER LOSSES	@ F / PS PSIA PSIA F PSI PSI PSI F PSIA F PSIA F	(2) (1, 2)	1.384 1000 / 1260 1260 0.9334 3,369 1000 460	1.35: 758 / 4 460 0.952 3,70: 758	3 460 21 9 3			
SUCTIC SUCTIC SL CC FL OF TE LII OT CC CC DISCH/ DI DISCH/ DI CC CC CC CC CC COMPF	SUCTION PRESSURE COMPR. FACTOR @ SUCTION FLOW AT SUCTION PRIGIN REMPERATURE INE LOSS CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS CONTROL VALVE LOSS DOTHER LOSSES	@ F / PS PSIA PSIA F PSI PSI PSI F PSIA F PSIA F	(2) (1, 2)	1000 / 1260 1260 0.9334 3,369 1000 460	758 / 4 460 0.952 3,70 758	9 9			
SUCTIC SUCTIC SL CC FL OF TE LII OT CC CC DISCH/ DI DISCH/ DI CC CC CC CC CC COMPF	SUCTION PRESSURE COMPR. FACTOR @ SUCTION FLOW AT SUCTION PRIGIN REMPERATURE INE LOSS CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS CONTROL VALVE LOSS DOTHER LOSSES	PSIA ACFM PSIA F PSI PSI PSIA F PSIA PSIA	(2) (1, 2)	1260 0.9334 3,369 1000 460	460 0.952 3,70 758	) 21 9			
SL CC FL OF TE CC DISCH/ DI DI CC DISCH/ DI DI CC CC CC COMPF	SUCTION PRESSURE COMPR. FACTOR @ SUCTION FLOW AT SUCTION PRIGIN REMPERATURE INE LOSS CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS CONTROL VALVE LOSS DOTHER LOSSES	ACFM PSIA F PSI PSI PSIA F PSIA PSIA	(1, 2)	0.9334 3,369 1000 460	0.952 3,70: 758	21			
CCC FL OF CCC DISCH/J DI DISCH/J DI DI CCC CCC CCMPF	COMPR. FACTOR @ SUCTION LOW AT SUCTION PRIGIN EMPERATURE INE LOSS DTHER LOSSES CONTINGENCY MARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS CONTROL VALVE LOSS DOTHER LOSSES	ACFM PSIA F PSI PSI PSIA F PSIA PSIA	(1, 2)	0.9334 3,369 1000 460	0.952 3,70: 758	21			
FLU OF TE LII OT OT OT DISCHJ DI DISCHJ DI DI CC CC OT CC COMPF	ELOW AT SUCTION DRIGIN EMPERATURE INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS CONTROL VALVE LOSS DOTHER LOSSES	PSIA F PSI PSI PSIA F PSIA PSIA	(1, 2)	3,369 1000 460	3,70	9			
	DRIGIN EMPERATURE INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSIA F PSI PSI PSIA F PSIA PSIA	(1, 2)	1000 460	758	5			
TE LII OT CC DISCH/ DI DI CC CC OT CC CC COMPF	EMPERATURE INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS CONTROL VALVE LOSS DTHER LOSSES	F PSI PSI PSIA F PSIA PSIA	(1, 2)	460					
	INE LOSS DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS HEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSI PSI PSIA F PSIA PSIA PSI	(1, 2)	460					
	DTHER LOSSES CONTINGENCY HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY JINE LOSS CONTROL VALVE LOSS DOTHER LOSSES	PSI PSIA F PSIA PSIA PSI	(1, 2)		100				
	AARGE CONDITIONS HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY JINE LOSS EXCHANGER LOSS HEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSIA PSIA F PSIA PSIA			100				
	HARGE CONDITIONS DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS XCHANGER LOSS HEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSIA F PSIA PSI	(2)		100				
DI DI CC DE LII ED HE CC CC OT CC COMPE	DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS HEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	F PSIA PSI	(2)		100				
DI DI CC DE LII ED HE CC CC OT CC COMPE	DISCH. PRESSURE DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS HEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	F PSIA PSI	(2)		100				
DI CC DE E) HE CC CO TC COMPF	DISCH. TEMPERATURE COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	F PSIA PSI	(2)		100			1	
	COMPR. FACTOR @ DISCH. DELIVERY INE LOSS EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSIA PSI	(2)	/58					
DE LII E> HE CO OT COMPE	DELIVERY INE LOSS EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSI		0.0501	472				
LII EX HE CC OT CC COMPF	INE LOSS EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	PSI		0.9521	0.974	4			
E) HE C( O) C( COMPF	EXCHANGER LOSS IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES		(2)	<u> </u>					
HE CC OT CC TC COMPF	IEATER LOSS CONTROL VALVE LOSS DTHER LOSSES	DCI	(2)	ł					
CC OT CC TC COMPF	ONTROL VALVE LOSS	PSI	(2)						
OT CC TC COMPF	THER LOSSES	PSI PSI	(2)						
CC TC COMPF			(2)						
TC COMPF	ONTINCENCY	PSI PSI	(2)						
COMPR	ONTINGENCY	PSI	(2)						
	PRESSION RATIO	101	(2)	-	-				
LIIIOI			(2)	0.75	0.75	5			
kW Gei	enerated		(2)	9,341	5,37				
	ne TYPE		(2)	Steam	Stear				
	ER TYPE			Otcum	oteu				
	COMPOSITION: Vol. %								
			H <sub>2</sub>						-
			CO <sub>2</sub>						
			CO						
			H <sub>2</sub> 0	100%	100%	6			
			CH <sub>4</sub>						
			C <sub>2</sub> H <sub>2</sub>						
			$C_2H_4$						
			C <sub>2</sub> H <sub>6</sub>						
			Benzene (C <sub>6</sub> H <sub>6</sub> )						
			Tar (C <sub>10</sub> H <sub>8</sub> )						
			$NH_3$						
			N <sub>2</sub>						
	CLUDES ALLOWANCE FOR SU				DESIGN		I		
NO	DATE	REVISI	ONS	PROC	PROJ.	CLIENT			
					11.00.		JOB NO NREL	Contract ACO	5-44027
							DRAWING NO	SUMMACE ACO-	8-44027 REV
	NREL BIO	MASS GAS	IFICATION: High Pre	essure Syngas Case (G	TI Gasifier)				

Site Location					e Specification		Date	Date		
				SERVICE OF H	IGH PRESSUR	E UNIT S-100				
Inlet Condition	IS			Flow	Viscosity	Density	Molecular Weight (Ave.)	Particle Size (mm) (Stokes' MMD)	Volumetric Flowrate	Temperature
				lb/h	lb/ft-sec	lb/ft3	lb/mole		acfm	۴
Gas Particulate				418,416.00 9,440.00	2.54E-05	0.47800 62.40	21.5	60	14,589.00	1,57
Particulate				9,440.00		02.40		00		
Gas Inlet Pressure				460.00						
Gas Discharge Pres Pressure Drop, Max				455.57						
Design/Test Pressu		1		460.00						
Design Particulate		4		50						
Design Separation	Efficiency at Cu	itpoint (%)		98						
Emery Design Calc	ulations Summa	ary for S-10		e Only)						
Mechanical Sizing		Inside Diam (in)	Uninsulated Outside Diam (in)		ID (in)	OD (in)	Thickness (in)			Overall Heigl (ft)
	In Out	32 24		Upper Shell Inner Tube	58 24	60 26		ASME VIII		2
-	Bottom	24	34	Cone	24	20		ASME VIII		
	<u>A-</u>			Refractory	50		4			
		onent Dat Solids	a Differential			Cyclo	ne Body Mate	rials of Cons	struction	
	Design Temperature (°F)	Removal Flowrate (CFM)	Design Pressure (psig)	Туре	Upper S		Lower Coni			zzles
Rotary Air Lock Level Indicator	1598 1598				Inner Wall Cercast™	Outer Shell MS	Inner Wall Cercast™	Outer Shell MS	Inner Wall Cercast™	Outer Shell MS
Level Indicator	1590				Inner Tube	MO	Cercasi	WI3	Cercasi	MG
					MS					
Vendor/Supplier S	Specifications	and Price	Ouote							
Fisher-Klosterma			QUOLE			(Refer to Ver	I Idor Communie	cations and D	ata Sheets)	
Ryan Bruner, Sale	es Manager									
P.O. Box 11190 Lousville, KY										
Ph: 502-572-4000										
Email: rab@fkinc.	.com									
Recommendation:	Replace S-10	0 and S-10	1 with one (1)	cyclone only:						
One (1) cyclone (2 Design, fabricated,					Interior surface	s to be lined v	with 4" of Vesus	ius Cercast	3300 castable	refractory
1-1/4" plate carbor			IT AONIE VC33		All welding per					Ciraciory
Dust receiver secti					Exterior to be s			high tempera	ature aluminun	n paint
Inlet transition to 2		lange			Design pressur		460			
30"Ø verticle gas Approximate Overa	Ŭ		<mark>5 ft∅ x 25 ft</mark>	tall	Design Tempe	rature (F)	650			
Gas Conditions a					nditions at Inl					
Volume per cylon Density (Ibm/ft3)	e (acfm)	14,589 0,478		Specific Grav Dust Loading		<u>1.000</u> 31.3				
Viscosity (lbm/ft-	sec)	2.54E-05				01.0				
		00.05			analasi Ci I	- Faulty of Et				
Inlet Velocity (ft/s No load pres. dro		68.39 106.35		Fraction Effic Dia.(microns)	iencies: Stoke Weight %		nciency			
Full load pres. Dr		85.46		2.5	6.11					
				3	15.75					
				<u>3.5</u> 4	<u>21.47</u> 27.4					
				4.5	33.3					
				5						ļ
				6.5	44.49					
				7.5	58.71					
				<u>8.5</u> 9.5	66.32 72.57					
				9.5	83.53					
				16	89.99					
				23 33	95.08 97.84					
Price (200	5 U.S.\$)	\$ 3	355,000.00		57.84					
Remarks: Inlet an				her-Klostermar	quote for these	four cylones.	Estimated co	st of splitter a	and collection is	\$ \$25,000.
Refer to supplier d										,

Site Location					e Specification		Date			Rev.
				SERVICE OF H	IIGH PRESSUR	E UNIT S-102				
nlet Conditior	าร			Flow	Viscosity	Density	Molecular Weight (Ave.)	Particle Size (mm) (Stokes' MMD)	Volumetric Flowrate	Temperatur
				lb/h	lb/ft-sec	lb/ft3	lb/mole		acfm	۴
Gas Particulate				434,982.00 9,440.00		0.38390 62.40	27.6	60	18,883.00	1,5
uniculate				0,440.00		02.40				
Bas Inlet Pressure Bas Discharge Pre				460.00 455.57						
Pressure Drop, Ma		)		120.00						
Design/Test Pressu Design Particulate				460.00						
Design Separation		utpoint (%)		98						
Emery Design Calc	ulations Summ	any for S 10'	) (for Doforono	o Only)						
Mechanical Sizing		Inside Diam (in)	Uninsulated Outside Diam (in)		ID (in)	OD (in)	Thickness (in)	Designation	Height (In)	Height (ft)
Connections Size	In	32.0769		Upper Shell			1	ASME VIII	160	1:
& Rating	Out Bottom			Inner Tube Cone	32.10		4	ASME VIII		
				Refractory			4			
	Com Design Temperature (°F)	ponent Dat Solids Removal Flowrate (CFM)	a Differential Design Pressure (psig)	Туре	Upper S		ne Body Mate Lower Coni			ozzles
Rotary Air Lock	1598	20.4	15		Inner Wall	Outer Shell	Inner Wall	Outer Shell		Outer Shell
evel Indicator	1598				Cercast™ Inner Tube	MS	Cercast™	MS	Cercast™	MS
					MS					
/endor/Supplier	Specifications	and Price	Quote							
isher-Klosterma	in, Inc					(Refer to Ven	dor Communi	cations and D	ata Sheets)	
Ryan Bruner, Sal P.O. Box 11190	es Manager									
_ousville, KY										
Ph: 502-572-4000 Email: rab@fkinc										
Recommendation:	[									
One (1) cyclone (										
Design, fabricated			n ASME vesse		Interior surface All welding per					refractory
Dust receiver sect	ion with flanged	d discharge			Exterior to be s	andblasted an	nd painted with			n paint
nlet transition to 2	-	flange			Design pressur Design Temper		460 650			
30"Ø verticle gas Approximate Over			<mark>5-1/2 ft∅ x 2</mark>	7 1/2 ft tall	Design Temper	ature (F)	000			
					l					
Gas Conditions a /olume per cylor		18,883		Particulate Co Specific Grav	onditions at Inle	et: 1.000				
Density (lbm/ft3)		0.3839		Dust Loading		6.97				
Viscosity (Ibm/ft-	sec)	2.78E-05								
nlet Velocity (ft/s		69.94			iencies: Stoke		ficiency			
No load pres. dro Full load pres. Dr		83.63 72.52		Dia.(microns) 3	Weight % 7.64					
				4	16.37					
				<u>4.5</u> 5						
				5.5	31.53					
				<u>6</u> 7						
				8	54.2					
				9 10						
				11	72.7					
				14 18						
				25	94.31					
				35						
				80	99.72					
Price (200 Remarks: Inlet an			10,000.00							

					e Specification		Date			Rev.
				SERVICE OF H	IIGH PRESSUR	E UNIT S-103				
nlet Conditior	าร			Flow	Specific Heat	Density	Molecular Weight (Ave.)	Particle Size (mm) (Stokes' MMD)	Volumetric Flowrate	Temperatur
				lb/h	BTU/lb°F	lb/ft3	lb/mole		acfm	۴F
Bas				434,982.00		0.41421	20.14507		16,835.82	1576
Particulate				9,440.00		33.00		60		
as Inlet Pressure	(noio)			460.00						
as Discharge Pre				455.57						
ressure Drop, Ma		)		120.00						
esign/Test Press		/		460.00						
esign Particulate				50						
Design Separation	Efficiency at Cu	utpoint (%)		98						
mery Design Calc	ulations Summ	ary for S-10	6 (for Reference Uninsulated	e Only)						
Mechanical Sizing		Inside Diam (in)	Outside Diam		ID (in)	OD (in)	Thickness (in)	Designation	Height (In)	Height (ft)
			(in)			L	ļ		100	
Connections Size & Rating	In Out	32.0769	42.10	Upper Shell Inner Tube	32.10		1	ASME VIII	160	1:
a naung	Bottom			Cone	32.10			ASME VIII		
			1	Refractory			4			
	Com	ponent Dat				Cyclor	ne Body Mate	rials of Cons	struction	
	Design Temperature (°F)	Solids Removal Flowrate (CFM)	Differential Design Pressure (psig)	Туре	Upper S	ection	Lower Coni	cal section	No	zzles
Rotary Air Lock	1598	20.4	15		Inner Wall	Outer Shell	Inner Wall	Outer Shell	Inner Wall	Outer Shell
evel Indicator	1598				Cercast™ Inner Tube	MS	Cercast™	MS	Cercast™	MS
					MS					
					MO					
/endor/Supplier	Specifications	and Price	Quote							
isher-Klosterma	in, Inc					(Refer to Ven	dor Communio	cations and D	ata Sheets)	
Ryan Bruner, Sal	es Manager									
P.O. Box 11190										
ousville, KY h: 502-572-4000	aut 242									
Email: rab@fkinc										
Recommendation:										
				e.						
						a ta la a lla a dua				etractory
Design, fabricated	, tested, and st	amped as a			Interior surface					
Design, fabricated	, tested, and st	amped as a ction			All welding per	FKI Class 3 p	reocedures wit	h 100% pene	etration	
Design, fabricated -1/4" plate carbor Dust receiver sect	, tested, and st n steel construction with flanged	amped as a ction d discharge			All welding per Exterior to be s	FKI Class 3 p andblasted ar	reocedures wit nd painted with	h 100% pene	etration	
Design, fabricated I-1/4" plate carbor Dust receiver sect nlet transition to 2	, tested, and st n steel construction with flanged 4"Ø gas inlet t	amped as a ction d discharge			All welding per Exterior to be s Design pressur	FKI Class 3 p andblasted ar e (psig)	reocedures wit ad painted with 460	h 100% pene	etration	
Design, fabricated I-1/4" plate carbor Dust receiver sect nlet transition to 2 80"Ø verticle gas	, tested, and st n steel construction with flanged 4"∅ gas inlet t outlet flange	amped as a ction d discharge flange	n ASME vesse		All welding per Exterior to be s	FKI Class 3 p andblasted ar e (psig)	reocedures wit nd painted with	h 100% pene	etration	
Design, fabricated -1/4" plate carbor Dust receiver sect nlet transition to 2 00"Ø verticle gas	, tested, and st n steel construction with flanged 4"∅ gas inlet t outlet flange	amped as a ction d discharge flange			All welding per Exterior to be s Design pressur	FKI Class 3 p andblasted ar e (psig)	reocedures wit ad painted with 460	h 100% pene	etration	
Design, fabricated 1-1/4" plate carbor Dust receiver sect nlet transition to 2 800"Ø verticle gas Approximate Over Bas Conditions a	, tested, and st in steel construc- ion with flanged 4"∅ gas inlet f outlet flange all Dimensions t Inlet:	amped as a ction d discharge flange	n ASME vesse	el tall	All welding per Exterior to be s Design pressur	FKI Class 3 p andblasted ar e (psig) rature (F)	reocedures wit ad painted with 460	h 100% pene	etration	
Design, fabricated -1/4" plate carbor Just receiver sect nlet transition to 2 80" Ø verticle gas Approximate Over Gas Conditions a /olume per cylor	, tested, and st n steel construc- ion with flange 4"Ø gas inlet f outlet flange all Dimensions t Inlet: ne (acfm)	amped as a ction d discharge flange :	n ASME vesse	el tall Particulate Co Specific Grav	All welding per Exterior to be s Design pressur Design Temper Design Temper onditions at Inle	FKI Class 3 p andblasted ar e (psig) rature (F) et: 1.000	reocedures wit ad painted with 460	h 100% pene	etration	
Dne (1) cyclone ( Design, fabricated 1-1/4" plate carbor Dust receiver sect inlet transition to 2 30 <sup>™</sup> Ø verticle gas Approximate Over Gas Conditions a Volume per cylor Density (lbm/14)	, tested, and st a steel construc- ion with flanged 4"∅ gas inlet flange all Dimensions it Inlet: he (acfm)	amped as a ction d discharge flange 8,223 0.5679	n ASME vesse	el tall Particulate Co	All welding per Exterior to be s Design pressur Design Temper Design Temper onditions at Inle	FKI Class 3 p andblasted ar e (psig) rature (F)	reocedures wit ad painted with 460	h 100% pene	etration	
Design, fabricated -1/4" plate carbor Dust receiver sect nlet transition to 2 00" Ø verticle gas Approximate Over Gas Conditions a folume per cylor Density (Ibm/ft3)	, tested, and st a steel construc- ion with flanged 4"∅ gas inlet flange all Dimensions it Inlet: he (acfm)	amped as a ction d discharge flange :	n ASME vesse	el tall Particulate Co Specific Grav	All welding per Exterior to be s Design pressur Design Temper Design Temper onditions at Inle	FKI Class 3 p andblasted ar e (psig) rature (F) et: 1.000	reocedures wit ad painted with 460	h 100% pene	etration	
Design, fabricated -1/4" plate carbor Dust receiver sect nlet transition to 2 80" Ø verticle gas Approximate Over Sas Conditions a /olume per cylor Density (Ibm/ft3) /iscosity (Ibm/ft-	, tested, and st s steel construc- ion with flanged 4" Ø gas inlet f outlet flange all Dimensions it Inlet: te (acfm) sec)	amped as a stion d discharge lange 8,223 0,5679 2,87E-05	n ASME vesse	al tall Particulate Co Specific Grav Dust Loading	All welding per Exterior to be s Design pressur Design Temper onditions at Inle ity (Grains/acf)	FKI Class 3 p andblasted ar e (psig) rature (F) et: 1.000 16	reocedures with ad painted with 460 650	h 100% pene	etration	
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Design, fabricated -1/4" plate carbor Just receiver sect nlet transition to 2 80" Ø verticle gas Approximate Over Gas Conditions a /olume per cylor	, tested, and st s steel construc- ion with flangee all Dimensions tt Inlet: te (acfm) sec) sec) p (in.W.C.)	amped as a tion d discharge lange 8,223 0,5679 2.87E-05 68.53 103.76	n ASME vesse	al tall Particulate Cc Specific Grav Dust Loading Fraction Effic Dia.(microns) 2.5 3.5 4	All welding per Exterior to be s Design pressur Design Temper onditions at Inle ity (Grains/acf) iencies: Stoke Weight % 6.71 15.89 21.16	FKI Class 3 p andblasted ar e (psig) rature (F) et: 1.000 16	reocedures with ad painted with 460 650	h 100% pene	etration	
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Design, fabricated -1/4" plate carbor Dust receiver sect het transition to 2 0"∅ verticle gas pproximate Over Gas Conditions a folume per cylor Density (Ibm/fta) fiscosity (Ibm/fta) fisco	, tested, and st steel construc- ion with flanged and the stress of the stress outlet flange all Dimensions t Inlet: te (acfm) sec) p (in.W.C.) op (in.W.C.) op (in.W.C.)	amped as a stition distribution of the second secon	A SME vessi 4 ft⊘ x 18 ft	el Particulate Cc Specific Grav Dust Loading Fraction Effic Dia.(microns) 2.5 3.5 4.4 4.5 5.5 6 7 8 9 9 100 133 17 24 34	All welding per Exterior to be s Design pressur Design Temper Inditions at Inle ity (Grains/acf)	FKI Class 3 p andblasted ar e (psig) rature (F) et: 1.000 16	reocedures with ad painted with 460 650	h 100% pene	etration	
Pesign, fabricated -1/4" plate carbor Dust receiver sect net transition to 2 0"∅ verticle gas spproximate Over Gas Conditions a folume per cylor Pensity (Ibm/fta) fiscosity (Ibm/fta) fol load pres. Dr iuli load pres. Dr Price (200	, tested, and st steel construction ion with flangeed 4"∅ gas inlet 1 outlet flange all Dimensions t Inlet: te (acfm) sec) pp (in.W.C.) op (in. W.C.) 5 U.S.\$)	amped as a attion discharge discharg	n ASME vesso 4 ft⊘ x 18 ft 	el Particulate Cc Specific Grav Dust Loading Fraction Effic Dia.(microns) 2.5 3.5 4.4 5.5 5.5 4.4 4.5 5.5 6.6 7 8 9 10 13 17 24 34 89 10 10 10 10 10 10 10 10 10 10	All welding per Exterior to be s Design pressur Design Temper Design Temper (Grains/acf) iencies: Stoke Weight % 6.71 15.89 21.16 26.55 31.91 37.11 42.08 51.14 58.96 65.59 71.14 88.89.12 94.37 97.41 99.83	FKI Class 3 p andblasted ar e (psig) ature (F) 1.000 16 s Equiv. % Ef	reocedures with 460 460 650 ficiency	h 100% pene high tempera 	tration ature aluminum	
lesign, fabricated -1/4" plate carbor Just receiver sect let transition to 2 0"∅ verticle gas pproximate Over Gas Conditions a folume per cylor pensity (Ibm/fta) fiscosity (Ibm/fta) niet Velocity (ft/s to load pres. Dr ull load pres. Dr	tested, and st steel construc- ion with flanged all Dimensions tit niet: ne (acfm) sec) p (in.W.C.) op (in.W.C.) 5 U.S.\$) d outlet manifo	amped as a attion discharge lange discharge lange discharge lange discharge lange discharge lange discharge discharg	n ASME vessi 4 ft∅ x 18 ft 	el rail Particulate Cc Specific Grav Dust Loading Fraction Effic Dia.(microns) 2.5 3.5 4 4.55 55 55 66 77 8 9 9 10 13 177 24 4 89 her-Klostermar	All welding per Exterior to be s Design pressur Design Temper (Grains/acf) (Grains/	FKI Class 3 p andblasted ar e (psig) ature (F) 1.000 16 s Equiv. % Ef	reocedures with 460 460 650 ficiency	h 100% pene high tempera 	tration ature aluminum	

## DATA SHEETS, LOW PRESSURE DESIGN

		Н	eat Exchar	nger Speci	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Tar Reformer S	SG Cooling/St	eam Generato	r	Item No	H-100 Tar Ref	Cooler
Size 72x 168		Туре	<b>BEM - HORZ</b>	Connected in	2 Parallel		1 Series
Surf/Unit (Eff)	10708 ft <sup>2</sup>	Shells/Unit	2	Surface/Shel	I (Effective)	5354 ft <sup>2</sup>	
			PERFORMAN	NCE OF ONE	<u>ÙNIT</u>		
Fluid Allocation				Shellside			Tubeside
Fluid Name			Syn	gas fr Tar Ref	ormer	F	Preheated BFW
Total Fluid Entering	q	lb/hr	,	329,000			251,800
Vapor	<u></u>			329,000			0
Liquid				0			251,800
Steam				-			
Noncondensa	ble						
Fluid Vaporized or				0			251.800
Liquid Density (In/		lb/ft <sup>3</sup>		0.000/0.000	)	1	46.533/45.419
Liquid Viscosity		cP		0.000		<u> </u>	0.092
Liquid Specific Hea	at	Btu/lb-F		0.000		<del> </del>	1.636
Liquid Thermal Co	nductivity	Btu/hr-ft-F		0.000		<u> </u>	0.321
Vapor Mol. Weight		Dtu/III-It-F		16.74/16.74			0.0/18.02
Vapor Viscosity	(III/Out)	cP		0.0280	•		0.0200
Vapor Specific Hea	<u>_</u>	Btu/lb-F		0.0280			0.0200
Vapor Thermal Co	,	Btu/hr-ft-F		0.078	0		0.025
Temperature (In/O		°F		1,598.0/624.	0		546.5/575.0
Operating Pressure	<u>e</u>	psi(Abs)		29.900			1,285.000
Velocity		ft/sec		280.241			7.682
Pressure Drop (All		psi		5.000/3.920			5.000/0.977
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.001000			0.005000
Heat Exchanged				mtd (corr)	318.656 °F		
Transfer Rate, Ser	vice	48.9		Clean	80.8 Btu/hr-ft	²-F	
			CONSTRUCT				
			lside	Tubes			Sketch
Docian/Toot Droc		45/		1 250	<u>ر</u>		
		45/		1,350			
Design Temp.	°F	45/ 1700		62			
Design Temp. No. Passes per Sh	°F	1700 1		62	5 6	-	
Design Temp. No. Passes per Sh	°F	1700 1 0.0625		62	5 6		
Design Temp. No. Passes per Sh Corrosion Allow.	°F nell	1700 1	0	62	5 6	•	
Design Temp. No. Passes per Sh Corrosion Allow. Connections	°F nell in	1700 1 0.0625		62 0.062	5 6		
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size &	°F nell in In	1700 1 0.0625 1-33.0		62 0.062 6.0	5 6		
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size &	°F iell in In Out	1700 1 0.0625 1-33.0 1-29.0		62 0.062 6.0 10.0	5 6		
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating	°F iell in In Out	1700 1 0.0625 1-33.0 1-29.0		62 0.062 6.0 10.0	5 6	ft	Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No	°F nell in Out Intermediate 1664	1700 1 0.0625 1-33.0 1-29.0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0	62 0.062 6.0 10.0 0	5 6 5	ft	Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube No Tube Type	°F nell in Out Intermediate 1664	1700 1 0.0625 1-33. 1-29. 0 OD 1.000 in	0	62 0.062 6.0 10.0 0 Thk 0.065	5 6 5	ft	Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube No Tube Type Shell	°F nell In Out Intermediate 1664	1700 1 0.0625 1-33.0 1-29.0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0	62 0.062 6.0 10.0 0 Thk 0.065 Material	5 6 5 Length 14.00		Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube No Tube Type Shell Channel or Bonnet	°F nell In Out Intermediate 1664 F	1700 1 0.0625 1-33.0 1-29.0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover	5 6 5 Length 14.00 er		Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Stationa	°F nell in Out Intermediate 1664 F tary	1700 1 0.0625 1-33.0 1-29.0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov	5 6 5 Length 14.00 er loating		Pitch 1.25000 / 30.0°
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station Floating Head Cov	°F nell in Out Intermediate 1664 F tary	1700 1 0.0625 1-33.0 1-29.0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	D in	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov Tubesheet-Fl	5 6 5 Length 14.00 er loating Protection	INT	Pitch 1.25000 / 30.0° 73.7
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles Cross	°F nell in Out Intermediate 1664 F tary	1700 1 0.0625 1-33. 1-29.0 0 0 0 0 1.000 in 2 LAIN 1.D 72.00 OE	D in	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov Tubesheet-Fl Impingement %Cut 34.7 (/	5 6 5 Length 14.00 er loating Protection	INT	
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station. Floating Head Cov Baffles Cross Baffles-Long	°F nell in Out Intermediate 1664 F tary	1700 1 0.0625 1-33. 1-29.0 0 0 0 0 1.000 in 2 LAIN 1.D 72.00 OE	D in SEG	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov Tubesheet-Fl Impingement	5 6 5 Length 14.00 er loating Protection Area)	INT	
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station: Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	°F nell In Out Intermediate 1664 F t ary er	1700 1 0.0625 1-33. 1-29.0 0 0 0 0 1.000 in 2 LAIN 1.D 72.00 OE	D in	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov Tubesheet-FI Impingement %Cut 34.7 ( <i>i</i> Seal Type	5 6 5 Length 14.00 er loating Protection Area) Type	INT	
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran	°F nell In Out Intermediate 1664 F t ary er	1700 1 0.0625 1-33. 1-29.0 0 0 0 0 1.000 in 2 LAIN 1.D 72.00 OE	D in SEG	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cov Tubesheet-FI Impingement %Cut 34.7 ( <i>i</i> Seal Type Tube-Tubesh	5 6 5 Length 14.00 er loating Protection Area) Type	INT	
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Stations Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint	°F nell in Out Intermediate 1664 F tary rer	1700 1 0.0625 1-33. 1-29. 0 OD 1.000 in PLAIN 1.D 72.00 OE Type VERT-	D in SEG U-Bend	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 34.7 (/ Seal Type Tube-Tubesh Type	5 6 5 Length 14.00 er loating Protection Area) Type neet Joint	INT YES Spacing-cc	73.7
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Stationa Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl	°F nell in Out Intermediate 1664 F tary rer	1700 1 0.0625 1-33. 1-29.0 0 0 0 0 1.000 in 2 LAIN 1.D 72.00 OE	D in SEG U-Bend Bundle Entrar	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 34.7 (/ Seal Type Tube-Tubesh Type	5 6 5 Length 14.00 er loating Protection Area) Type	INT YES Spacing-cc Bundle Exit	
Design Temp. No. Passes per Sh Corrosion Allow. Connections Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Stationä Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl Gasket-Shellside	°F nell in Out Intermediate 1664 F tary rer gement	1700 1 0.0625 1-33. 1-29. 0 OD 1.000 in PLAIN 1.D 72.00 OE Type VERT- 2,611	D in SEG U-Bend Bundle Entrar Tubeside	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 34.7 (/ Seal Type Tube-Tubesh Type	5 6 5 Length 14.00 er loating Protection Area) Type neet Joint	INT YES Spacing-cc Bundle Exit Floating Head	73.7 4,375
Size & Rating Tube No Tube Type Shell Channel or Bonnet Tubesheet-Station Floating Head Cov Baffles-Coss Baffles-Long Supports-Tube Bypass Seal Arran Expansion Joint Rho-V2 Inlet Nozzl	°F nell in Out Intermediate 1664 F tary rer gement	1700 1 0.0625 1-33. 1-29. 0 OD 1.000 in PLAIN 1.D 72.00 OE Type VERT- 2,611	D in SEG U-Bend Bundle Entrar	62 0.062 6.0 10.0 0 Thk 0.065 Material Shell Cover Channel Cover Channel Cover Tubesheet-FI Impingement %Cut 34.7 (/ Seal Type Tube-Tubesh Type ce	5 6 5 Length 14.00 er loating Protection Area) Type neet Joint	INT YES Spacing-cc Bundle Exit	73.7

		Н	eat Exchang	er Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Tar Reformer	SG Cooling/BF	W Preheat		Item No	H-101 Tar Ref	Cooler
Size 57x 168		Туре	BEM - HORZ CO	onnected in	2 Parallel		1 Series
Surf/Unit (Eff)	13334 ft <sup>2</sup>	Shells/Unit	2 St	urface/Shell	(Effective)	6667 ft <sup>2</sup>	
			PERFORMANC				
Fluid Allocation				Shellside			Tubeside
Fluid Name			Synga	s fr Tar Refo	rmer		BFW
Total Fluid Enterin	a	lb/hr	Oyligu	329,000			142,594
Vapor	9	10/11		329,000			0
Liquid				0			142.594
Steam				0			142,594
	- h l e						
Noncondensa							0
Fluid Vaporized or				0			0
Liquid Density (In/	Out)	lb/ft <sup>3</sup>	(	0.000/0.000			58.509/46.533
Liquid Viscosity		cP		0.000			0.139
Liquid Specific He		Btu/lb-F		0.000			1.340
Liquid Thermal Co	,	Btu/hr-ft-F		0.000			0.359
Vapor Mol. Weigh	t (In/Out)			16.74/16.74			0.0/0.0
Vapor Viscosity	· · ·	cP		0.0189			0.0000
Vapor Specific He	at	Btu/lb-F		0.475			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.049			0.000
Temperature (In/C		°F	F	524.0/300.0			240.0/546.5
Operating Pressur		psi(Abs)		26.900			1,295.000
Velocity	0	ft/sec		234.572			1,200.000
Pressure Drop (All	ow/Colo)	psi		5.000/4.568			5.000/0.513
		hr-ft²-F/Btu		0.001000			0.002000
Fouling resistance					00 077 °F		0.002000
Heat Exchanged				td (corr)	68.377 °F	-	
Transfer Rate, Sei	vice	55.5		ean	68.5 Btu/hr-ft	<b>-</b> F	
		<u> </u>	CONSTRUCTIO			-	
			lside	Tubesi			Sketch
Design/Test Pres.		35/		1,360/			
Design Temp.	°F	675		600			
No. Passes per Sl	nell	1		1			
Corrosion Allow.	in	0.0625		0.0625			
Connections	In	1-29.	C	3.0			
Size &	Out	1-29.	)	4.0			
Rating	Intermediate	0	-	0			
		Ţ					
Tube No	2688	OD 0.750 in	Tł	nk 0.065	Length 14.00	ft	Pitch 0.93750 / 30.0°
Tube Type		PLAIN		aterial	_011gt1 14.00	••	
Shell	1	I.D 57.00 OE		nell Cover		INT	
	ł	1.0 JI.00 OL		hannel Cover	r	IINT	
Channel or Ponno	ι		-				
Channel or Bonne	0.00		11	ubesheet-Flo	<u> </u>	YES	
Tubesheet-Station			1			TES .	
Tubesheet-Station Floating Head Cov		T		npingement			75 5
Tubesheet-Station Floating Head Cov Baffles Cross		Type VERT-	SEG %	Cut 37.1 (A		Spacing-cc	75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long		Type VERT-	SEG % Se		rea)		75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	ver	Type VERT-	SEG % Se U-Bend	Cut 37.1 (A eal Type	rea) Type		75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar	ver	Type VERT-	SEG % Se U-Bend	Cut 37.1 (A	rea) Type		75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	ver		SEG % Se U-Bend Tu Ty	Cut 37.1 (A eal Type ube-Tubeshe /pe	rea) Type eet Joint		75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar	ngement	Type VERT- 2,563	SEG % Se U-Bend Tu	Cut 37.1 (A eal Type ube-Tubeshe /pe	rea) Type		75.5
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint	ngement		SEG % Se U-Bend Tu Ty	Cut 37.1 (A eal Type ube-Tubeshe /pe	rea) Type eet Joint	Spacing-cc	3,741
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside	ngement le	2,563	SEG % Se U-Bend Tu Ty Bundle Entrance	Cut 37.1 (A eal Type ube-Tubeshe /pe	rea) Type eet Joint	Spacing-cc Bundle Exit	3,741
Tubesheet-Station Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	ngement	2,563	SEG % St U-Bend Tu Bundle Entrance Tubeside	Cut 37.1 (A eal Type ube-Tubeshe /pe	rea) Type eet Joint	Spacing-cc Bundle Exit Floating Head	3,741

		Н	eat Exchar	nger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	ISE
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Tar Reformer S	G Cooler/Dea	aerator FW Pre	eheat	Item No	H-102 Tar Ref	Cooler
Size 75x 168		Туре	BEM - HORZ				1 Series
Surf/Unit (Eff)	5621 ft²			Surface/Shell		5621 ft²	
			PERFORMAN	ICE OF ONE	UNIT		
-luid Allocation				Shellside			Tubeside
Iuid Name				Syngas fr H-10	)1	Dea	erator Feed Water
Fotal Fluid Enterin	g	lb/hr		329,000			257,000
Vapor	•			329,000			0
Liquid				0			257,000
Steam							
Noncondensa	able						
luid Vaporized or				0		İ	0
iquid Density (In/		lb/ft <sup>3</sup>		0.000/0.000		1	59.592/58.402
iquid Viscosity	-7	cP		0.000		1	0.273
Liquid Specific He	at	Btu/lb-F		0.000		1	1.054
iquid Thermal Co		Btu/hr-ft-F		0.000			0.392
Vapor Mol. Weight				16.74/16.74			0.0/0.0
Vapor Viscosity	(III Out)	сP		0.0156			0.0000
Vapor Specific He	at	Btu/lb-F		0.461			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.040			0.000
Temperature (In/C		°F		300.0/225.0			195.0/237.0
Operating Pressur		psi(Abs)		23.880			30.000
elocity	6	ft/sec		168.427			30.000
Pressure Drop (All	ow/Calc)	psi		5.000/2.790			5.000/0.489
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged				mtd (corr)	44.478 °F		0.002000
Transfer Rate, Ser		45.4		Clean	58.8 Btu/hr-ft	2 🗆	
Transier Rate, Sei	VICE	40.4	CONSTRUCT			-F	
		Shel		Tubes	-	1	Sketch
Design/Test Pres.	nei	35/	ISIUE	45		4	Sketch
Design/Test Ples. Design Temp.	°F	350		280		4	
No. Passes per Sh		350			J 1	4	
Corrosion Allow.	in	0.0625		0.0625			
Connections	In	1-41.0	<u> </u>	6.0	)	4	
Size &		1-41.0	-			4	
	Out	0	J	6.0 0		4	
Rating	Intermediate	0		0			
Tube No	2096	OD 0.750 in		Thk 0.065	Longth 11.00	4	Pitch 1.25000 / 30.0°
Tube No		PLAIN		Material	Length 14.00	π	FIGH 1.20000 / 30.0
Shell	F	I.D 75.00 OD	lin	Shell Cover		INT	
	ł	1.0 13.00 UL	/ 111	Channel Cover	or .	INT	
Channel or Bonne							
Tubesheet-Station				Tubesheet-Fle		YES	
Floating Head Cov				Impingement			91.0
Baffles Cross		Type VERT-	369	%Cut 41.0 (A	(ied)	Spacing-cc	81.9
Baffles-Long			II Bond	Seal Type	Turno		
Supports-Tube			U-Bend	Tube Tube 1	Type		
Bypass Seal Arran	igement			Tube-Tubesh	eet Joint		
Expansion Joint	1-	0.001		Туре	4.074	Descaller 5 11	0.455
Rho-V2 Inlet Nozz	le	2,021	Bundle Entrar	nce	1,271	Bundle Exit	2,155
Gasket-Shellside			Tubeside			Floating Head	
Code Requiremen	t	ASME Sectio					R
Weight/Shell			Filled with Wa	ter		Bundle	
Remarks:				-			

		Н	eat Exchai	nger Speci	fication shee	et	
					Job No.		
Customer N	IREL				Ref No.	LP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit F	lue Gas Cool	er/Steam Sup	erheater		Item No	H-103	
Size 90x 168				Connected in	1 Paralle		1 Series
	770 ft <sup>2</sup>	71	1	Surface/Shell		5770 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE	UNIT		
Fluid Allocation		I		Shellside			Tubeside
Fluid Name			Flu	e Gas fr. Tar F	?eaen	Su	perheated Steam
Total Fluid Entering		lb/hr	110	248,400	logen	00	251,800
Vapor		10/111		248,400			251,800
Liquid				0			0
Steam				0			0
Noncondensab	-		l	0		ł	0
Fluid Vaporized or C		11- 1612	l	0		1	0
Liquid Density (In/Ou	it)	lb/ft <sup>3</sup>	<b> </b>	0.000/0.000			0.000/0.000
Liquid Viscosity		cP	<b></b>	0.000			0.000
Liquid Specific Heat		Btu/lb-F	L	0.000			0.000
_iquid Thermal Conc	,	Btu/hr-ft-F		0.000			0.000
√apor Mol. Weight (I	n/Out)			27.57/27.57			18.02/18.02
Vapor Viscosity		cP		0.0405			0.0254
Vapor Specific Heat		Btu/lb-F		0.313			0.678
Vapor Thermal Conc	luctivity	Btu/hr-ft-F		0.040			0.036
Temperature (In/Out		°F		1,798.0/935.	0		575.0/1,000.0
Operating Pressure	/	psi(Abs)		14.700			1,275.000
Velocity		ft/sec		215.255			5.762
Pressure Drop (Allow	v/Calc)	psi		2.000/1.727			5.000/0.629
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.001000			0.005000
Heat Exchanged 6	7 260 000 Bt		<u> </u>	mtd (corr)	550.248 °F		0.000000
Transfer Rate, Servi		21.2		Clean	25.0 Btu/hr-ft	2 ⊑	
		21.2	CONSTRUC	TION OF ONE		1	
		Shal	lside	Tubes		1	Sketch
Design/Test Dress	oi	30/	Isiue	1,350			Skelch
Design/Test Pres. p		÷ •.					
Design Temp. °		1900		110	-		
No. Passes per Shel		1			1		
Corrosion Allow. ir		0.0625		0.062	5		
Connections Ir		1-57.0		12.0			
	Dut	1-53.0	<u>)</u>	15.0			
Rating Ir	ntermediate	0		0			
Tube No 2	475	OD 0.750 in		Thk 0.065	Length 14.00	ft	Pitch 1.25000 / 45.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 90.00 OD	) in	Shell Cover		INT	
Channel or Bonnet				Channel Cov	er		
Fubesheet-Stationar	V			Tubesheet-Fl	oating		
-loating Head Cover				Impingement	0	YES	
Baffles Cross		Type VERT-	SEG	%Cut 38.4 (A		Spacing-cc	71.2
Baffles-Long				Seal Type			
Supports-Tube			U-Bend		Туре		
Bypass Seal Arrange	ement		C Dona	Tube-Tubesh			
Expansion Joint	Shight						
Rho-V2 Inlet Nozzle		006	Bundle Entrai	Туре	2 220	Pundle Evit	1 020
		906		nce	2,230	Bundle Exit	1,030
Gasket-Shellside		A ON 45 O	Tubeside			Floating Head	
Code Requirement		ASME Sectio	n 8, Divsion 1			TEMA Class	R
Veight/Shell Remarks:			Filled with Wa	ater		Bundle	

		Н	leat Exchai	nger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	NREL Biomass	6			Item No	H-200 Quech	Water Cooler
Size 71x 120		Туре	BEM - HORZ	Connected in			1 Series
Surf/Unit (Eff)	9232 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	9232 ft²	
			PERFORMA	NCE OF ONE	UNIT		
Fluid Allocation				Shellside			Tubeside
Fluid Name				Quench Wate	er		Cooling Water
Total Fluid Enterin	g	lb/hr		1,189,000			1,107,500
Vapor				0			0
Liquid				1,189,000			1,107,500
Steam							
Noncondensa	able						
Fluid Vaporized or	Condensed			0			0
Liquid Density (In/		lb/ft <sup>3</sup>		61.342/61.76	5		62.470/62.000
Liquid Viscosity		cP		0.578			0.744
Liquid Specific Hea	at	Btu/lb-F		1.003			0.998
Liquid Thermal Co		Btu/hr-ft-F		0.366			0.361
Vapor Mol. Weight	(In/Out)			0.0/0.0		1	0.0/0.0
Vapor Viscosity		cP		0.0000			0.0000
Vapor Specific Hea	at	Btu/lb-F		0.000			0.000
Vapor Thermal Co	nductivity	Btu/hr-ft-F		0.000			0.000
Temperature (In/O	ut)	°F		128.0/110.0			80.0/100.0
Operating Pressur	e	psi(Abs)		26.000			20.000
Velocity		ft/sec		1.924			-
Pressure Drop (All	ow/Calc)	psi		5.000/1.722			5.000/0.549
Fouling resistance	,	hr-ft <sup>2</sup> -F/Btu		0.001000			0.002000
Heat Exchanged	22,150,000 Bt			mtd (corr)	28.989 °F		
Transfer Rate, Ser	vice	82.8		Clean	115.4 Btu/hr-	ft²-F	
,			CONSTRUCT	TION OF ONE	SHELL		
		She	lside	Tubes	ide		Sketch
Design/Test Pres.	psi	45/		45	5/	1	
Design Temp.	°F	215		15	0	1	
No. Passes per Sh	nell	1			1		
Corrosion Allow.	in	0.0625		0.062	5		
Connections	In	1-13.	0	12.0		1	
Size &	Out	1-13.	0	12.0			
Rating	Intermediate	0		0			
0	•						
Tube No	4860	OD 0.750 in		Thk 0.065	Length 10.00	ft	Pitch 0.93750 / 30.0°
Tube Type	F	PLAIN		Material	Ţ		
Shell		I.D 71.00 OE	) in	Shell Cover		INT	
Channel or Bonnel	t			Channel Cove	er		
Tubesheet-Station	ary			Tubesheet-Fl	oating		
Floating Head Cov				Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 9.2 (Ar		Spacing-cc	14.1
Baffles-Long		2.		Seal Type	,		
Supports-Tube			U-Bend	<i></i>	Туре		
Bypass Seal Arran	gement		-	Tube-Tubesh			
Expansion Joint	U			Туре	· · · ·		
Rho-V2 Inlet Nozz	e	2,093	Bundle Entra		913	Bundle Exit	2,867
Gasket-Shellside	-	_,	Tubeside		5.0	Floating Head	
Code Requirement	t	ASME Section	n 8, Divsion 1			TEMA Class	
Weight/Shell			Filled with Wa	ater		Bundle	
Remarks:							
tomanto.							

		Н	eat Exchar	nger Specif	ication shee	et	
-					Job No.		
	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
	Compressor In				Item No	H-300A	
Size 82x 144		Туре		Connected in	2 Parallel		1 Series
Surf/Unit (Eff)	28471 ft²	Shells/Unit	2	Surface/Shell	(Effective)	14235 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE	UNIT		
luid Allocation				Shellside			Tubeside
luid Name				Cooling wate	•	1:	st Stage Syngas
otal Fluid Entering	j	lb/hr		6,100,000			317,400
Vapor				0			317,400
Liquid				6,100,000			0
Steam							
Noncondensa	ble						
luid Vaporized or	Condensed			0		1	85,698
iquid Density (In/C		lb/ft <sup>3</sup>		62.000/62.000	)	Ì	0.000/62.020
iquid Viscosity		cP		0.762		İ	0.432
iquid Specific Hea	t	Btu/lb-F		1.000		1	1.035
iquid Thermal Cor		Btu/hr-ft-F		0.363		1	0.380
apor Mol. Weight				0.0/0.0		1	16.7/16.7
apor Viscosity	(	сP		0.0000			0.0157
apor Specific Hea	t	Btu/lb-F		0.000		ł	0.460
apor Thermal Cor		Btu/hr-ft-F		0.000			0.043
emperature (In/Ou		°F		80.0/100.0			344.0/110.0
perating Pressure		psi(Abs)		65.000			35.000
elocity		ft/sec		3.977		ł	39.521
Pressure Drop (Allo	w/Cala)	psi		5.000/4.889		ł	5.000/0.642
ouling resistance	w/Calc)	hr-ft²-F/Btu		0.002000			0.001000
leat Exchanged	100 000 F				80.189 °F		0.001000
				mtd (corr)	64.5 Btu/hr-ft	2 -	
ransfer Rate, Serv	lice	53.4	CONCTRUCT	Clean		F	
		Chal	Iside		-	ı –	Cleater
			Isiae	Tubes		4	Sketch
Design/Test Pres.		80/		50		4	
Design Temp.	°F	150		400		4	
lo. Passes per She		1		,		4	
	in	0.0625		0.0625	)		
	In	1-23.0		25.0			
	Out	1-23.0	0	23.0			
lating	Intermediate	0		0			
						-	
	6298	OD 0.750 in		Thk 0.065	Length 12.00	ft	Pitch 0.93750 / 30.0°
ube Type	F			Material			
shell		I.D 82.00 OE	) in	Shell Cover		INT	
hannel or Bonnet				Channel Cove			
ubesheet-Stationa				Tubesheet-Flo			
loating Head Cove	er 🗌			Impingement		YES	
affles Cross		Type VERT-	SEG	%Cut 12.4 (A	rea)	Spacing-cc	24.0
affles-Long				Seal Type			
upports-Tube			U-Bend		Туре		
ypass Seal Arrang	jement			Tube-Tubesh	eet Joint		
xpansion Joint				Туре			
ho-V2 Inlet Nozzle	è	1,391	Bundle Entrai		1,525	Bundle Exit	2,034
			Tubeside			Floating Head	
Gasket-Shellside Code Requirement		ASME Section	n 8, Divsion 1			TEMA Class	R
Basket-Shellside		ASME Sectio	n 8, Divsion 1 Filled with Wa	ater		TEMA Class Bundle	R

		Н	leat Exchar	nger Speci	fication she	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Compressor In	terstage Cool	ing		Item No	H-300B	
Size 47x 120	•	Туре	BEM - HORZ	Connected in			1 Series
Surf/Unit (Eff)	3435 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		3435 ft <sup>2</sup>	
()			PERFORMA				
Fluid Allocation				Shellside	-	T	Tubeside
Fluid Name			2	nd Stage Syng	nas		Cooling water
Total Fluid Entering	a	lb/hr		232,600	500		1,639,500
Vapor	9			232.600			0
Liquid				0			1,639,500
Steam				0			1,000,000
Noncondensa	blo						
				0		+	0
Fluid Vaporized or Liquid Density (In/0		lb/ft <sup>3</sup>		0.000/0.000			62.000/62.000
	Julj	cP				+	
Liquid Viscosity	<b>.</b> +	Etu/lb-F		0.000		+	0.762
Liquid Specific Hea						+	
Liquid Thermal Co		Btu/hr-ft-F		0.000		<b> </b>	0.363
Vapor Mol. Weight	(In/Out)			16.26/16.26			0.0/0.0
Vapor Viscosity		cP		0.0162			0.0000
Vapor Specific Hea		Btu/lb-F		0.470			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.050			0.000
Temperature (In/O	ut)	°F		350.0/110.0			80.0/100.0
Operating Pressure	e	psi(Abs)		84.000			65.000
Velocity		ft/sec		119.731			1.938
Pressure Drop (All		psi		5.000/3.994			5.000/0.664
Fouling resistance		hr-ft²-F/Btu		0.001000			0.002000
Heat Exchanged	32,790,000 Bt	u/hr		mtd (corr)	103.761 °F		
Transfer Rate, Ser	vice	92.0		Clean	134.2 Btu/hr-	-ft²-F	
			CONSTRUCT	TION OF ONE	SHELL		
		She	Iside	Tubes	side	T	Sketch
Design/Test Pres.	psi	100/		80	)/	1	
Design Temp.	°F	400		15	0	1	
No. Passes per Sh	ell	1			1	1	
Corrosion Allow.	in	0.0625		0.062	5	1	
Connections	In	1-25.	0	15.0	-	-	
Size &	Out	1-23.	-	15.0		-	
Rating	Intermediate	0	•	0		-	
rating	Internediate	0		Ū			
Tube No	1808	OD 0.750 in		Thk 0.065	Length 10.00	) ft	Pitch 0.93750 / 30.0°
Tube Type		PLAIN		Material	Longin 10.00		1 1101 0.007 00 7 00.0
Shell	1	I.D 47.00 OE	) in	Shell Cover		INT	
Channel or Bonnet		1.00 UL		Channel Cover	or	IINT	
Tubesheet-Station				Tubesheet-FI Impingement		YES	
Floating Head Cov	ei						<b>FQ O</b>
Baffles Cross		Type VERT-	356	%Cut 37.2 (A	nea)	Spacing-cc	58.0
Baffles-Long			LI Den i	Seal Type	Turne		
Supports-Tube			U-Bend	<b>T T</b>	Туре		
Bypass Seal Arran	gement			Tube-Tubesh	eet Joint		
Expansion Joint				Туре			
Rho-V2 Inlet Nozz	е	2,286	Bundle Entrar	nce	3,535	Bundle Exit	3,995
Gasket-Shellside			Tubeside			Floating Head	
Code Requirement		ASME Section	on 8, Divsion 1			TEMA Class	R
Weight/Shell			Filled with Wa	ater		Bundle	
0							

		Н	eat Exchar	nger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	ase
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Compressor In	terstage Cooli			Item No	H-300C	
Size 51x 120		Туре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	4368 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		4368 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE	UNIT		
Fluid Allocation				Shellside			Tubeside
Fluid Name				Cooling water	•	3	rd Stage Syngas
Total Fluid Entering	]	lb/hr		1,384,500			225,800
Vapor				0			225,800
Liquid				1,384,500			0
Steam							
Noncondensa							
Fluid Vaporized or				0			2,710
Liquid Density (In/C		lb/ft³		62.000/62.000	)		0.000/62.250
Liquid Viscosity		cP		0.762			0.558
Liquid Specific Hea		Btu/lb-F		1.000			1.038
Liquid Thermal Cor		Btu/hr-ft-F		0.363			0.368
Vapor Mol. Weight	(In/Out)			0.0/0.0			16.21/16.21
Vapor Viscosity		cP		0.0000			0.0164
Vapor Specific Hea		Btu/lb-F		0.000			0.468
Vapor Thermal Cor	nductivity	Btu/hr-ft-F		0.000			0.051
Temperature (In/O	ut)	°F		80.0/100.0			349.0/110.0
Operating Pressure	;	psi(Abs)		65.000			220.000
Velocity		ft/sec		3.395			26.531
Pressure Drop (Allo	ow/Calc)	psi		5.000/3.694			5.000/0.747
Fouling resistance		hr-ft²-F/Btu		0.002000			0.001000
Heat Exchanged	27,690,000 Bt	u/hr		mtd (corr)	92.157 °F		
Transfer Rate, Serv	/ice	68.8		Clean	88.3 Btu/hr-ft	²-F	
			CONSTRUCT	TION OF ONE	SHELL		
		Shel	lside	Tubesi	ide		Sketch
Design/Test Pres.	psi	80/		245/	/	1	
Design Temp.	°F	150		400	)	1	
No. Passes per Sh	ell	1		1		1	
	in	0.0625		0.0625	5	1	
Connections	In	1-15.0	0	19.0		1	
Size &	Out	1-15.0	C	17.0		1	
Rating	Intermediate	0		0		1	
						•	
Tube No	2350	OD 0.750 in		Thk 0.065	Length 10.00	ft	Pitch 0.93750 / 30.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 51.00 OD	) in	Shell Cover		INT	
Channel or Bonnet				Channel Cove	r		
Tubesheet-Stationa	ary			Tubesheet-Flo	pating		
Floating Head Cove				Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 18.9 (A		Spacing-cc	24.0
Baffles-Long		<i>.</i>		Seal Type	,	1 0	
Supports-Tube			U-Bend	21	Туре		
Bypass Seal Arran	gement		-	Tube-Tubeshe	21		
Expansion Joint	2			Туре	-		
Rho-V2 Inlet Nozzl	e	1,584	Bundle Entra	71	1,413	Bundle Exit	2,336
Gasket-Shellside		-	Tubeside			Floating Head	-
Code Requirement		ASME Section	n 8, Divsion 1			TEMA Class	
Weight/Shell			Filled with Wa	ater		Bundle	

		h	leat Exchar	nger Specif	fication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Compressor In	terstage Cool	ing		Item No	H-300D	
Size 42x 120		Туре	BEM - HORZ	Connected in	1 Paralle		1 Series
Surf/Unit (Eff)	2934 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	2934 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE	ÙNIT <sup>(</sup>		
Iuid Allocation			_	Shellside	-		Tubeside
luid Name				Cooling wate	r	4	th Stage Syngas
otal Fluid Enterin	a	lb/hr		910,500	•		223,200
Vapor	3			0			223,200
Liquid				910,500			0
Steam				510,500			0
Noncondensa	hlo						
				0			670
Iuid Vaporized or		ال <i>ہ \1</i> 43		0	0		670
iquid Density (In/	Jul)	lb/ft <sup>3</sup>		62.000/62.00	0	ł	0.000/62.210
iquid Viscosity	- 1	cP		0.762			0.580
iquid Specific Hea		Btu/lb-F		1.000			1.036
iquid Thermal Co		Btu/hr-ft-F		0.363			0.367
apor Mol. Weight	: (In/Out)			0.0/0.0		ļ	16.2/16.2
/apor Viscosity		cP		0.0000			0.0160
apor Specific Hea		Btu/lb-F		0.000			0.470
/apor Thermal Co		Btu/hr-ft-F		0.000			0.049
emperature (In/O		°F		80.0/100.0			277.0/110.0
<b>Dperating Pressur</b>	e	psi(Abs)		65.000			450.000
/elocity		ft/sec		3.281			17.909
Pressure Drop (All	ow/Calc)	psi		5.000/3.891			5.000/0.750
ouling resistance		hr-ft <sup>2</sup> -F/Btu		0.002000			0.001000
Heat Exchanged	18.210.000 Bt		•	mtd (corr)	79.340 °F	•	
Fransfer Rate, Ser		78.2		Clean	104.5 Btu/hr-	ft²-F	
,			CONSTRUCT	TION OF ONE	SHELL		
		She	Iside	Tubes	ide		Sketch
Design/Test Pres.	psi	80/		500			
Design Temp.	°F	150		33			
lo. Passes per Sh		1			1	-	
Corrosion Allow.	in	0.0625		0.062		-	
Connections	In	1-12.	0	15.0	5		
Size &	Out	1-12.		15.0		-	
	Intermediate	0	0	0		_	
Rating	Internetiate	0		0			
when Nie	4504	00.0750		The 0.005	Length 10.00	ti i	Ditate 0.00750 / 00.08
ube No	1594	OD 0.750 in		Thk 0.065	Length 10.00	π	Pitch 0.93750 / 30.0°
Tube Type	F		<b>)</b> :=	Material			
Shell		I.D 42.00 OI	חו <b>כ</b>	Shell Cover		INT	
Channel or Bonnel				Channel Cove			
ubesheet-Station				Tubesheet-FI			
loating Head Cov	er	_		Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 19.2 (A	Area)	Spacing-cc	19.2
Baffles-Long				Seal Type			
Supports-Tube			U-Bend		Туре		
Bypass Seal Arran	gement			Tube-Tubesh	eet Joint		
xpansion Joint				Туре			
xpansion Joint	le	1,673	Bundle Entrai		1,289	Bundle Exit	2,454
					-	Floating Head	
Rho-V2 Inlet Nozz			lubeside				
Rho-V2 Inlet Nozz Gasket-Shellside	ł	ASME Section	Tubeside on 8. Divsion 1				
Rho-V2 Inlet Nozz	t	ASME Section	Tubeside on 8, Divsion 1 Filled with Wa	ater		TEMA Class Bundle	R

Customer NREL						
Customer NIDEI				Job No.		
Oustomer NREL				Ref No.	LP Syngas Ca	se
Address				Proposal No.		
Plant Location				Date		Rev. 0
Service of Unit ZnO Prehe	ater			Item No	H-420 ZnO Pr	eheater
Size 90x 96	Туре	BEM - HORZ	Connected in	1 Paralle		1 Series
Surf/Unit (Eff) 14480 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	14480 ft <sup>2</sup>	
		PERFORMA	NCE OF ONE	ÛNIT		
Fluid Allocation			Shellside			Tubeside
Fluid Name		Flu	ie Gas fr. Tar F	Regen		Sweet Syngas
Total Fluid Entering	lb/hr		248,400	0		127,000
Vapor			248,400			127,000
Liquid			0			0
Steam			-			
Noncondensable						
Fluid Vaporized or Condensed			0		1	0
Liquid Density (In/Out)	lb/ft <sup>3</sup>		0.000/0.000			0.000/0.000
Liquid Viscosity	cP		0.000		<del> </del>	0.000
Liquid Specific Heat	Btu/lb-F		0.000		<del> </del>	0.000
Liquid Specific Heat	Btu/hr-ft-F		0.000		ł	0.000
Vapor Mol. Weight (In/Out)	Dtu/III-IL-F		27.57/27.57			10.99/10.99
Vapor Viscosity	cP		0.0256			0.0182
	-		0.0256			
Vapor Specific Heat	Btu/lb-F					0.659
Vapor Thermal Conductivity	Btu/hr-ft-F		0.024			0.076
Temperature (In/Out)	°F		935.0/190.0			114.0/750.0
Operating Pressure	psi(Abs)		14.500			440.000
Velocity	ft/sec		-			-
Pressure Drop (Allow/Calc)	psi		1.000/-			5.000/0.287
Fouling resistance	hr-ft²-F/Btu		0.001000			0.001000
Heat Exchanged 52,900,000			mtd (corr)	122.52.15 °F		
Transfer Rate, Service	29.82		Clean	Btu/hr-ft <sup>2</sup> -F		
			TION OF ONE			
	Shel	lside	Tubes	side		Sketch
Design/Test Pres. psi	30/		480			
Design Temp. °F	990		80	0		
No. Passes per Shell	1			1		
Corrosion Allow. in	0.0625		0.062	5		
Connections In	1-35.	0	10.0			
Size & Out	1-31.	0	12.0			
Rating Intermediat	e 0	-	0			
	-					
Tube No 12160	OD 0.750 in		Thk 0.065	Length 8.00 f	ť	Pitch 0.9375 / 30.0°
Tube Type	PLAIN		Material		-	
Shell	I.D 96.00 OE	) in	Shell Cover		INT	
Channel or Bonnet	1.2 00.00 01		Channel Cover	er		
Tubesheet-Stationary			Tubesheet-Fl	-		
Floating Head Cover			Impingement		YES	
Baffles Cross	Type VERT-	SEG	%Cut 49.0 (A		Spacing-cc	50
Baffles-Long	1,90 10111		Seal Type		epacing of	
Supports-Tube		U-Bend	Sour Type	Туре		
Bypass Seal Arrangement		C Dona	Tube-Tubesh			
Expansion Joint						
Rho-V2 Inlet Nozzle		Bundle Entra	Туре		Bundle Exit	
Gasket-Shellside						
Gaskel-Shellside		Tubeside			Floating Head TEMA Class	
						R
Code Requirement	ASME Section					IX
	ASME Section	Filled with W			Bundle	

		Н	eat Exchai	nger Specif	fication shee	et	
					Job No.		
Customer NF	REL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit Zr	nO Syndas C	ooler/BFW Pr	eheat		Item No	H-421	
Size 64x 144		Туре		Connected in			1 Series
Surf/Unit (Eff) 69	915 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		6915 ft <sup>2</sup>	
				NCE OF ONE		001010	
Fluid Allocation				Shellside	0		Tubeside
Fluid Name			S	yngas fr ZnO I	Sed		BFW
Total Fluid Entering		lb/hr	0	127,000	500		114,306
Vapor		10/11		127,000			0
Liquid				0			114,306
Steam				0			114,300
	-						
Noncondensable	-						
Fluid Vaporized or Co		11 16:0		0		<b> </b>	0
Liquid Density (In/Out	t)	lb/ft <sup>3</sup>		0.000/0.000			58.509/46.533
Liquid Viscosity		cP		0.000			0.139
Liquid Specific Heat		Btu/lb-F		0.000			1.340
Liquid Thermal Condu		Btu/hr-ft-F		0.000			0.359
Vapor Mol. Weight (Ir	n/Out)			10.99/10.99			0.0/0.0
Vapor Viscosity		cP		0.0196			0.0000
Vapor Specific Heat		Btu/lb-F		0.660			0.000
Vapor Thermal Condu	uctivitv	Btu/hr-ft-F		0.082		1	0.000
Temperature (In/Out)		°F		750.0/265.0			240.0/546.5
Operating Pressure		psi(Abs)		425.000			1,295.000
Velocity		ft/sec		27.606			1,200.000
Pressure Drop (Allow		psi		5.000/2.034			5.000/0.399
Fouling resistance	/Calc)	hr-ft <sup>2</sup> -F/Btu		0.001000			0.002000
					05 400 °F		0.002000
Heat Exchanged 40				mtd (corr)	85.130 °F	° –	
Transfer Rate, Servic	e	68.9	<u>AANATRUA</u>	Clean	90.2 Btu/hr-ft	<u>-</u> -	
		0		TION OF ONE		1	Olas fals
			lside	Tubes		4	Sketch
Design/Test Pres. ps		470/		1,360			
Design Temp. °F		800		60	-		
No. Passes per Shell		1			1		
Corrosion Allow. in		0.0625		0.062	5		
Connections In		1-15.		4.0			
Size & Ou	ut	1-13.	C	6.0			
Rating Int	termediate	0		0		1	
						•	
		00 0 750 in			Lanath 10.00	ft	Pitch 1.00000 / 30.0°
Tube No 33	364	OD 0.750 in		Thk 0.065	Length 12.00		
	-	PLAIN			Length 12.00		
Tube Type	-	PLAIN	) in	Material	Length 12.00		
Tube Type Shell	-		) in	Material Shell Cover		INT	
Tube Type Shell Channel or Bonnet	F	PLAIN	) in	Material Shell Cover Channel Cover	er		
Tube Type Shell Channel or Bonnet Tubesheet-Stationary	F	PLAIN	) in	Material Shell Cover Channel Cover Tubesheet-Fl	er oating	INT	
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover	F	Plain I.d. 64.00 of		Material Shell Cover Channel Cove Tubesheet-Fl Impingement	er oating Protection	INT	
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross	F	PLAIN		Material Shell Cover Channel Cover Tubesheet-Fl Impingement %Cut 18.6 (A	er oating Protection	INT	24.0
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long	F	Plain I.d. 64.00 of	SEG	Material Shell Cover Channel Cove Tubesheet-Fl Impingement	er oating Protection Area)	INT	
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube	,	Plain I.d. 64.00 of		Material Shell Cover Channel Cover Tubesheet-Fil Impingement %Cut 18.6 (A Seal Type	er oating Protection Area) Type	INT	
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrange	,	Plain I.d. 64.00 of	SEG	Material Shell Cover Channel Cover Tubesheet-Fil Impingement %Cut 18.6 (/ Seal Type Tube-Tubesh	er oating Protection Area) Type	INT	
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrange Expansion Joint	,	PLAIN I.D 64.00 OI Type VERT-	SEG U-Bend	Material Shell Cover Channel Cover Tubesheet-Fil Impingement %Cut 18.6 (A Seal Type Tube-Tubesh Type	er oating Protection Area) Type eet Joint	INT YES Spacing-cc	24.0
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrange Expansion Joint Rho-V2 Inlet Nozzle	,	Plain I.d. 64.00 of	SEG U-Bend Bundle Entra	Material Shell Cover Channel Cover Tubesheet-Fil Impingement %Cut 18.6 (A Seal Type Tube-Tubesh Type	er oating Protection Area) Type	INT YES Spacing-cc Bundle Exit	24.0
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arranger Expansion Joint Rho-V2 Inlet Nozzle Gasket-Shellside	,	2LAIN 1.D 64.00 OI Type VERT- 2,297	SEG U-Bend Bundle Entra Tubeside	Material Shell Cover Channel Cover Tubesheet-Fil Impingement %Cut 18.6 (A Seal Type Tube-Tubesh Type	er oating Protection Area) Type eet Joint	INT YES Spacing-cc Bundle Exit Floating Head	24.0 3,020
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arranger Expansion Joint Rho-V2 Inlet Nozzle Gasket-Shellside Code Requirement	,	2LAIN 1.D 64.00 OI Type VERT- 2,297	SEG U-Bend Bundle Entra Tubeside n 8, Divsion 1	Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 18.6 (/ Seal Type Tube-Tubesh Type Tube-Tubesh	er oating Protection Area) Type eet Joint	INT YES Spacing-cc Bundle Exit Floating Head TEMA Class	24.0
Tube Type Shell Channel or Bonnet Tubesheet-Stationary Floating Head Cover Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arranger Expansion Joint Rho-V2 Inlet Nozzle Gasket-Shellside	,	2LAIN 1.D 64.00 OI Type VERT- 2,297	SEG U-Bend Bundle Entra Tubeside	Material Shell Cover Channel Cover Tubesheet-FI Impingement %Cut 18.6 (/ Seal Type Tube-Tubesh Type Tube-Tubesh	er oating Protection Area) Type eet Joint	INT YES Spacing-cc Bundle Exit Floating Head	24.0 3,020

		Н	eat Exchar	nger Specif	fication shee	ət	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	ZnO Syngas C	ooler			Item No	H-422	
Size 30x 96		Туре	BEM - HORZ	Connected in			1 Series
	1190 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell		1190 ft <sup>2</sup>	
000(2)			PERFORMA				
Fluid Allocation				Shellside		1	Tubeside
Fluid Name				Syngas fr H-42	21		Cooling Water
Total Fluid Entering		lb/hr		127,000		────	593,000
Vapor		10/11		127,000		┢─────	0
Liquid				0			593,000
				0			593,000
Steam	1.						
Noncondensat	-						
Fluid Vaporized or (		11 /6/01		0		<u> </u>	0
Liquid Density (In/O	ut)	lb/ft <sup>3</sup>		0.000/0.000		<b></b>	62.850/62.283
Liquid Viscosity		cP		0.000		<u> </u>	0.734
Liquid Specific Heat		Btu/lb-F		0.000			1.027
Liquid Thermal Con		Btu/hr-ft-F		0.000			0.363
Vapor Mol. Weight	[In/Out)			10.99/10.99			0.0/0.0
Vapor Viscosity		cP		0.0140			0.0000
Vapor Specific Heat		Btu/lb-F		0.645			0.000
Vapor Thermal Con		Btu/hr-ft-F		0.062		1	0.000
Temperature (In/Ou		°F		265.0/120.0			80.0/100.0
Operating Pressure	-/	psi(Abs)		420.000			65.000
Velocity		ft/sec		54,190			1.566
Pressure Drop (Allo	w/Calc)	psi		5.000/4.440			5.000/0.420
Fouling resistance	Wicald)	hr-ft <sup>2</sup> -F/Btu		0.001000		<u> </u>	0.002000
Heat Exchanged	11 960 000 Dt			mtd (corr)	88.210 °F	<u> </u>	0.002000
Transfer Rate, Serv		113.0		Clean	184.6 Btu/hr-	<del>4</del> 2 F	
Transier Rate, Serv	ice	113.0	CONSTRUCT			1( <sup>-</sup> -F	
		Chal	Iside	Tubes			Sketch
Design (Test Dress	:		Isiae			-	Sketch
Design/Test Pres.		465/		80		-	
	Ϋ́F	315		15		_	
No. Passes per She		1			1		
	n	0.0625		0.062	ō		
	n	1-13.0	-	10.0			
	Out	1-12.0	0	10.0			
Rating	ntermediate	0		0			
						-	
Tube No 8	302	OD 0.750 in		Thk 0.065	Length 8.00 f	t	Pitch 0.93750 / 30.0°
Tube Type	F	PLAIN		Material			
Shell		I.D 30.00 OE	) in	Shell Cover		INT	
Channel or Bonnet				Channel Cove	er		
Tubesheet-Stationa	rv			Tubesheet-FI			
Floating Head Cove				Impingement	-	NO	
Baffles Cross	•	Type VERT-	SEG	%Cut 32.3 (A		Spacing-cc	24.0
Baffles-Long			020	Seal Type	u cu j	opaonig-oc	27.0
Supports-Tube			U-Bend	oca Type	Туре		
Bypass Seal Arrang	omont		U-Denu	Tubo Tuboob			
	ement			Tube-Tubesh	eet Juint		
Expansion Joint		0.400		Туре	0.070	Durally 5 11	
Rho-V2 Inlet Nozzle		2,469	Bundle Entrai	nce	3,979	Bundle Exit	4,341
Gasket-Shellside			Tubeside			Floating Head	
Code Requirement		ASME Sectio	n 8, Divsion 1			TEMA Class	R
Weight/Shell Remarks:			Filled with Wa	ater	······	Bundle	

		Н	eat Exchai	nger Specif	ication shee	et	
					Job No.		
Customer NF	EL				Ref No.	LP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit Me	OH Compres	sor Interstag	e Cooling		Item No	H-500A	
Size 24x 72		уре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff) 51		Shells/Unit	1	Surface/Shell	(Effective)	511 ft <sup>2</sup>	
			PERFORMA	NCE OF ONE			
Fluid Allocation				Shellside		1	Tubeside
Fluid Name				Cooling water	r		Syngas
Total Fluid Entering		lb/hr		553.000			127.000
Vapor		10/111		0			127,000
Liquid				553.000			0
Steam				555,000			0
Noncondensable							0
Fluid Vaporized or Co		11 16:0		0		ļ	0
Liquid Density (In/Out		lb/ft <sup>3</sup>		62.000/62.000	J		0.000/0.000
Liquid Viscosity		cP		0.762			0.000
Liquid Specific Heat		Btu/lb-F		1.000		L	0.000
Liquid Thermal Condu		Btu/hr-ft-F		0.363			0.000
Vapor Mol. Weight (In	/Out)			0.0/0.0			10.99/10.99
Vapor Viscosity		cP		0.0000			0.0155
Vapor Specific Heat		Btu/lb-F		0.000			0.655
Vapor Thermal Condu	ctivity	Btu/hr-ft-F		0.000		1	0.068
Temperature (In/Out)	··· <b>·</b>	°F		80.0/100.0			333.0/200.0
Operating Pressure		psi(Abs)		65.000			1,000.000
Velocity		ft/sec		4.182			25.131
Pressure Drop (Allow/	Calc)	psi		5.000/2.552			5.000/0.721
Fouling resistance	Calc)	hr-ft <sup>2</sup> -F/Btu		0.002000		-	0.001000
Heat Exchanged 11	000 000 Dt.				170.297 °F		0.001000
				mtd (corr)		612 F	
Transfer Rate, Service	; 1	27.1	00107010	Clean TION OF ONE	215.4 Btu/hr-	I(*-F	
		011				r	Obstal
<u> </u>			lside	Tubes			Sketch
Design/Test Pres. psi		80/		1,050			
Design Temp. °F		150		385	5		
No. Passes per Shell		1		1			
Corrosion Allow. in		0.0625		0.0625	5		
Connections In		1-10.0	0	12.0			
Size & Ou	t	1-10.0	0	10.0		1	
Rating Int	ermediate	0		0		1	
-							
Tube No 47	<u> </u>	DD 0.750 in		Thk 0.065	Length 6.00 f	t	Pitch 0.93750 / 30.0°
Tube Type		AIN		Material	<b>U</b>		
Shell		D 24.00 OE	) in	Shell Cover		INT	
Channel or Bonnet	1			Channel Cove	٩٢		
Tubesheet-Stationary				Tubesheet-Flo			
Floating Head Cover				Impingement		YES	
Baffles Cross	-		<u> </u>				16.2
		ype VERT-	326	%Cut 23.4 (A	ned)	Spacing-cc	16.3
Baffles-Long				Seal Type	<b>T</b>		
Supports-Tube			U-Bend	<del></del>	Туре		
Bypass Seal Arranger	nent			Tube-Tubeshe	eet Joint		
Expansion Joint				Туре			
Rho-V2 Inlet Nozzle	1	,279	Bundle Entra	nce	1,349	Bundle Exit	2,039
Gasket-Shellside			Tubeside			Floating Head	
	F	SME Sectio	n 8, Divsion 1			TEMA Class	R
Code Requirement							
Weight/Shell			Filled with Wa			Bundle	

		h	leat Exchar	iger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	se
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	MeOH Syngas	Preheat			Item No	H-501	
Size 73x 168		Туре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	12712 ft <sup>2</sup>	Shells/Unit	1	Surface/Shell	(Effective)	12712 ft <sup>2</sup>	
			PERFORMA				
Fluid Allocation				Shellside			Tubeside
Fluid Name				Steam		Svn	gas to MeOH Rxn
Total Fluid Enterin	α	lb/hr		18,830			127,000
Vapor	9			18,830			127,000
Liquid				0			0
Steam				0			8
Noncondensa	ahla						
				10 020			0
Fluid Vaporized or		lb/ft <sup>3</sup>		18,830			0.000/0.000
Liquid Density (In/	Outj						
Liquid Viscosity	-	cP		0.148			0.000
Liquid Specific He		Btu/lb-F		1.120			0.000
Liquid Thermal Co		Btu/hr-ft-F		0.404			0.000
Vapor Mol. Weigh	t (In/Out)			18.02/18.02			10.99/10.99
Vapor Viscosity		cP		0.0161			0.0170
Vapor Specific He		Btu/lb-F		0.492			0.659
Vapor Thermal Co		Btu/hr-ft-F		0.020			0.074
Temperature (In/C		°F		487.0/324.0			239.0/460.0
Operating Pressur	e	psi(Abs)		100.000			1,160.000
Velocity		ft/sec		4.726			2.192
Pressure Drop (Al	low/Calc)	psi		5.000/0.548			5.000/0.492
Fouling resistance	}	hr-ft <sup>2</sup> -F/Btu		0.005000			0.001000
Heat Exchanged	18,450,000 Bt		•	mtd (corr)	60.365 °F		
Transfer Rate, Ser	vice	24.0		Clean	28.3 Btu/hr-ft	²-F	
			CONSTRUCT	TON OF ONE		•	
		She	Iside	Tubes			Sketch
Design/Test Pres.	nsi	100/		1,220			
Design Temp.	°F	540		515			
No. Passes per St		040 1		510			
Corrosion Allow.		0.0625		0.062			
Connections	in In	0.0625	)	10.0	)		
Size &	Out	1-2.0	)	12.0			
Rating	Intermediate	0		0			
Taka Ma	5040	00 0 750 :		The 0.005	1	0	Ditate 0.00750 / 00.00
Tube No	5242	OD 0.750 in		Thk 0.065	Length 14.00	π	Pitch 0.93750 / 30.0°
<b>-</b> - <b>-</b>		PLAIN	<u>_</u> ·	Material			
Tube Type	Г		Jin	Shell Cover		INT	
Shell		I.D 73.00 OI					
Shell Channel or Bonne	t	1.D 73.00 OL		Channel Cove			
Shell Channel or Bonne Tubesheet-Station	t ary	1.D 73.00 OL		Tubesheet-Flo	oating		
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov	t ary			Tubesheet-Fle Impingement	oating Protection	NO	
Shell Channel or Bonne Tubesheet-Station Floating Head Cov Baffles Cross	t ary	Type VERT-	SEG	Tubesheet-Flo Impingement %Cut 10.4 (A	oating Protection	NO Spacing-cc	14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long	t ary			Tubesheet-Fle Impingement	oating Protection		14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long	t ary		SEG U-Bend	Tubesheet-Flo Impingement %Cut 10.4 (A	oating Protection		14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube	t ary /er			Tubesheet-Flo Impingement %Cut 10.4 (A	oating Protection vrea) Type		14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar	t ary /er			Tubesheet-Fle Impingement %Cut 10.4 (A Seal Type Tube-Tubesh	oating Protection vrea) Type		14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cox Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint	t ary /er	Type VERT-	U-Bend	Tubesheet-Fle Impingement %Cut 10.4 (A Seal Type Tube-Tubeshe Type	oating Protection vrea) Type eet Joint	Spacing-cc	14.5
Shell Channel or Bonne Tubesheet-Statior Floating Head Cox Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	t ary /er		U-Bend Bundle Entrar	Tubesheet-Fle Impingement %Cut 10.4 (A Seal Type Tube-Tubeshe Type	oating Protection vrea) Type	Spacing-cc Bundle Exit	
Shell Channel or Bonne Tubesheet-Statior Floating Head Cox Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz Gasket-Shellside	t ary /er ngement	Type VERT- 1,228	U-Bend Bundle Entrar Tubeside	Tubesheet-Fle Impingement %Cut 10.4 (A Seal Type Tube-Tubeshe Type	oating Protection vrea) Type eet Joint	Spacing-cc Bundle Exit Floating Head	1,384
Shell Channel or Bonne Tubesheet-Statior Floating Head Cov Baffles Cross Baffles-Long Supports-Tube Bypass Seal Arrar Expansion Joint Rho-V2 Inlet Nozz	t ary /er ngement	Type VERT- 1,228	U-Bend Bundle Entrar	Tubesheet-Fli Impingement %Cut 10.4 (A Seal Type Tube-Tubesh Type nce	oating Protection vrea) Type eet Joint	Spacing-cc Bundle Exit	1,384

		Н	leat Excha	nger Specif	ication shee	et	
					Job No.		
Customer	NREL				Ref No.	LP Syngas Ca	ise
Address					Proposal No.		
Plant Location					Date		Rev. 0
Service of Unit	Blowdown Coo	ler			Item No	H-601	
Size 12x 48		Туре	BEM - HORZ	Connected in	1 Parallel		1 Series
Surf/Unit (Eff)	89 ft²	Shells/Unit	1	Surface/Shell		89 ft²	
			PERFORMA	NCE OF ONE	UNIT		
Fluid Allocation				Shellside			Tubeside
Fluid Name				Blowdown			Cooling water
Total Fluid Entering	g	lb/hr		3,164			30,465
Vapor				0			0
Liquid				3,164			30,465
Steam							
Noncondensa	able						
Fluid Vaporized or	Condensed			0			0
Liquid Density (In/0		lb/ft <sup>3</sup>		56.607/62.00	0		62.000/62.000
Liquid Viscosity		cP		0.311			0.762
Liquid Specific Hea	at	Btu/lb-F		1.059		1	1.000
Liquid Thermal Co		Btu/hr-ft-F		0.382			0.363
Vapor Mol. Weight	(In/Out)			0.0/0.0			0.0/0.0
Vapor Viscosity	· /	cP		0.0000			0.0000
Vapor Specific Hea	at	Btu/lb-F		0.000			0.000
Vapor Thermal Co		Btu/hr-ft-F		0.000			0.000
Temperature (In/O		°F		298.0/110.0			80.0/100.0
Operating Pressure	e	psi(Abs)		65.000			65.000
Velocity		ft/sec		0.170			0.561
Pressure Drop (All	ow/Calc)	psi		5.000/0.111			5.000/0.536
Fouling resistance		hr-ft <sup>2</sup> -F/Btu		0.001000			0.002000
Heat Exchanged			•	mtd (corr)	89.027 °F	•	
Transfer Rate, Ser		76.9		Clean	104.4 Btu/hr-	ft²-F	
			CONSTRUC	TION OF ONE	SHELL		
		Shel	Iside	Tubes			Sketch
Design/Test Pres.	psi	80/		80			
Design Temp.	°F	350		150			
No. Passes per Sh		1			-		
Corrosion Allow.	in	0.0625		0.062	5		
Connections	In	1-1.0	)	2.0	•		
Size &	Out	1-1.0		2.0			
Rating	Intermediate	0	, 	0			
. wainig	internetate	, v		Ű			
Tube No	116	OD 0.750 in		Thk 0.065	Length 4.00 f	t	Pitch 0.93750 / 30.0°
Tube Type	-	PLAIN		Material	_og.:	•	
Shell	•	I.D 12.00 OE	) in	Shell Cover		INT	
Channel or Bonnet	ł	1.0 12.00 01		Channel Cove	er		
Tubesheet-Station				Tubesheet-Fl			
Floating Head Cov				Impingement		YES	
Baffles Cross		Type VERT-	SEG	%Cut 10.1 (A		Spacing-cc	2.3
Baffles-Long				Seal Type			
Supports-Tube			U-Bend	2001 1990	Туре		
Bypass Seal Arran	gement		e bond	Tube-Tubesh			
Expansion Joint	gement			Type	Coroonit		
Rho-V2 Inlet Nozzl	٥	459	Bundle Entra		10	Bundle Exit	268
Gasket-Shellside		-53	Tubeside		10	Floating Head	
Code Requirement	+	ASME Section	n 8, Divsion 1			TEMA Class	
Weight/Shell	L		Filled with Wa	ator		Bundle	IX
Remarks:						Dunule	
NEIIIdiks.							

				DRAWING NO	REV
				JOB NO NREL Contract ACC	-5-44027
REVISIONS	PROC	PROJ.	CLIENT		
		<u> </u>		4	
		r —	1	1	
D IS ESTIMATED AND MUST BE VERIFIED	BY FINAL MECHANICAL	DESIGN			
ANCE FOR SUCTION OR DISCHARGE SN		I		1 1	
N <sub>2</sub>	75.7	<u> </u>			
Ar	0.9				
O <sub>2</sub>	20.3			<u>├</u> ───	
Vol. % H <sub>2</sub> O	3.1			<u>├</u> ───	
(2)	0.75	<u> </u>			
<b>)</b>	1.36				
PSI (2)					
PSI (2)					
E LOSS PSI (2) PSI (2)					
PSI (2)		<b> </b>			
<b>SS</b> PSI (2)					
PSIA (2)					
PSIA	0.999	<u> </u>			
ATURE F (2)	157				
RE PSIA	20				
ONS					
PSI					
PSI (1, 2)		<b> </b>			
PSI (2)					
F	90				
DN ACFM PSIA	54,910				
	0.999				
URE PSIA	14.7				
s					
Walue @ F / PSIA	<u> </u>				
	28.63				
SCFM					
SCFM LB/HR	51,965 235,200				
	Air				
	Combustion Air				
ER			Combustion Air	Combustion Air	Combustion Air

SCFM LB/HR SCFM Value @ F / PSIA		1 Syngas 120,208	Syngas Compressor Stage 2		Syng Compress	
LB/HR SCFM Value		120,208		3	. 4	
LB/HR SCFM Value			Syngas	Syngas	Syng	gas
SCFM Value			90,448	88,044	87,1	158
Value		317,371	232,617	225,773	223,2	220
					<u> </u>	
		16.7	16.26	16.21	16.	
WF/FSIA		1.36 157 / 15.88	1.374 110 / 30	1.379 110 / 79	1.3 110 /	
	-	1377 13.88	1107 30	110779	1107	215
PSIA		15.88	30	79	21	5
		0.9979	0.999	0.9985	0.99	
ACFM		131,756	48,531	17,936	6,5	
PSIA						
F		157.1	110	110	11	0
	(1, 2)	<b> </b>			<b> </b>	
PSI						
		<u> </u>			<u> </u>	
		35	84	220	A5	50
	(2)				45 27	
	<u>_</u>		1.001	1.003	1.00	
PSIA						
PSI (	(2)					
PSI (	(2)					
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-31 (	<u>,                                    </u>	2 204	2.8	2 78	2.0	93
	(2)		1		1	
		11,248	10,251	10,251	7,0	
	-					
		0.02	0.22	0.23	0.0	
		0	0.00001	0.00002	0.00	
		0	0.000006	0.000006	0.000	0007
		0	0.00001	0.000001	0.000	
		0.01	0.01	0.01	0.0	
					0.0	
	N <sub>2</sub>	0.06	0.08	0.08	0.0	18
	PSIA         PSI         (PSI         (PSI         (PSI         (PSI         (PSIA         PSIA         PSIA <t< th=""><th>PSIA         PSI       (2)         PSI       (1, 2)         PSI       (1, 2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSI (2)       (2)</th><th>ACFM         131,756           PSIA         157.1           PSI         (2)           PSI         (1, 2)           PSI         0           PSI         0.9982           PSIA         35           (2)         344.2           0.9982         0.9982           PSI         (2)           PSI         (2)</th><th>ACFM         131,756         48,531           PSIA         157.1         110           PSI         (2)         110           PSI         (1, 2)         110           PSI         110         110           PSI         21         110           PSI         22         344.2         349.6           PSI         (2)         1001         1001           PSI         (2)         1001         1001           PSI         (2)         110         110           PSI         (2)         110         110           PSI         (2)         110         110           PSI         (2)         1110         110           PSI         (2)         111248         100251           PSI         (2)         112.36         16.42           PSI         CO         18.42         24.48           H2O         27.97         4.28</th><th>ACFM         131,756         48,531         17,936           SIA         157.1         110         110           SI         (2)         157.1         110         110           PSI         (1, 2)         100         110         110           PSI         (1, 2)         100         100         110           PSI         35         84         220         100         1003           PSIA         35         84         220         1001         1.003           PSIA         0.9982         1.001         1.003         1003           PSIA         100         1003         1003         1003           PSI         (2)         1001         1.003         1003           PSI         (2)         1003         1003         1003           PSI         (2)         1003         1003         1003           PSI         (2)         0.75         0.75         0.75</th><th>ACFM         131,756         48,531         17,936         6,5           SIA         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (1, 2)         100         100         111         110         110           PSI         0.9982         1.001         1.003         1.00         1.003         1.00           PSI         0.9982         1.001         1.003         1.00         1.003         1.00           PSI         (2)         100         1003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.01</th></t<>	PSIA         PSI       (2)         PSI       (1, 2)         PSI       (1, 2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSIA       (2)         PSI (2)       (2)	ACFM         131,756           PSIA         157.1           PSI         (2)           PSI         (1, 2)           PSI         0           PSI         0.9982           PSIA         35           (2)         344.2           0.9982         0.9982           PSI         (2)           PSI         (2)	ACFM         131,756         48,531           PSIA         157.1         110           PSI         (2)         110           PSI         (1, 2)         110           PSI         110         110           PSI         21         110           PSI         22         344.2         349.6           PSI         (2)         1001         1001           PSI         (2)         1001         1001           PSI         (2)         110         110           PSI         (2)         110         110           PSI         (2)         110         110           PSI         (2)         1110         110           PSI         (2)         111248         100251           PSI         (2)         112.36         16.42           PSI         CO         18.42         24.48           H2O         27.97         4.28	ACFM         131,756         48,531         17,936           SIA         157.1         110         110           SI         (2)         157.1         110         110           PSI         (1, 2)         100         110         110           PSI         (1, 2)         100         100         110           PSI         35         84         220         100         1003           PSIA         35         84         220         1001         1.003           PSIA         0.9982         1.001         1.003         1003           PSIA         100         1003         1003         1003           PSI         (2)         1001         1.003         1003           PSI         (2)         1003         1003         1003           PSI         (2)         1003         1003         1003           PSI         (2)         0.75         0.75         0.75	ACFM         131,756         48,531         17,936         6,5           SIA         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (2)         157.1         110         110         111           PSI         (1, 2)         100         100         111         110         110           PSI         0.9982         1.001         1.003         1.00         1.003         1.00           PSI         0.9982         1.001         1.003         1.00         1.003         1.00           PSI         (2)         100         1003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.003         1.00         1.01

SERVICE GAS HAI NORMAI DESIGN MOL WT Cp/Cv SUCTION SUC COI FLC ORI FLC ORI TEM DISCHAI DISCHAI DISCHAI DISCHAI DISCHAI DISCHAI EFFICIEI BHP COMPRE EFFICIEI BHP COMPRE COI COI TOT COMPRE	ANDLED AL FLOW AL FLOW N FLOW	ACFM PSIA F PSI PSI PSIA F PSIA PSIA PSI PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2) (2) (2) (2)	K-420           Flue Gas Blower           Flue Gas           56,988           248,400           27.57           1.367           176 / 14.3           0.9982           71,490           175.8           14.7           182           0.9982				
GAS HAI       NORMAI       NORMAI       DESIGN       MOL       WT       Cp/Cv       SUCTION       SUCTION       SUCTION       DISCHAI       DISCHAI       DISCHAI       DISCHAI       DISCHAI       DISCHAI       EEFFICIE       BHP       COMPRE       BHP       COMPRE       GAS CO       INVER       GAS CO	ANDLED AL FLOW AL FLOW AL FLOW T. DN CONDITIONS UCTION PRESSURE DMPR. FACTOR @ SUCTION LOW AT SUCTION COW AND COM AND COM AND COM COM AND COM AND COM COM br>COM AND COM COM COM COM COM COM COM COM COM COM	LB/HR SCFM Value @ F / PS PSIA F PSIA F PSI PSI PSIA F PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSIA PSI PSI	(2) (1, 2) (2) (2) (2) (2)	Flue Gas 56,988 248,400 27.57 1.367 176 / 14.3 14.3 0.9982 71,490 175.8 175.8				
NORMAL           NORMAL           DESIGN           MOL           SUCTION           OTH           CON           DISCHAR           DISCHAR           DISCHAR           DISCHAR           DISCHAR           BIS           COMPRE           BHP           COMPRE           DRIVER           GAS CO           Image: State	AL FLOW AL FLOW AL FLOW N FLOW T. ON CONDITIONS UCTION PRESSURE OMPR. FACTOR @ SUCTION OMPR. FACTOR @ SUCTION OMPR. FACTOR @ SUCTION OMPR. FACTOR @ SUCTION SCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS	LB/HR SCFM Value @ F / PS PSIA F PSIA F PSI PSI PSIA F PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSIA PSI PSI	(2) (1, 2) (2) (2) (2) (2)	56,988           248,400           27.57           1.367           176 / 14.3           0.9982           71,490           175.8           14.7           182				
NORMAL DESIGN MOL WT Cp/Cv SUCTION SUC CON FLC ORI ORI ORI ORI ORI ORI ORI ORI	AL FLOW N FLOW N FLOW T. DN CONDITIONS UCTION PRESSURE OMPR. FACTOR @ SUCTION OWAT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES ONTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS ONTROL VALVE LOSS	LB/HR SCFM Value @ F / PS PSIA F PSIA F PSI PSI PSIA F PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSIA PSI PSI	(2) (1, 2) (2) (2) (2) (2)	248,400 27.57 1.367 176 / 14.3 14.3 0.9982 71,490 175.8 14.7 14.7 182				
DESIGN MOL WT Cp/Cv SUCTO SUC CO CO TEM OR OR OR OR OR OR OR OR OR OR	N FLOW AT. DN CONDITIONS UCTION PRESSURE DMPR. FACTOR @ SUCTION OW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE DMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS DNTROL VALVE LOSS	SCFM Value @ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2)	27.57 1.367 176 / 14.3 14.3 0.9982 71,490 175.8 14.7 14.7 182				
MOL WT C <sub>p</sub> /C <sub>v</sub> SUCTOR SUC SUC COR COR TEM COR TEM COR TEM COR TEM COR COR COR COR COR COR COR COR COR COR	T. ON CONDITIONS UCTION PRESSURE OMPR. FACTOR @ SUCTION OW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES ONTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS	Value @ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F PSIA PSI PSIA PSI PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2)	1.367 176 / 14.3 14.3 0.9982 71,490 175.8 175.8 14.7 14.7 182				
Cop/Cv SUCTION SUC COI FLC ORI TEM LINI OTH COI DISCHAI DISCHAI DISCHAI DISCHAI DISCHAI COI DEL LINI COI COI COI COI COI COI COI CO	DN CONDITIONS JCTION PRESSURE DMPR. FACTOR @ SUCTION LOW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES ONTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE DMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	@ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2)	1.367 176 / 14.3 14.3 0.9982 71,490 175.8 175.8 14.7 14.7 182				
SUCTION SUCTION SUC CON FLC ORI TEM DISCHAR DIS DISCHAR DIS DISCHAR DIS CON DEL LINI EXC CON TOT COMPRE EFFICIEN BHP COMPRE BHP COMPRE COMPRE COMPRE COMPRE COMPRE COMPRE	UCTION PRESSURE DMPR. FACTOR @ SUCTION OW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	@ F / PS PSIA PSIA F PSI PSI PSI PSIA F PSIA F PSIA PSIA PSI PSIA PSI PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2)	176 / 14.3 14.3 0.9982 71,490 175.8 14.7 14.7 182				
SUC COI FLC ORI TEM LINI OTI COI DISCHAI DISCHAI DISCHAI DISCHAI DISCHAI COI DEL LINI EXC HEA COI OTI COMPRE EFFICIEI BHP COMPRE GAS COI DRIVER GAS COI	UCTION PRESSURE DMPR. FACTOR @ SUCTION OW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	PSIA PSIA F PSI PSI PSI PSI F PSIA F PSIA PSI PSI PSI PSI	(2) (1, 2) (2) (2) (2) (2)	14.3 0.9982 71,490 175.8 175.8 14.7 182				
SUC COI FLC ORI TEM LINI OTI COI DISCHAI DISCHAI DISCHAI DISCHAI DISCHAI COI DEL LINI EXC HEA COI OTI COMPRE EFFICIEI BHP COMPRE GAS COI DRIVER GAS COI	UCTION PRESSURE DMPR. FACTOR @ SUCTION OW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	ACFM PSIA F PSI PSI PSI F PSIA F PSIA PSI PSI PSI	(1, 2) (2) (2) (2) (2)	0.9982 71,490 175.8 175.8 14.7 182				
COI FLC ORI TEM LINI OTH COI DISCHAI DIS DIS DIS COI DEL LINI EXC HE/ COI TOT COMPRE BHP COMPRE BHP COMPRE GAS COI	DMPR. FACTOR @ SUCTION LOW AT SUCTION RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE SCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	ACFM PSIA F PSI PSI PSI F PSIA F PSIA PSI PSI PSI	(1, 2) (2) (2) (2) (2)	0.9982 71,490 175.8 175.8 14.7 182				
ORI TEM LINI OTH COI DISCHAI DIS DIS COI DEL LINI EXC HEA COI OTH COMPRE BHP COMPRE BHP COMPRE BHP	RIGIN EMPERATURE NE LOSS THER LOSSES DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	PSIA F PSI PSI PSIA F PSIA PSI PSI PSI PSI	(1, 2) (2) (2) (2) (2)	175.8 175.8 14.7 182				
TEM LINI OTH COI DISCHAI DIS DIS COI DEL LINI EXC COI COMPRE BHP COMPRE BHP COMPRE BHP COMPRE BHP COMPRE	EMPERATURE NE LOSS THER LOSSES ONTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS	F PSI PSI PSIA F PSIA PSI PSI PSI PSI	(1, 2) (2) (2) (2) (2)	14.7 182				
LINI OTH COI DISCHAI DIS DIS COI DEL LINI EXC HEA COMPRE EFFICIE BHP COMPRE BHP COMPRE GAS CO	NE LOSS THER LOSSES ONTINGENCY ARGE CONDITIONS SCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS EATER LOSS	PSI PSI PSIA F PSIA PSIA PSI PSI PSI PSI	(1, 2) (2) (2) (2) (2)	14.7 182				
OTH COI DISCHAI DIS DIS COI DEL LINI EXC COI COMPRE EFFICIEI BHP COMPRE DRIVER GAS COI	THER LOSSES ONTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	PSI PSIA F PSIA F PSIA PSI PSI PSI PSI	(1, 2) (2) (2) (2) (2)	182				
COI DISCHAI DIS DIS COI DEL LINI EXC COI TOT COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	DNTINGENCY ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE DMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	PSIA F PSIA PSIA PSI PSI PSI PSI	(2) (2) (2)	182				
DISCHAI DIS DIS COI DEL LINI EXC HEA COI TOT COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	ARGE CONDITIONS ISCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	PSIA F PSIA PSI PSI PSI PSI	(2) (2)	182				
DIS DIS COI DEL LINI EXC HEA COI TOI COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	SCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	F PSIA PSI PSI PSI PSI	(2) (2)	182				
DIS DIS COI DEL LINI EXC HEA COI TOI COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	SCH. PRESSURE ISCH. TEMPERATURE OMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	F PSIA PSI PSI PSI PSI	(2) (2)	182				
DIS COI DEL LINI EXC COI OTH COMPRE EFFICIEI BHP COMPRE DRIVER GAS COI COMPRE	SCH. TEMPERATURE DMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	F PSIA PSI PSI PSI PSI	(2) (2)	182				<u> </u>
COI DEL LIN EXC COI OTH COMPRE EFFICIEI BHP COMPRE DRIVER GAS COI COMPRE	DMPR. FACTOR @ DISCH. ELIVERY NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	PSIA PSI PSI PSI PSI	(2) (2)					+
DEL LIN EXC COI OTH COMPRE EFFICIEL BHP COMPRE DRIVER GAS CO	ELIVERY NE LOSS KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	PSI PSI PSI PSI	(2)					1
LINI EXC OI COI COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	NE LOSS KCHANGER LOSS EATER LOSS DNTROL VALVE LOSS	PSI PSI PSI PSI	(2)					1
EXC HEA COI OTH COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	KCHANGER LOSS EATER LOSS ONTROL VALVE LOSS	PSI PSI PSI	(2)					
COI OTH COMPRE EFFICIEI BHP COMPRE DRIVER GAS COI	ONTROL VALVE LOSS	PSI	(2)					
OTH COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO								
COI TOT COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	THER LOSSES		(2)					
TOT COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO		PSI	(2)					
COMPRE EFFICIEI BHP COMPRE DRIVER GAS CO	ONTINGENCY	PSI	(2)					
EFFICIEI BHP COMPRE DRIVER GAS CO	OTAL LOSSES	PSI	(2)					
BHP COMPRE DRIVER GAS CO	RESSION RATIO			1.028				
COMPRE DRIVER GAS CO	ENCY		(2)	0.75				
DRIVER GAS CO			(2)	177				
GAS CO								+
(1) INCLU	OMPOSITION: Vol. %							
• •		CO <sub>2</sub>		0.03				
• •		H₂O		3.1				
• •		O <sub>2</sub>		20.29				
• •		Ar		0.91				
• •		N <sub>2</sub>		75.67				
• •								
• •					<u> </u>			<b></b>
• •								┼────
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	LUDES ALLOWANCE FOR SU				DEGION			
(2) VALO	UE TABULATED IS ESTIMAT	ED AND MU	IST BE VERIFIE	D BY FINAL MECHANICAL	LDESIGN			
							4	
						ļ		
NO D			IONS	PROC	PROJ.	CLIENT		
	DATE	REVIS						ntract ACO-5-44027
	DATE	REVIS			CL Gasifier)		DRAWING NO	REV
			IFICATION: Low	Pressure Syngas Case (B				I

CON	IPRESSOR	NUMBER			K-500A	K-5	00B	1		
		NOMBER			MeOH Compressor	1	ompressor			
SER	VICE				Stage 1		ge 2			
GAS	HANDLED				Treated Syngas	Treated	l Syngas			
	RMAL FLOW		SCFM		73,055	73,	055			
	RMAL FLOW	1	LB/HR		127,035	127	,035			
	IGN FLOW		SCFM							
MOL	<u>. wт.</u>				10.99		.99		_	
C <sub>p</sub> /C <sub>v</sub>			Value @ F / PS	14	1.418 115 / 415		424 / 995			
SUC		DITIONS	@1/10		1137413	200	/ 335			
		PRESSURE	PSIA		415	9	95			
		ACTOR @ SUCTION			1.006		021			
	FLOW AT	SUCTION	ACFM		2,881	1,4	400			
	ORIGIN		PSIA							
	TEMPERA		F		110	2	00		_	
	LINE LOSS		PSI	(2)					_	
	OTHER LO		PSI	(1, 2)	<u> </u>					
	CONTINGE		PSI		<u> </u>	<u> </u>				
DISC	CHARGE CO	ONDITIONS								
	DISCH. PR		PSIA		1,000	1.	160			
		MPERATURE	F	(2)	326		9.3			
		ACTOR @ DISCH.			1.023		026			
	DELIVERY		PSIA							
	LINE LOSS	3	PSI	(2)						
	EXCHANG	ER LOSS	PSI	(2)						
	HEATER L		PSI	(2)						
		VALVE LOSS	PSI	(2)						
	OTHER LO		PSI	(2)						
	CONTINGE		PSI PSI	(2)					-	
	IPRESSION		1.01	(2)	2.41	1	.17			
				(2)	0.75		.75			
BHP				(2)	7,377		340			
CON	IPRESSOR	TYPE		••						
DRI\	/ER TYPE									
GAS	COMPOSI	TION: Vol. %								
				H₂	65.45		.45			
				CO <sub>2</sub>	1.63		.63			
				<u>CO</u> H <sub>2</sub> 0	30.3 0.26		0.3 .26			
				CH <sub>4</sub>	1.96		.96		_	
				C <sub>2</sub> H <sub>2</sub>	0.03		.03		-	
				C <sub>2</sub> H <sub>4</sub>	0.28		28			
				C <sub>2</sub> H <sub>6</sub>	0.00002	0.00	0002			
				Benzene (C <sub>6</sub> H <sub>6</sub> )	0.00008	0.00	8000			
				Tar (C <sub>10</sub> H <sub>8</sub> )	0.000001		0001			
				NH <sub>3</sub>	0.01		.01			
				N <sub>2</sub>	0.095	0.0	095		_	
		LLOWANCE FOR SUG			BER ( FINAL MECHANICAL	DESIGN				
NO	DATE		REVISI	ONS	PROC	PROJ.	CLIENT			- / /007
								JOB NO NREL C	untract ACO-	
		NREL BION	MASS GASI	FICATION: Low Pre	ssure Syngas Case (B0	CL Gasifier)		DRAWING NO		REV

M-601A Steam Turbine - Extraction Stage 1 Steam 88,402 251,800 18.02 1.384 1000 / 1265 0.9332 2,691 1000 1000 487 0.977	M-601B Steam Turbine - Extraction Stage 2 Steam 81,815 232,900 18.02 1.336 564.8 / 165 100 0.977 21,390 487 487 50 50 376		
Extraction Stage 1 Steam 88,402 251,800 18.02 1.384 1000 / 1265 0.9332 2,691 2,691 1000 1000 1000 1000 487	Extraction Stage 2 Steam 81,815 232,900 18.02 1.336 564.8 / 165 100 0.977 21,390 487 487	Image: section of the section of t	
Steam 88,402 251,800 18.02 1.384 1000 / 1265 0.9332 2,691 1000 1000 487	<u>Steam</u> 81,815 232,900 18.02 1.336 564.8 / 165 100 0.977 21,390 487 50		
88,402 251,800 18.02 1.384 1000 / 1265 0.9332 2,691 1000 1000 487	81,815 232,900 18.02 1.336 564.8 / 165 100 0.977 21,390 487 487		
251,800 18.02 1.384 1000 / 1265 1265 0.9332 2,691 1000 1000 487	232,900 18.02 1.336 564.8 / 165 100 0.977 21,390 487 50		
1.384 1000 / 1265 0.9332 2,691 1000 100 487	1.336 564.8 / 165 100 0.977 21,390 487 50	-         -           -         -	
1.384 1000 / 1265 0.9332 2,691 1000 100 487	1.336 564.8 / 165 100 0.977 21,390 487 50	Image: set of the set of th	
1000 / 1265 1265 0.9332 2,691 1000 100 487	564.8 / 165 100 0.977 21,390 487 50		
1265 0.9332 2,691 1000 1000 487	100 0.977 21,390 487 50		
0.9332 2,691 1000 1000 100 487	0.977 21,390 487 50		
0.9332 2,691 1000 1000 100 487	0.977 21,390 487 50		
2,691 1000 1000 100 487	21,390 487 50		
1000 100 100 487	487		
100 487	50		
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487			
487			
	376		
0.977			
	0.9833		
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		<u> </u>	
-	-		
0.75	0.75		
16,067	3,343		
Steam	Steam		
		<b>↓</b>	
1000/	1000/		
100%	100%		
		+ +	
		<u>├</u> ───	
ER FINAL MECHANICAL		L 1	
	0.75 16,067 Steam 100%	0.75 0.75 16,067 3,343 Steam Steam 100% 100%	0.75     0.75       16,067     3,343       Steam

Site Location	(Note: Four	(4) paral	el cyclones	5)			Date			Rev.
			SERV	ICE OF LOW F	PRESSURE UN	T S-100 and	S-101			
nlet Conditior	าร			Flow	Viscosity	Density	Molecular Weight (Ave.)	Particle Size (mm) (Stokes' MMD)	Volumetric Flowrate	Temperature
				lb/h	lb/ft-sec	lb/ft3	lb/mole		acfm	۴F
Bas (Split into fou	r parallel flows	)		316,369.00	2.35 x 10-5	0.03500	18.7	<u> </u>	150,652.00	1,59
articulate				40,407.00		62.40		60		
as Inlet Pressure	(psia)			33.00						
as Discharge Pre	ssure (psig)			32.64						
Pressure Drop, Max Design/Test Pressu		)		10.48 33.00						
esign/Test Press				50						
esign Separation		utpoint (%)		98						
mery Design Calc	ulations Summ		Uninsulated	e Only)						
lechanical Sizing		Inside Diam (in)	Outside Diam (in)		ID (in)	OD (in)	Thickness (in)			Overall Heig (ft)
Connections Size & Rating		48		Upper Shell	82	84	1	ASME VIII		
a nating	Out Bottom	36 TBD	46	Inner Tube Cone	36		1	ASME VIII		
				Refractory	74		4			
	Com Design Temperature (°F)	ponent Dat Solids Removal Flowrate (CFM)	a Differential Design Pressure (psig)	Туре	Upper S		ne Body Mate			ozzles
Rotary Air Lock	1598		(psig)		Inner Wall	Outer Shell	Inner Wall	Outer Shell	Inner Wall	Outer Shell
evel Indicator	1598				Cercast™	MS	Cercast™	MS	Cercast™	MS
					Inner Tube MS					
					IVI3					
/endor/Supplier	<b>Specifications</b>	and Price	Quote							
isher-Klosterma						(Refer to Ver	dor Communi	cations and D	ata Sheets)	
yan Bruner, Sal O. Box 11190	es Manager									
ousville, Ky										
h: 502-572-4000										
Email: rab@fkinc	.com									
Recommendation:	Replace S-10	0 and S-10	1 with 4 cvclor	es operated in	parallel using sr	olit air flow:				
our (4) XQ120-4 Design, fabricated					ures: Interior surface	s to be liped y	with 4" of Vosu	ius Coreast	2200 opstable i	rofractory
8/8" plate carbon s			IT AGIVIE VESS		All welding per					renaciony
Oust receiver section	ion with flanged	d discharge			Exterior to be s					n paint
0"Ø gas inlet flar					Design pressur		33			
8"Ø verticle gas					Design Temper	rature (F)	650			
Approximate Over	all Dimensions:		<mark>7 ft⊘ x 35 ft</mark>	tall						
		1								
Bas Conditions a					onditions at Inle					
/olume per cylon Density (Ibm/ft3)	ne (acfm)	37,663 0.035		Specific Grav Dust Loading		<u>1.000</u> 31.3				
/iscosity (lbm/ft-	sec)	2.53E-05		Base Loading		31.3				
nlet Velocity (ft/s lo load pres. dro		78.46		Fraction Effic Dia.(microns)	iencies: Stoke Weight %		ticiency			
full load pres. Dr	op (in. W.C.)	12.0		Dia.(Inicrons)						
				3.5	16.3					
		<b> </b>		4.5	21.44 26.75					
	L			4.5						
				5.5	37.27					
				<u>6.5</u> 7.5	42.27 51.48					
				8.5	51.48					
				9.5	66.29					
		<u> </u>		10.5	71.99					
				<u>13</u> 17	82.36 89.12					
				24	94.36					
				34	97.39					
		1	1	89	99.83					
Drice (200		¢ 44	225 000 00							
Price (200) Remarks: Inlet an			225,000.00	her-Klostermar	a unite for these	four cylones	Estimated co	st of splitter a	and collection is	\$25,000

Site Location							Date			Rev.
			5	SERVICE OF L	OW PRESSURE	E UNIT S-102				
				Flow	Viscosity	Density	Molecular Weight	Particle Size (mm) (Stokes'	Volumetric Flowrate	Temperature
Inlet Condition	າຣ						(Ave.)	MMD)		
Gas				lb/h 328,979.00	BTU/Ib°F 2.78E-05	lb/ft3 0.34470	lb/mole 16.7		acfm 150,612.01	•F 1,59
Particulate				40,407.00	2.702-03	62.40	10.7	60	130,012.01	1,00
Gas Inlet Pressure				33.00						
Gas Discharge Pres Pressure Drop, Max		<u> </u>		32.64 10.00						
Design/Test Pressu		/		33.00						
Design Particulate	Cutpoint			50						
Design Separation	Efficiency at Cu	utpoint (%)		98						
Emery Design Calc	ulations Summ	ary for S-102	(for Reference	Only)						
Mechanical Sizing		Inside Diam (in)	Uninsulated Outside Diam		ID (in)	OD (in)	Thickness (in)	Designation		Overall Heigh (ft)
Connections Size	In	34	(in) 44	Upper Shell	58	60	1	ASME VIII		()
& Rating	Out	26		Inner Tube	34					
	Bottom			Cone			1	ASME VIII		
		ponent Dat		Refractory	50	Cuolo	4 ne Body Mate	rials of Con	truction	
	Design Temperature (°F)	Solids Removal Flowrate	Differential Design Pressure	Туре	Upper S		Lower Coni			ozzles
Rotary Air Lock	1598	(CFM) 20.4	(psig) 15		Inner Wall	Outer Shell	Inner Wall	Outer Shell	Inner Wall	Outer Shell
Level Indicator	1598	20.4	13		Cercast™	MS	Cercast™	MS	Cercast™	MS
					Inner Tube					
					MS					
Vendor/Supplier	Spacifications	and Price (								
Fisher-Klosterma				1		(Refer to Ver	dor Communi	cations and D	ata Sheets)	
Ryan Bruner, Sale						(				
P.O. Box 11190										
Lousville, Ky Ph: 502-572-4000	) oxt 212									
Email: rab@fkinc										
Recommendation:	Quote Pendin	g								
Four (4) XQ120-4	8M cyclone as	semblies ea	ch with the f	l ollowing Featu	Ires.					
Design, fabricated,					Interior surface	s to be lined v	vith 4" of Vesu	vius Cercast	3300 castable	refractory
3/8" plate carbon s					All welding per					
Dust receiver secti		discharge			Exterior to be s				ature aluminum	n paint
40"Ø gas inlet flar					Design pressur		460			
48"Ø verticle gas Approximate Overa			<mark>5 ft∅ x 25 ft t</mark>		Design Temper	rature (F)	650	<u> </u>		
pproximate over										
Gas Conditions a		45.000			onditions at Inle					
Volume per cylon Density (Ibm/ft3)	ie (acim)	15,906 0,3447		Specific Grav Dust Loading		<u>1.000</u> 7.33				
Viscosity (Ibm/ft-	sec)	2.78E-05								
						<b>-</b>				
Inlet Velocity (ft/s No load pres. dro		70.11 73.64		Fraction Effic Dia.(microns)	iencies: Stoke Weight %		liciency			
Full load pres. Dr		63.69		2.5	4.91		1			
•				3.5	12.88					
				4.5	22.89					
				5.5	28.13 33.31		ł			
				6						
				7	47.7					
		<u> </u>		8			ļ			
				9 10			<u> </u>	<u> </u>		
				10						
				13	81.64					
				17						
			1	24						
				34	97.56					
				34 74						
Price (200	5 U.S.\$)	\$ 3	370,000.00							
Price (200 Remarks: Inlet an Refer to supplier d	d outlet manifo	lding is not ir	ncluded in Fish	74 ner-Klosterman	99.67 quote for these	four cylones.	Estimated cos	t of splitter a	nd collection is	\$25,000.

SERVICE OF LOW PRESSURE VMT 5-103           Inite Conditions         Flow         Voscally         Density         Weight Weight (Avs.)         Particle State (Avs.)         Particle (Avs.)         Particle State (Avs.)         Particle (Avs.)         Particle (Avs.) <th< th=""><th>Site Location</th><th></th><th></th><th></th><th></th><th>e Specification</th><th></th><th>Date</th><th></th><th></th><th>Rev.</th></th<>	Site Location					e Specification		Date			Rev.
Flow         Voccasity         Density         Molecular Network         Flow (No.9)         Density         Molecular Network         Energy (No.9)         Summaria         Tem Provide           sas         106         107         Bit Marco         7.288.00			_		SERVICE OF L	OW PRESSUR	E UNIT S-103				
Image: Section of the sectio	nlet Condition	c						Molecular Weight	Size (mm) (Stokes'		Temperatur
as         246.386.00         2.87.00         0.03501         27.6         7.289.00           as load Pressure (pth)         33.00         1.00         60         1.00         60           as load Pressure (pth)         32.00         1.00         60         1.00         60           as load Pressure (pth)         32.00         1.00         1.00         1.00         1.00         1.00           Statistical Statistics Statis Statis Statistics Statistis Statistics Statistics Statis Stat	met conultion	5			lb/h	lb/ft-sec	lb/ft3	lb/mole	MMD)	acfm	°F
sas intel Pressure (psis)         33.00         1         1         1           as Dichtarge Pressure (psis)         32.64         1         1         1           as Dichtarge Pressure (psis)         32.64         1         1         1           besign Particulate Curpoint         50.64         1         1         1           besign Particulate Curpoint (%)         50         1         1         1         1           besign Particulate Curpoint (%)         0         0         0         0         1         1         1         1           besign Particulate Curpoint (%)         0	Gas										1,7
base Discharge Pressure (prig)         32,64	Particulate				40,407.00		1.00		60		
ase Decharge Pressure (psig)         32,64											
Description         Max Allow (*, Mc.)         1000         Image: Comparison of the second											
Design Protective Cupoint         33.00			)								
Design Paraticulate Cutpoint (%)         60         1         1         1           imary Design Catcutations Summary for 5.102 (br) Reference only)         2			/								
Analysis of the second se	Design Particulate	Cutpoint									
Mechanical Stateg         Inside (m)         Uninsulted (m)         D0 (n)         D0 (n)         Thickness (n)         Designation         Over 6 (m)           6. Rating 0 at 16         201         201         201         201         46         48         1         ASME VIII         1           6. Rating 0 at 16         201         10         201         10         1         ASME VIII         1           8. Rating 0 at 10         Southall         Refractory         3.8         1         1         ASME VIII         1           0 at 201         Southall States         Southall States         Type         Upper Soction         Lower Conical soctian         Nezzles           201 ary Air Lock         9.38         20.4         15         Inner Wall         Outer Shell	Design Separation	Efficiency at Cu	itpoint (%)		98						
Mechanical Sizing         Inside (m)         Uninsulated (m)         D0 (n)         D0 (n)         D0 (n)         Thickness (n)         Designation         Over 6 (m)           6 Reting         0:t         26         26         26         46         48         ASME VIII         46           8 Reting         0:t         16         26         16         4         4         4           Bottom         16         20         16         20         46         4         4           Bottom         16         20         17         Refractory         38         Cyclone Body Matrials of Construction         Nezries           Rolary Air Lock         338         20.4         15         Inner Wait         Outer Shell         Inner Wait         Outer Shell         Inner Wait         Nezries           Rolary Air Lock         338         20.4         15         Inner Wait         Outer Shell         Inner Wait         Nezries         Nezries           WeinderStauppiler         20         MS         Cereast <sup>10</sup> MS         Cereast <sup>10</sup> NS	Emony Decign Colo	ulationa Summ	onu for C 10	(for Deference	o Only)						
Metchanical Sizing         Distriction of the second s	mery Design Calc	ulations Summa			e Only)						
Connections Size         In         28         101         36 (upper Shell)         46         48         1         ASME VIII         ASME VIII           Bottom         Conne         18         18         18         1<	Mechanical Sizing			Outside Diam		ID (in)	OD (in)	Thickness (in)	Designation		Overall Heig (ft)
& Rating         Out         18         28 [Inner Tube         18         4         4           Battom         Concount Data         Concount Data         Cyclone Body Materials of Construction         Cyclone Body Materials of Construction         Nozzles           Tensprature         Tensprature         Tensprature         Type         Upper Section         Lower Conical section         Nozzles           Kolary Air Lock         938         Concorasting         Mis         Cercasting         Mis         Cer	Connections Size	In	26		Linner Shell	46	48	1	ASME VIII		. ,
Bottom         Cone         Type         Type         Type         Type         Cyclone Body Materials of Construction           Component Data         Forwards         Provate         Provat								4			
Component Data         Cyclone Body Materials of Construction           Design (F)         Solids Removal (F)         Differential Design (F)         Differential Design (F)         Type         Upper Section         Lower Conical section         Nozzles           Solids ALLOK         938         2.0         15         Inner Wall         Outer Shell         Inner Wall         Inner Wall         Inner Wall         Inner Wall         Inner Wall         Outer Shell         Inner		Bottom						1	ASME VIII		
Design Temperature (°F)         Solids Pressure (°F)         Differential Pressure (°F)         Type         Upper Section         Lower Conical section         Nozzles           Rotary Ar Lock         938         20.4         15         Inner Wall         Outer Shell         Inner Wall		0		L	Refractory	38		4 4			
Image: Temperature (FP)         Removal (PP)         Design (PP)         Type         Upper Section         Lower Conical section         Nozzles           Rotary Air Lock         938         20.4         15         Inner Wall         Outer Shell         Inner Wall         Outer Shel							Cyclo	ne Body Mate	rials of Cons	struction	
Optime         Optim <thoptim< th="">         Optim</thoptim<>		Temperature	Removal Flowrate	Design Pressure	Туре	Upper S	ection	Lower Coni	cal section	No	ozzles
evel Indicator         938         Cercast™         MS         Cercast™         MS         Cercast™         MS           /endor/Supplier Specifications and Price Quote         MS         Imme Tube         MS         Imme Tube         Imme Tu	Rotary Air Lock	938				Inner Wall	Outer Shell	Inner Wall	Outer Shell	Inner Wall	Outer Shell
Vendor/Supplier Specifications and Price Quote       MS       MS         Vendor/Supplier Specifications and Price Quote       (Refer to Vendor Communications and Data Sheets)         Syn Bruner, Sales Manager       0       0         So. Box 11190       0       0         Jones Timer, Sales Manager       0       0         So. Box 11190       0       0         Jones Timer, Sales Manager       0       0         Sole Sort 1190       0       0         Sole Sort 1190       0       0         Sole Sort 2000 ext 213       0       0         Recommendation: Guote Pending       0       0         Second Steel Construction       1       0       0         Design, fabricated, tested, and stamped as an ASME vessel       1       1       0         Jate receiver section with flanged discharge       0       0       0         Design pressure (psig)       460       0       0       0         Sas Conditions at Inter:       0       0       0       0       0         Sas Conditions at Inter:       0       0       0       0       0       0         Sole Sort (Witec)       72.20       Fraction Efficiencies: Stokes Equiv. % Efficiency       0 <t< td=""><td></td><td></td><td>20.1</td><td></td><td></td><td></td><td></td><td></td><td></td><td></td><td></td></t<>			20.1								
fendor/Suppler         Specifications and Price Quote         Image: Control of the second sec											
isher Klosterman, Inc         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 11190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to Vendor Communications and Data Sheets)           20. Box 21190         (Refer to V						MS					
isher Klosterman, Inc.       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. Box 11190       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:       (Refer to Vendor Communications and Data Sheets)         20. State Commendation:	(and an (Orman line)		and Dates								
Syn Bruner, Sales Manager         Image: Sales Manager         Image: Sales Manager         Image: Sales Manager           0.0.sotille, Ky         Image: Sales Manager         Image: Salesalesales Manager         Image: Salesales Man			and Price	Quote			(Refer to Ver	dor Communi	cations and D	(ata Sheets)	
20. Box 11190											
ht: 602-672-4000 ext 213       Image: additional state of the state o											
Enail: rab@(tkinc.com       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)         Recommendation: Quote Pending       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)         Four (4) XO120-45M cyclone assemblies each with the following Features:       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)       Image: rab@(tkinc.com)         Design fabricated, tested, and stamped as an ASME vessel       Interior surfaces to be lined with 4" of Vexuvius Carcasta 3300 castable refracts 3000 castable refracts											
Recommendation:       Quote Pending       Automation       Quote Pending         Four (4) X0120-45M cyclone assemblies each with the following Features:       Interior surfaces to be lined with 4" of Vesuvius Cercast 3300 castable refracts         Design, fabricated, tested, and stamped as an ASME vessel       Interior surfaces to be lined with 4" of Vesuvius Cercast 3300 castable refracts         Dist receiver section with finged discharge       Extentor to be sandblasted and painted with high temperature aluminum paint         D'C gas inleft flange       Design pressure (pis)       460         Alf Welding per FXL Class 3 precodures with 100% penetration       Design pressure (pis)       460         Approximate Overall Dimensions:       4 ftØ x 17 ft tall       Design Cerains at Inlet:       0000         Outcle effmit       7.288       Specific Gravity       1.000       0000         Density (Ibm/ft3)       0.5679       Dust Loading (Grains/acf)       16       0000         No load pres. drop (in.W.C.)       120.63       Dia.(microns)       Weight %       1.000       0000         Full load pres. drop (in.W.C.)       120.63       Dia.(microns)       Weight %       1.000       0000         Full load pres. drop (in.W.C.)       120.63       Dia.(microns)       Weight %       1.000       0000       0000       0000       0000       00000       0											
Four (4) XQ120-45M cyclone assemblies each with the following Features:         Interior surfaces to be lined with 4" of Vesuvus Cercast 3300 castable refract           20% ign, fabricated, tested, and stamped as an ASME vessel         Interior surfaces to be lined with 4" of Vesuvus Cercast 3300 castable refract           20% ign tested construction         All welding per FKI Class 3 preocedures with 100% penetration           20% receiver section with flanged discharge         Design pressure (psig)         460           10° Ø gas inlet flange         Design pressure (psig)         460           Approximate Overall Dimensions:         4 ftØ x 17 ft tall         Interior surfaces (psig)         460           Sas Conditions at Inlet:         Particulate Conditions at Inlet:         Interior (psig)         460           Volcosity (Ibm/ft3)         0.5679         Dust Loading (Grains/acf)         16         Interior (psig)           Viscosity (Ibm/ft3ec)         72.29         Fraction Efficiencies: Stokes Equiv. % Efficiency         Interior (psig)         460           Full load pres. drop (in.W.C.)         120.63         Dia_(microns)         Weight %         Interior (psig)         461           Viscosity (Ibm/ft3ec)         72.29         Fraction Efficiencies: Stokes Equiv. % Efficiency         Interior (psig)         Interior (psig)         Interior (psig)           No load pres. drop (in.W.C.)		com									
Cour (4) XQ120-45M cyclone assemblies each with the following Features:         Interior surfaces to be lined with 4" of Vesuvus Cercast 3300 castable refracts           20esign, fabricated, tested, and stamped as an ASME vessel         Interior surfaces to be lined with 4" of Vesuvus Cercast 3300 castable refracts           20esign, fabricated, tested, and stamped as an ASME vessel         Interior surfaces to be lined with 4" of Vesuvus Cercast 3300 castable refracts           20es intel flange         Design pressure (psig)         460           20es intel flange         Design pressure (psig)         460           4ft@ x 17 ft tall         Design Temperature (F)         650           20es could that the following Features:         1000         1000           20es could that the following Features:         10000         10000           20es co	Recommendation:	Quote Pendin	q								
Design, fabricated, tested, and stamped as an ASME vessel       Interior surfaces to be lined with 4° of Vesuvius Cercast 3300 castable refracts         30° plate carbon steel construction       All velding per FKI Class 3 precedures with 100% penetration         20 store cerver section with flanged discharge       Exterior to be sandblasted and painted with high temperature aluminum paint         10° Ø gas inlet flange       Design pressure (psig)       460         All velding per FKI Class 3 precedures with 100% penetration       Design pressure (psig)       460         All velding per SKI Class 3 precedures with 100% penetration       Design pressure (psig)       460         Approximate Overall Dimensions:       4 ftØ x 17 ft tall       Design Temperature (F)       660         3as Conditions at Inlet:       Particulate Conditions at Inlet:       Image: Conditions at Inlet:       Image: Conditions at Inlet:         Orlow per cylone (acfm)       7.289       Specific Gravity       1.000       Image: Conditions at Inlet:         Interview (ft/sec)       2.87E-05       Image: Conditions at Inlet:       Image: Conditions at Inlet:       Image: Conditions at Inlet:         Interview (ft/sec)       72.29       Fraction Efficiencies: Stokes Equiv. % Efficiency       Image: Conditions at Inlet:         Interview (ft/sec)       72.29       Specific Gravity       Specific Gravity       Image: Conditions at Inlet:											
M8" plate carbon steel construction       All welding per FKI Class 3 procedures with 100% penetration         Dust receiver section with flanged discharge       Exterior to be sandblasted and painted with high temperature aluminum paint         0% G as inlet flange       Design pressure (psig)       460         M8" Overrall Dimensions:       4 ftØ x 17 ft tall       Design Temperature (F)       650         Approximate Overall Dimensions:       4 ftØ x 17 ft tall       Image: Construction of the sandblasted and painted with high temperature aluminum paint         Gass Conditions at Inlet:       Particulate Conditions at Inlet:       Image: Construction of the sandblasted and painted with high temperature aluminum paint         Goes Conditions at Inlet:       Particulate Conditions at Inlet:       Image: Construction of the sandblasted and painted with high temperature aluminum paint         Yelocity (ft/sec)       7,289       Specific Gravity       1,000         Osit Loading (Grains/acf)       16       Image: Construction fficiencies: Stokes Equiv. % Efficiency       Image: Construction fficiencies: Stokes Equiv. % Efficiency         Velocity (ft/sec)       72.29       Fraction Efficiencies: Stokes Equiv. % Efficiency       Image: Construction fficiencies: Stokes Equiv. % Efficiency       Image: Construction fficiencies: Stokes Equiv. % Efficiency         Velocity (ft/sec)       72.29       Fraction Efficiencies: Stokes Equiv. % Efficiency       Image: Construction fficiencies: Stokes Equiv. % Ef							. to be Reed				
Dust receiver section with flanged discharge       Exterior to be sandblasted and painted with high temperature aluminum paint         10° // gas inlet flange       Design pressure (psig)       460         Approximate Overall Dimensions:       4 ft // x 17 ft tall       Design Temperature (F)       650         Approximate Overall Dimensions:       4 ft // x 17 ft tall       Image: Comparison of the second				n ASIVIE Vess							refractory
40° @ gas inlet flange         Design pressure (psig)         460           48° @ verticle gas outlet flange         Design Temperature (F)         650           Approximate Overall Dimensions:         4 ft@ x 17 ft tall         Design Temperature (F)         650           Approximate Overall Dimensions:         4 ft@ x 17 ft tall         Design Temperature (F)         650         Design Temperature (F)           Gas Conditions at Inlet:         Particulate Conditions at Inlet:         Design Temperature (F)         650         Design Temperature (F)           Gas Conditions at Inlet:         Particulate Conditions at Inlet:         Design Temperature (F)         660         Design Temperature (F)           Goume per cylone (acfm)         7,289         Specific Gravity         1.000         Design Temperature (F)         66           Opensity (Ibm/ft-sec)         2.87E-05         Image: Temperature (F)         66         Image: Temperature (F)         66           Inlet Velocity (ft/sec)         72.29         Fraction Efficiencies: Stokes Equiv. % Efficiency         Image: Temperature (F)         Imag											n paint
H8"∅ verticle gas outlet flange       4 ft∅ x 17 ft tall       0esign Temperature (F)       650       0         Approximate Overall Dimensions:       4 ft∅ x 17 ft tall       0       0       0         Gas Conditions at Inlet:       Particulate Conditions at Inlet:       0       0       0         Joint per cylone (acfm)       7.289       Specific Gravity       1.000       0       0         Joint (Ibm/ft-sec)       2.87E-05       0       0       0       0       0         Viscosity (Ibm/ft-sec)       2.87E-05       0       0       0       0       0       0       0       0         Viscosity (Ibm/ft-sec)       7.299       Fraction Efficiencies: Stokes Equiv. % Efficiency       0	40" $\varnothing$ gas inlet flar	ige									
Gas Conditions at Inlet:       Particulate Conditions at Inlet:	18"Ø verticle gas	outlet flange						650			
/olume per cytone (acfm)         7,289         Specific Gravity         1.000           Density (Ibm/ft3)         0.5679         Dust Loading (Grains/acf)         16         1           /iscosity (Ibm/ft3)         0.287E-05         16         1         1           Intet Velocity (ft/sec)         72.29         Fraction Efficiencies: Stokes Equiv. % Efficiency         1         1           No load pres. drop (in.W.C.)         120.63         Dia.(microns)         Weight %         1         1           Full load pres. Drop (in. W.C.)         99.82         2.5         8.46         1         1           Image: Strap (in.W.C.)         99.82         3.5         19.29         1         1           Image: Strap (in.W.C.)         97.7	Approximate Overa	all Dimensions:		4 ft∅ x 17 ft	tall						
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# Appendix D

## D.1 INTRODUCTION

The first task undertaken by the team was to examine commercial technologies that are suitable for synthesis gas cleanup for biomass gasification. Currently, there are various types of technologies available dependent upon the specific cleanup requirements. For example, the clean-up required for syngas that will ultimately be fed to a reciprocating engine is much less than for syngas used in chemical synthesis. This study examined all technologies that could be required for syngas that will be used for Fischer-Tropsch (FT) liquids and alcohol synthesis.

The gas cleanup configuration for a system is generally determined by the composition of the syngas exiting the gasifier, the cleanup requirements for the intended use of the syngas, and economic considerations. Technologies such as cyclone separators, barrier filters, and electrostatic precipitators are routinely used for solid particulate removal. Catalytic tar crackers are employed to destroy tars and nitrogen contaminants. Wet scrubbers are used to remove a number of contaminants such particulates, alkali species, halides, soluble gases, and condensable liquids. Acid gas removal technologies encompass a large selection of processes including amine-based, physical solvent, liquid phase oxidation, and catalytic absorbent. Each section focuses on the operating size ranges and conditions, materials of construction, and cleanup parameters for each technology considered.

## D.2 PARTICULATE REMOVAL TECHNOLOGIES

## D.2.1 INTRODUCTION

During the gasification process, the mineral matter contained in the biomass feedstock will form inorganic ash, and the unconverted biomass will form char. These particulates are entrained in the syngas stream as it exits the gasifier. The concentration of particulates produced is often influenced by the gasifier design. These particulates can present emissions problem and can cause abrasion to downstream equipment. Therefore, the particulates concentration must be reduced using various technologies discussed in the following paragraphs.

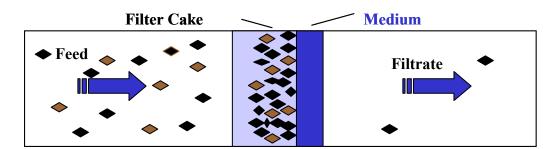
## **Cyclone Separators**

Cyclones use centrifugal forces to separate the bulk of large size particulates from a gas stream. In gasification systems, cyclones are normally used as the first step in the gas cleanup process. They are relatively inexpensive to manufacture and easy to operate which translate to low capital and maintenance costs. In general, 90-98% of particulates 10 µm or larger in diameter can be removed, but the removal efficiency decreases significantly for smaller particulates<sup>13</sup>. The removal efficiency also decreases as the operating temperatures increases. Cyclones are capable of handling operating temperatures up to 2000°F and can be designed to operate at pressures normally encountered in gasifiers. Cyclones are usually made from carbon steel and are refractory lined to withstand high temperature environments. A flow range from 300 to 13,000 CFM is typical for cyclones. This flow range is within the parameter of the syngas flow rate specified by NREL for this project.

<sup>&</sup>lt;sup>13</sup> Donaldson Co., Inc. "Cyclone Dust Collectors," July 2003, <<u>http://www.donaldson.com/en/industrialair/literature/000984.pdf</u>

## D.2.2 BARRIER FILTERS

Barrier filters remove particulates by capturing the particulates on the filter surfaces as the gas stream passes through the filter medium. The particulates accumulated on the filter surfaces form a cake, which can be dislodged by initiating a blowback flow. The blowback gas flows in the reverse direction of normal process flow and dislodges the filter cake, which is then removed from the system. The operating principle of barrier filters is illustrated in Figure D-1. Barrier filters include high-temperature filters, such as ceramic and metal candle filters, and low-temperature filters, such as baghouse filters.



#### FIGURE D-1 PRINCIPLE OF BARRIER FILTERS

### **Ceramic Candle Filters**

Ceramic filters are designed to remove particulate matter from gas streams at elevated temperatures. Ceramic filters can be designed for any flow requirement and can remove 90% of particulates larger than  $0.3 \ \mu m^{14}$ . In theory, the ceramic filter elements, normally made of aluminosilicate or silicon carbide powder with a sodium aluminosilicate binder, have exceptional physical and thermal properties, and should be able to withstand high temperature operations of up to 1800°F. However, commercial operations using ceramic filters at this temperature range have not been successful due to the susceptibility of the filter elements to cracking. Advances in composite filter element materials that have resistance to crack propagation at high temperatures are being developed and tested<sup>15</sup>. At temperatures below 850°F, ceramic filters have demonstrated satisfactory operational reliability.

In operations where tars are formed in the gasifier, ceramic filters should be operated at temperatures above the dew point of the tars (usually about 700-750°F) to avoid tar condensation. Condensed tar accumulates on filter surfaces and leads to plugging which will reduce the lifetime of the filter and impact process flowrates.

#### Metal Candle Filters

Metal filters are used in high temperature cleanup systems to remove particulate matter and can achieve filtration level as low as 1  $\mu$ m. They can be designed to meet any flow requirement and can operate over a wide range of temperatures depending on the material of construction. Metal

<sup>&</sup>lt;sup>14</sup> Pall Corp., "Syngas Filter Proposal," 26 January 2005, office communication

<sup>&</sup>lt;sup>15</sup> Jay E. Lane, Jean-Francois LeCostaouec, "Ceramic Composite Hot Gas Filter Development," <<u>http://www.netl.doe.gov/publications/proceedings/98/98ps/pspb-5.pdf</u>

filters made from stainless steel can be used in cleanup systems for temperatures below  $650^{\circ}$ F while Inconel or alloy HR filters are suitable for operating temperatures up to  $1100^{\circ}$ F. At even higher temperatures, Fercalloy can withstand temperatures up to  $1800^{\circ}$ F<sup>16</sup>, although commercial operation at this temperature has not been demonstrated. Commercial operation of metal filters operating at a maximum temperature of  $915^{\circ}$ F has been successful at a few gasification facilities in Europe<sup>17</sup>.

Some operational considerations for metal filters are the corrosion rate and tar deposition on filter elements. Under similar stream compositions and conditions, the corrosion rate of metal filter elements is ten times that of the surrounding piping; thus, a regular maintenance schedule is essential to ensure operational reliability. Additionally, in operations where filter elements are subjected to frequent cleaning cycles due to tar deposition, the lifetime of the filter will be reduced. Therefore, it is recommended that the filter be operated at a temperature above the dew point of the tars in the syngas stream to avoid tar condensation and deposition.

#### **Baghouse Filters**

Baghouse filters are made of a woven fabric or felted (non-woven) material to remove particulate matter from an air or gas stream and can remove particulates down to 2.5  $\mu$ m<sup>18</sup>. For woven fabric filters, the removal efficiency increases as the thickness of filter cake increases; thus, the removal efficiency of these systems is constantly changing. Felted filter systems have a constant removal efficiency that does not depend on the thickness of the filter cake<sup>19</sup>. Baghouse filters are modular in design and thus can accommodate a wide flow range from 1,500 to 150,000 CFM. The air-to-cloth ratio, or ratio of the volumetric flow to cloth area, sets the size of a baghouse unit. The bag fabric can be made from various materials including polyester, acrylic, NOMEX, Teflon, Ryton, and fiberglass<sup>20</sup>. The operating temperature range of an application influences the selection of bag material. For example, materials such as polyester or acrylic are suitable for applications with operating temperatures below 300°F. Due to the temperature limits of the filter fabric, baghouse filters are only used in the low-temperature cleanup systems. They are often used downstream of the cyclones so that the particulate loading on the filters can be reduced.

Disadvantages of baghouse filters include the need for periodic bag replacement that can result in high maintenance costs and the potential for bag fire or explosion. A spark detection and extinguishment system, along with bag grounding strips, are recommended safety measures to mitigate the fire potential. Additionally, the performance of the filter fabrics degrades drastically with tar deposition on the fabric surface, so fabric surface treatments such as Teflon coating and pre-coating with limestone or other compatible filter aids is recommended. Such pre-coats can

<sup>&</sup>lt;sup>16</sup> Mott Corp., "Fiber Metal. The High-Flow, Low-Pressure Drop Alternative," June 2003, http://www.mottcorp.com/resource/pdf/PSFIBERfinal.pdf

<sup>&</sup>lt;sup>17</sup> Mike Wilson, Mott Corp., "Fercalloy Metal Filters," 2 February 2005, Vendor input

<sup>&</sup>lt;sup>18</sup> Donaldson Co., Inc. "Dalamatic Dust Collectors," December 2002, <u>http://www.donaldson.com/en/industrialair/literature/000983.pdf</u>

<sup>&</sup>lt;sup>19</sup> EPA, "Air Pollution Technology Fact Sheet-Fabric Filter – Pulse-Jet Cleaned Type," http://www.macrotek.net/pdf/FS\_Pulse\_Clean\_Dust\_Collector.pdf

<sup>&</sup>lt;sup>20</sup> Ducon, "Baghouse Filter," 2003, <u>http://www.ducon.com/bag-house-filter.php</u>

also be used to adsorb mercury and other contaminants.. Industry experience suggests that either ceramic or metal filters should be used in place of baghouse filters in high temperature operations.

## D.2.3 ELECTROSTATIC PRECIPITATORS (ESPs)

ESPs are commonly used in large power plants to control fly ash emissions. ESPs consist of discharge electrodes centered between positively grounded collection plates. As the gas stream laden with particulates passes through the ESP, the discharge electrodes provide a negative charge to the particulates. The positively grounded collection plates act as a magnet for the negatively charged particulates, which collect on the plates. The collected particulates are transported into the collection hopper by the rapper or vibrator system.

ESPs are classified as either wet or dry processes. In wet ESPs, a water quench is applied either intermittently or continuously to the collection plates. The purpose of the water quench is to prevent possible fires that have occasionally resulted from the use of dry ESPs. The wastewater from wet ESPs must be treated prior to disposal.

For dry ESPs, the removal efficiency decreases for particulates with a high electrical resistivity since these particulates can introduce positive ions into the gas space resulting in reduced attraction of the negatively charged particulates to the collection plates. Particulates with a high resistivity are commonly produced from combustion of low-sulfur coals. Flow ranges of 10,000 – 300,000 CFM are typical for dry ESPs. Dry ESPs operate in the pressure range from vacuum conditions up to 150 psi and can operate at temperatures up to 750°F<sup>21</sup>.

Wet ESPs can achieve 99.9% removal of sub-micron particulates down to 0.01  $\mu$ m. Particulate resistivity does not affect removal efficiency of wet ESPs since the humid operating environment often reduces the resistivity of particulates. These systems are generally designed for gas flow range from 1,000 to 100,000 CFM. Gas streams with particulate sizes larger than 2  $\mu$ m or with an exceptionally high particulate loading should be pretreated to reduce the load on the ESP. Wet ESPs operate in the pressure range from vacuum conditions up to 150 psi, with operating temperatures limited to 170-190°F<sup>22,23</sup>.

The type of ESP selected for an application is largely influenced by the operating parameter and the type of particulates to be removed. However, the use of ESPs is limited in gasification systems due to the significant capital costs compared to other systems. Additionally, the removal efficiency of ESPs is sensitive to fluctuations in process conditions, such as changes in temperatures and pressures, gas compositions, and particulate loading. Therefore, ESPs are not suitable for biomass gasification applications that have highly variable syngas compositions from different feedstocks.

<sup>&</sup>lt;sup>21</sup> Gerry Graham, "Controlling Stack Emissions in the Wood Products Industry," http://www.ppcesp.com/ppcart.html

<sup>&</sup>lt;sup>22</sup> Ducon, "Wet & Dry Electrostatic Precipitators," 2003, <u>http://www.ducon.com/wet-dry-precipitators.php</u> (24 January 2005)

<sup>&</sup>lt;sup>23</sup> EPA, "Air Pollution Technology Fact Sheet-Wet Electrostatic Precipitator (ESP)-Wire-Pipe Type," <u>http://www.p2pays.org/ref/10/09890.pdf</u> (25 January 2005)

## D.3 TAR REMOVAL TECHNOLOGIES

### D.3.1 INTRODUCTION

Following NREL guidelines for the purpose of this project, tar is defined as C10+ hydrocarbons. Tar in syngas products can cause serious operational problems when the syngas stream cools below the dew point of the tars (usually about 700-750°F) and tar deposition occurs on downstream equipment and piping. Thus, tar removal is critical when there is tar present in the syngas. Tar can be removed either by physical or chemical processes. The most common physical process involves cooling the syngas stream to condense the tar into fine droplets and removing these droplets by wet scrubbing. Chemical process involves catalytic steam reforming of tars to lighter gases.

## D.3.2 WET SCRUBBERS

Wet scrubbing is generally used to remove water-soluble contaminants from the syngas by absorption into a solvent. Tar components are water-soluble can be removed by this method. Additionally, wet scrubbing is also often used to remove a number of other contaminants such as particulates, alkali species, halides, soluble gases, and condensable liquids. In wet scrubbing, water is a common solvent choice. Wet scrubbers with the venturi design are frequently used in gas cleanup applications to achieve sub-micron particulate removal requirements. As the gas stream enters the venturi scrubber, the scrubbing liquid is sprayed into the gas stream. The two streams are thoroughly mixed by the turbulence in the venturi throat section where fine particles are impacted and agglomerate into liquid droplets. The liquid droplets are separated from the gas stream in a separator unit consisting of a cyclone separator or a mist eliminator.

Venturi scrubbers can achieve 99.9% removal efficiency of sub-micron particulates. Flow range for a single-throat venturi is 500-100,000 SCFM. Flows above this range require either multiple venturi scrubbers in series or a multiple-throat venturi<sup>24</sup>. Venturi scrubbers with a quench section can accommodate high temperature gas streams up to 450°F, and they can operate over a wide range of pressures<sup>25</sup>.

The standard material of construction for venturi scrubbers is carbon steel. For corrosive or high temperature applications, stainless steel or special alloys such as FRP (fiberglass reinforced plastic) and Inconel are used.

The disadvantages of scrubbers include high pressure drop, the need to treat the wastewater effluent prior to disposal, and the loss of sensible heat of the syngas due to quenching. In power generation applications, the loss of sensible heat reduces the energy content of the gas and thus is undesirable; however, it is less of a concern in biomass refinery applications. Nevertheless, sensible heat loss will result in reduced overall system efficiency.

<sup>&</sup>lt;sup>24</sup> EPA, "Air Pollution Technology Fact Sheet-Venturi Scrubber" <<u>http://www.macrotek.net/pdf/FS\_Venturi\_Scrubber.pdf</u>

<sup>&</sup>lt;sup>25</sup> Envitech, Inc., "Venturi Scrubber," <<u>http://www.envitechinc.com/scrubber.zhtml</u>

## D.3.3 CATALYTIC TAR REFORMING

Catalytic reforming of biomass tars is a developing technology for tar removal from syngas streams. The concept of this technology is to reform tar in a fluidized reactor bed, or tar cracker, into lighter gases using a proprietary catalyst. In addition to tar, light hydrocarbons (C1 to C5), benzene, and ammonia are also removed. A few large-scale biomass gasification facilities, such as Carbona in Denmark and the FERCO gasifier in Vermont, have demonstrated a novel catalyst in their tar crackers since commercial catalysts are too friable for this application<sup>26</sup>. The FERCO tar cracker removed 90% of the tar in the syngas stream using a novel catalyst known as DN34<sup>27</sup>. In both of these processes, a wet scrubber was used downstream of the tar cracker to remove residual tars and impurities.

A tar cracker known as the Reverse Flow Tar Cracking (RFTC) reactor developed by BTG uses the steam reforming process with a commercial nickel catalyst<sup>28</sup>. The nickel catalyst is very sensitive to sulfur impurities; therefore, a syngas stream containing sulfur contaminants has to be desulfurized prior to entering the RFTC reactor. Due to the cooling requirement for the desulfurization process, the syngas is fed to the reactor at a temperature from 660 -1200°F and is heated to the reaction temperature of 1650 -1740°F in the reactor entrance section. The heated gas passes through a bed of nickel catalyst where tar, light hydrocarbons, and ammonia are removed by steam reforming. The main reactions of the RFTC reactor are:

$C_nH_m + nH_2O \iff nCO + (\frac{1}{2}m+n)H_2$	Hydrocarbon reforming
$2NH_3 \Leftrightarrow N_2 + 3H_2$	Reverse ammonia synthesis
$CO + H_2O \iff CO_2 + H_2$	Water-gas shift

A small amount of the syngas is combusted to counterbalance the endothermic tar reforming reactions:

 $H_{2} + \frac{1}{2} O_{2} \rightarrow H_{2}O$   $CO + \frac{1}{2} O_{2} \rightarrow CO_{2}$   $CH_{4} + 2O_{2} \rightarrow CO_{2} + 2H_{2}O$ 

The typical conversion for the RFTC reactor is as follows:

<b>Components</b>	<b>Conversion</b>
Benzene	82
Napthalene	99
Phenol	96
Total Aromatic	94
Total Phenols	98
Total Tar	96
Ammonia	99

<sup>26</sup> Don J. Stevens, "Hot Gas Conditioning: Recent Progress with Larger-Scale Biomass Gasification Systems," prepared by Pacific Northwest National Laboratory for NREL, August, 2001

Mark A. Paisley, Mike J. Welch, "Biomass Gasification Combined Cycle Opportunities Using the Future Energy *SilvaGas* Gasifier Coupled to Alstrom's Industrial Gas Turbines," ASME Turbo Expo Land, Sea, and Air, Georgia World Congress Center, June 16-19, 2003
 BTG Biomass Technology Group, "Tar & Tar Bernoval," 22 March 2004, http://www.bioworld.com/technologies/tar-removal.html

BTG Biomass Technology Group, "Tar & Tar Removal," 22 March 2004, http://www.btgworld.com/technologies/tar-removal.html

The partial oxidation reaction (POx) was also investigated as a possible process for tar and hydrocarbons removal. In this process, the syngas enters the POx reactor and mixes with oxygen that is at about 300°F. Partial oxidation and reforming reactions occur in a combustion zone where tar, methane, light hydrocarbons, and benzene are converted to CO and  $H_2$ . The reformed gas exits the reactor at about 2500°F.

The main disadvantage of POx is a reduction of the product gas heating value. In order to achieve destruction of the tars and oils, a high temperature reactor is required. While it is possible to crack the tars and oils at moderate temperatures, it is very difficult to selectively react methane. However at high temperatures oxidation of CO and  $H_2$  also occur. As a result, the gas composition will be shifted toward a lower  $H_2$ :CO ratio.

In order to improve the efficiency of POx, a catalyst can be used to lower the temperature, and hence also the amount of oxidizer required to destroy the tars and oils. A catalytic auto-reformer technology may provide a solution to biomass tar and oil elimination. Such an application would only apply to a particulate-free gas since any particulate in the gas could shortly blind the catalytic reactor. As shown in Table D-1 below, an auto-thermal reformer is essentially a hybrid between POx and steam reforming.

Gas Reforming Process	Typical H₂/CO ratio	Comments
Tar Cracking/Reforming	wide range	Developing technology. Operating information not widely available.
Steam (Methane) Reforming SR or SMR	3-4	Dominant technology for industrial H <sub>2</sub> production Typically high efficiency
Partial Oxidation (POx)	1.7-1.8	Used in refining to upgrade heavy liquid fuels Low efficiency May generate coke or soot
Auto-thermal Reforming (ATR)	2.4-4	Hybrid of POx and SR

## TABLE D-1 COMPARISON OF SYNGAS REFORMING PROCESS TECHNOLOGY

## D.4 ACID GAS REMOVAL TECHNOLOGIES

## D.4.1 INTRODUCTION

Sulfur contaminants such as H<sub>2</sub>S, COS, CO<sub>2</sub>, mercaptans, and HCN poison catalysts used in liquid fuel synthesis. Therefore, the syntheses of methanol and FT liquids from syngas require that the sulfur be removed from the syngas to a residual level of 0.10 ppm or less. The syngas considered for this study contains approximately 400 ppmv of H<sub>2</sub>S; therefore, acid gas removal is critical in the gas cleanup process. Acid gas removal technologies can be categorized as amine-based, physical solvent, liquid phase oxidation, or catalytic absorbent processes. The type of technology selected is largely influenced by the system operating conditions, the sulfur level in the syngas stream, and the desired purity of the treated syngas. Brief descriptions to explain the overall process for each system are given in the following paragraphs.

## D.4.2 AMINE-BASED SYSTEM

Amine processes are proven technologies for the removal of  $H_2S$  and  $CO_2$  from gas streams by absorption. Amine systems generally consist of an absorber, a stripper column, a flash separator, and heat exchangers. This is a low-temperature process in which the gas to be treated usually enters the absorber at approximately 110°F. In the absorber, acid gases are removed from the gas stream by chemical reactions with the amine solution. The sweet gas stream exits at the top of the absorber. Regeneration of the rich amine is accomplished through the flash separator to remove absorbed hydrocarbons followed by a stripper column to remove the  $H_2S$  and  $CO_2$  from the amine solution. The lean amine solution is cooled and returned to the absorber. The stripped acid gas stream is cooled to recover water and then sent to a sulfur recovery unit. A typical amine system is shown in Figure D-2.

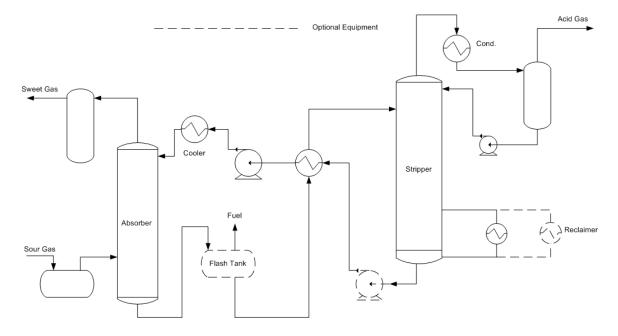


FIGURE D-2 TYPICAL AMINE SYSTEM FLOW DIAGRAM

Amine systems normally operate in the low to medium pressure range of 70-360 psi, although higher pressures can be accommodated with a specific amine solvent. However, in applications where the partial pressure of acid gases is high, the economy of an amine system declines in comparison to other systems. Amine systems can be designed to meet specific flow range and sulfur removal requirements. A sulfur removal level as low as 1 ppm can be achieved but at the expense of operating cost due to the large solvent circulation rate required<sup>29</sup>.

There are a variety of amine solutions available. Each offers distinct advantages based on the specific treating condition. Commercially available amine solutions include<sup>30</sup>:

<sup>30</sup> GPSA

<sup>&</sup>lt;sup>29</sup> Input from GTI, "Gas Cleanup Technologies Discussion," 3 February 2005, office communication

 $MEA - Monoethanolamine removes both H_2S and CO_2$  from gas streams and is generally used in low-pressure systems and in operations requiring stringent sulfur removal.

DGA - Diglycolamine is used when there is a need for COS and mercaptan removal in addition to H<sub>2</sub>S. DGA can hydrolyze COS to H<sub>2</sub>S; thus, a COS hydrolysis unit is not needed in the cleanup system.

DEA - Diethanolamine is used in medium- to high-pressure systems (above 500 psi) and is suitable for gas stream with a high ratio of  $H_2S$  to  $CO_2$ .

MDEA - Methyldiethanolamine has a higher affinity for H<sub>2</sub>S than CO<sub>2</sub>. MDEA is used when there is a low ratio of H<sub>2</sub>S to CO<sub>2</sub> in the gas stream so that the H<sub>2</sub>S can be concentrated in the acid gas effluent. If a Claus plant is used for sulfur recovery, a relatively high concentration of H<sub>2</sub>S (>15%) in the acid gas effluent is required for optimal Claus operation.

After prolonged use, MEA, DGA, and MDEA solutions accumulate impurities that reduce the  $H_2S$  removal efficiency of the solutions. A reclaim unit is needed to remove the impurities in order to improve system efficiency.

One major operating concern for amine systems is corrosion. In water,  $H_2S$  dissociates to form a weak acid while  $CO_2$  forms carbonic acid. These acids attack and corrode metal. Therefore, equipment in the amine systems may be clad with stainless steel to improve equipment life.

## D.4.3 PHYSICAL SOLVENT SYSTEM

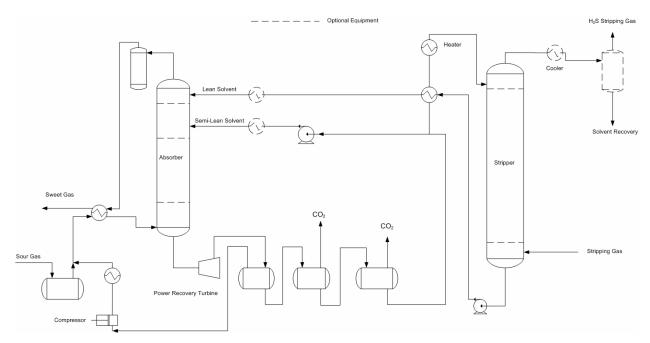
This acid gas removal technology uses an organic solvent to remove acid gases from gas streams by physical absorption without chemical reaction. The driving force of this process is the high solubility of acid gases in the organic solvent. In most cases, solubility increases as the temperature decreases and the pressure increases. Thus, physical absorption is a low-temperature, high-pressure process, with high partial pressure of acid gases required for the economy and efficiency of this process. The temperature of the solvent should be as low as possible while the temperature of the gas to be treated usually enters the absorber at about 100°F. Physical solvent systems normally operate at pressures above 150 psi<sup>31</sup>.

In general, physical solvent systems consist of an absorber, a stripper column, a series of flash separators, and heat exchangers. In the absorber, acid gases in the syngas stream are absorbed into the solvent solution. The sweet syngas stream exits the top of the absorber. Regeneration of the rich solvent stream is accomplished through a series of flash separators at reduced pressures to remove absorbed hydrocarbons followed by the stripper column to remove the acid gases from the solvent. The lean solvent solution is cooled and returned to the absorber. The stripped acid gas stream is cooled to recover water and then sent to a sulfur recovery unit. A typical physical solvent system is shown in Figure D-3.

<sup>&</sup>lt;sup>31</sup> Gerhard Ranke, "Advantages of the Rectisol-Wash Process in Selective H<sub>2</sub>S Removal from Gas Mixtures," 1973, office communication, 30 January 2005

The two common physical systems are Rectisol and Selexol. The Rectisol process, which uses methanol at temperatures  $< 32^{\circ}$ F, can achieve a sulfur removal level as low as 0.1 ppm. The Selexol process, which uses mixtures of dimethyl ethers of polyethylene glycol, can achieve a sulfur removal level of 1ppm<sup>32</sup>.

Selection of material of construction depends on the solvent used. For example, stainless steel is required for much of the Rectisol process equipment, contributing to a significant capital cost. In the Selexol process, carbon steel is the standard material of construction, except for those areas with high severity where stainless steel will be used.



### FIGURE D-3 TYPICAL PHYSICAL SOLVENT SYSTEM FLOW DIAGRAM

## D.4.4 LIQUID PHASE OXIDATION PROCESS -- LO-CAT™

LO-CAT<sup>TM</sup> is an oxidation process that uses iron catalyst held in a chelating agent to oxidize H<sub>2</sub>S to elemental sulfur. H<sub>2</sub>S is the only acid gas being removed in this process but a high CO<sub>2</sub> concentration in the feedgas requires caustic for pH adjustment. A LO-CAT<sup>TM</sup> process consists of 3 sections that include an absorber, an oxidizer for catalyst regeneration, and a sulfur handling unit. Figure D-4 illustrates a typical LO-CAT<sup>TM</sup> unit. When the gas stream comes in contact with the LO-CAT<sup>TM</sup> solution in the absorber, H<sub>2</sub>S in the gas stream is converted to elemental sulfur. The spent catalyst along with the elemental sulfur exit the absorber, then enter the oxidizer where the spent catalyst is regenerated by contact with oxygen in air, and the elemental sulfur is concentrated into a sulfur slurry. The sulfur slurry moves to the sulfur handling unit where it is washed to recover any entrained catalyst. The sulfur recovered from a LO-CAT<sup>TM</sup>

<sup>&</sup>lt;sup>32</sup> D.J. Kubek, E. Polla, F.P. Wilcher, "Purification and Recovery for Gasification," Gasification Technologies Conference, October 1996, San Francisco, CA.

process contains a small amount of entrained residual catalyst and is considered low-value sulfur that is suitable for agricultural purposes but is undesirable as a chemical feedstock.

The LO-CAT<sup>TM</sup> process is suitable for small-scale applications that require less than 20 TPD of sulfur recovery capacity, making the LO-CAT<sup>TM</sup> a candidate process for this study, which has less than 5 TPD of sulfur recovery. This process can achieve 99.9%+ of H<sub>2</sub>S removal efficiency<sup>33</sup>. This process can operate over a wide range of pressures from atmospheric up to 600 psi, but most are low-pressure applications in amine acid gas service. The operating temperature is normally maintained at about 110°F since high temperatures degrade the LO-CAT<sup>TM</sup> solution that can affect removal efficiency. Advantages of this process include the ability to treat a wide range of gas compositions, a significant turndown flexibility, and less capital costs in comparison to the Claus process with the associated tail gas treating unit.

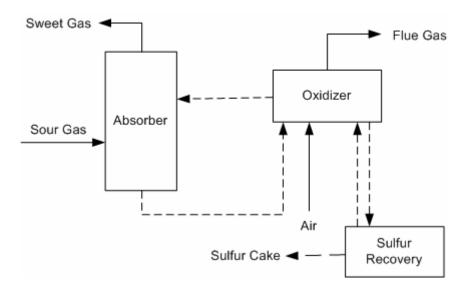


FIGURE D-4 TYPICAL LO-CAT™ SYSTEM FLOW DIAGRAM

Since  $\text{LO-CAT}^{\text{TM}}$  only removes  $\text{H}_2\text{S}$ , a COS hydrolysis unit upstream of the  $\text{LO-CAT}^{\text{TM}}$  is needed to hydrolyze any COS in the gas stream to  $\text{H}_2\text{S}$ . Other acid gases, such as HCN and mercaptans, would have to be removed by wet scrubbing.

The standard material used for LO-CAT<sup>TM</sup> systems is stainless steel. Under certain conditions where there is build-up of chloride ions from the feed gas, FRP (fiberglass reinforced plastic) material is used to provide added stability for the stainless steel components<sup>34</sup>.

## D.4.5 CATALYTIC ABSORBENT-ZnO

ZnO is often used as a polishing step for sulfur removal in gas streams where the sulfur level is below 20 ppmv. In a traditional purification system, illustrated in Figure D-5, ZnO is used in

<sup>&</sup>lt;sup>33</sup> Douglas L. Heguy, Gary J. Nagl, "The State of Iron Redox Sulfur Plant Technology New Developments to an Established Technology," <<u>http://www.gtpmerichem.com/support/technical\_papers/state\_of\_iron\_redox.html</u>> (25 January 2005)

<sup>&</sup>lt;sup>34</sup> GTP-Merichem, "FAQ's About Sulfur Removal and Recovery Using the LO-CAT System," <<u>http://www.gtp-merichem.com/support/faq.html</u>> (25 January 2005)

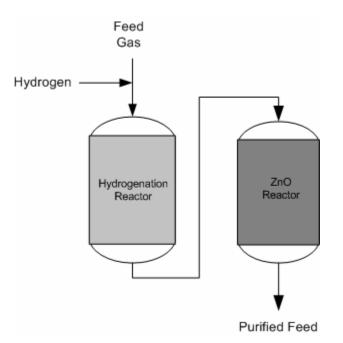
conjunction with hydrogenation catalysts based on cobalt, molybdenum and nickel. This system involves the hydrogenation of sulfur compounds such as mercaptans to  $H_2S$ , and halides such as chlorides to HCl. These compounds are then reacted with the ZnO absorbent where  $H_2S$  is converted to zinc sulfide, and HCl forms a stable chloride. Additionally, ZnO also removes COS by hydrolysis to form  $H_2S$  which is then adsorbed to form zinc sulfide. The general reactions are summarized below<sup>35</sup>:

#### Hydrogenation reactions:

 $RSH + H_2 \rightarrow RH + H_2S$  $RC1 + H_2 \rightarrow RH + HC1$ 

#### Reaction with ZnO:

 $\begin{array}{l} ZnO + H_2S \Leftrightarrow ZnS + H_2O \\ ZnO + COS \Leftrightarrow ZnS + H_2O \end{array}$ 



#### FIGURE D-5 TRADITIONAL ZNO PURIFICATION SYSTEM

A sulfur removal below 50 ppb is attainable with  $ZnO^{36}$ . Since the sulfur specifications for alcohols and FT liquids are 0.10 ppm or less, ZnO will be used to achieve these requirements. However, a hydrogenation reactor will not likely be required since the syngas stream given by NREL does not contain halogens or any other sulfur compounds other than H<sub>2</sub>S.

<sup>&</sup>lt;sup>35</sup> Johnson Matthey Group, "Purification Catalysts and Absorbents for Hydrogen Production," available at <u>http://www.jmcatalysts.com</u> (25 January 2005)

<sup>&</sup>lt;sup>36</sup> Johnson Matthey Group, "Absorbent for Sulphur Polishing," available at <u>http://www.jmcatalysts.com</u> (25 January 2005)

ZnO is active over a wide range of temperatures from ambient to 750°F; however, operating temperatures range between 660°F and 750°F are normally used to maximize absorption efficiency. Operating pressure limits are not a concern for the use of ZnO absorbent. The ZnO reactor is normally constructed from carbon steel clad with stainless steel to prevent corrosion caused by acid gases.

One drawback of this process is the significant operating costs contributed by frequent replacement and disposal of ZnO absorbent since it cannot be regenerated.

## D.4.6 COS HYDROLYSIS

COS can be removed simultaneously with  $H_2S$  and other acid gases in some of the acid gas removal processes described above. In chemical absorption processes, the degree of COS removal is dependent upon the reactivity of the solvent solution with COS. For example, DGA can remove virtually all of the COS whereas MDEA has little reactivity with COS. In physical absorption processes, the solubility of COS in the physical solvent and the COS partial pressure determine the level of removal. A COS level of 0.1 ppm is attainable with the Rectisol process while the Selexol process can achieve 10 ppm  $COS^{37}$ . In the ZnO process, approximately 80% of the COS can be removed by hydrolysis.

When COS cannot be effectively removed by the conventional acid gas removal processes, a COS hydrolysis reactor is required and is placed upstream of the acid gas removal unit. COS removal is accomplished by hydrolysis of COS on a catalyst to form H<sub>2</sub>S which is sent to the downstream acid gas removal unit. Activated alumina catalysts are often used in these applications. COS removal to 0.1 ppm or below can be achieved<sup>38</sup>. COS hydrolysis reactors can operate over a wide range of pressures with temperatures in the range of 100°F – 450°F. The COS hydrolysis reactor is normally constructed from carbon steel clad with stainless steel to prevent corrosion caused by acid gases.

## D.4.7 SULFUR RECOVERY UNIT (SRU)

In the sulfur recovery unit, the acid gas stream from the amine or physical solvent unit is recovered to elemental sulfur. In operations where the sulfur recovery is more than 20 TPD, a Claus SRU is generally an economical approach. However, since the amount of sulfur in the syngas for this study is small (< 5 TPD), a Claus operation would not be a cost-effective solution. For a low sulfur recovery capacity, a LO-CAT SRU would be a more suitable process.

## D.5 AMMONIA, ALKALI, AND OTHER CONTAMINANTS

## D.5.1 AMMONIA REMOVAL

Two methods for removing ammonia include catalytic tar reforming and wet scrubbing. Tar cracker catalysts have been demonstrated to be effective at reducing ammonia in the syngas stream by conversion to  $N_2$  and  $H_2$ . A tar cracker can be used to remove ammonia followed by

#### <sup>38</sup> United Catalysts Inc., "UCI COS Hydrolysis Catalysts," 22 June 1992, and office communication, 17 February 2005

<sup>&</sup>lt;sup>37</sup> Robert Chu, Senior Design Engineer, Nexant, "COS Removal," office communication, 17 February 2005

gas cooling and a wet scrubber to remove residual ammonia. This cleanup configuration should achieve complete removal of ammonia.

#### D.5.2 ALKALI REMOVAL

Alkali removal is normally accomplished by cooling the syngas stream below 1100°F to allow condensation of alkali species followed by barrier filtration or wet scrubbing. Corrosion potential should be taken into consideration when using metal or ceramic candle filters due to possible reactions between the alkali and filter materials at high temperatures. Several demonstration facilities had used barrier filters to removal alkali along with other impurities. For example, ceramic filters were used at the Lahti facility in Finland and Varnamo in Sweden<sup>39,40</sup>. The Varnamo facility experienced breakage of the ceramic filter elements and replaced them with sintered metal filters, which operated successfully. Baghouse filters were used in Lahti's low-pressure gasification system and the FERCO facility in Vermont.

Alkali can easily be removed by wet scrubbing, thus it is often the preferred method for alkali removal. Descriptions of operating and cleanup parameters for barrier filters and wet scrubbing are given earlier in this Appendix.

### D.5.3 REMOVAL OF OTHER CONTAMINANTS

Contaminants such as halides or metals (i.e. nickel or iron) are not typical, but may exist in syngas produced from biomass gasification. If present, these impurities can be removed by wet scrubbing or purification by hydrogenation and ZnO absorption.

<sup>&</sup>lt;sup>39</sup> OPET Finland, OPET Report 4 "Review of Finnish Biomass Gasification Technologies," May 2002

<sup>&</sup>lt;sup>40</sup> Krister Stahl, et al. "Biomass IGCC at Varnamo, Sweden-Past and Future," GCEP Energy Workshop, 27 April 2004, Stanford University, CA.

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